

# Introduction to Simulation - Lecture 24

## Model-Order Reduction

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Thanks to Luca Daniel, Guillaum Lassauxx, Jing Li,  
Mark Reichelt, Deepak Ramaswamy, Michal  
Rewienski, Mary Tolikas, Karen Veroy and Karen  
Willcox

# MOR Outline

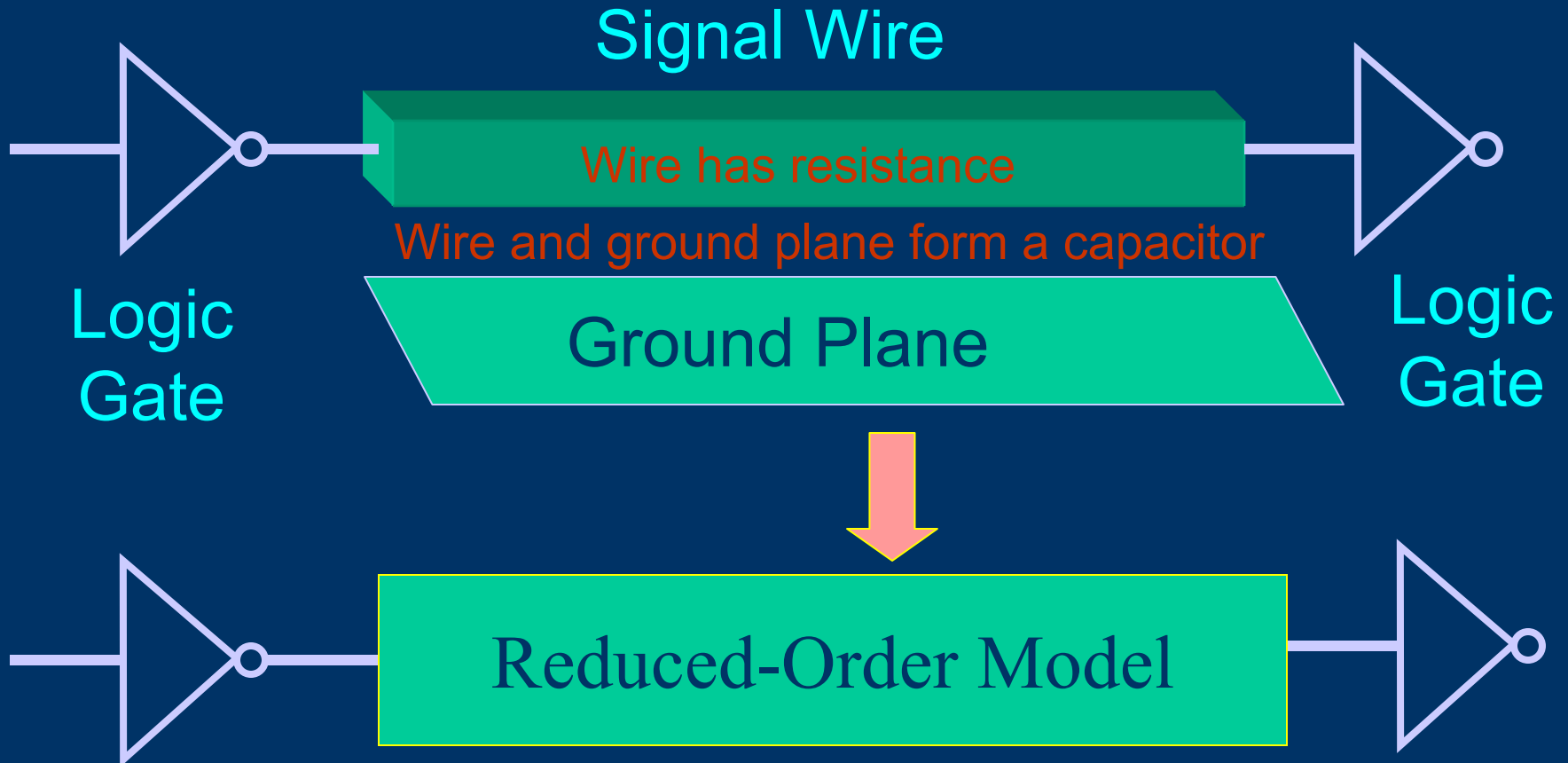
- Need For Model Reduction
  - Circuits, MEMS, Jet Engines
- Steady-State Case (linear and nonlinear)
- Dynamic Linear Case
  - Eigenmodes and Rational Functions
  - Projection Framework, Krylov, TBR
- Nonlinear Case
  - Projection Framework

# Today's Outline

- Need For Model Reduction
  - Circuits, MEMS, Optics, Jet Engines
- Simple Example Problem
  - Heat Conducting bar example
- Steady-State Case (linear and nonlinear)
- Dynamic Linear Case
  - Truncating Eigenmodes
  - Rational Function Fitting

# Application Example

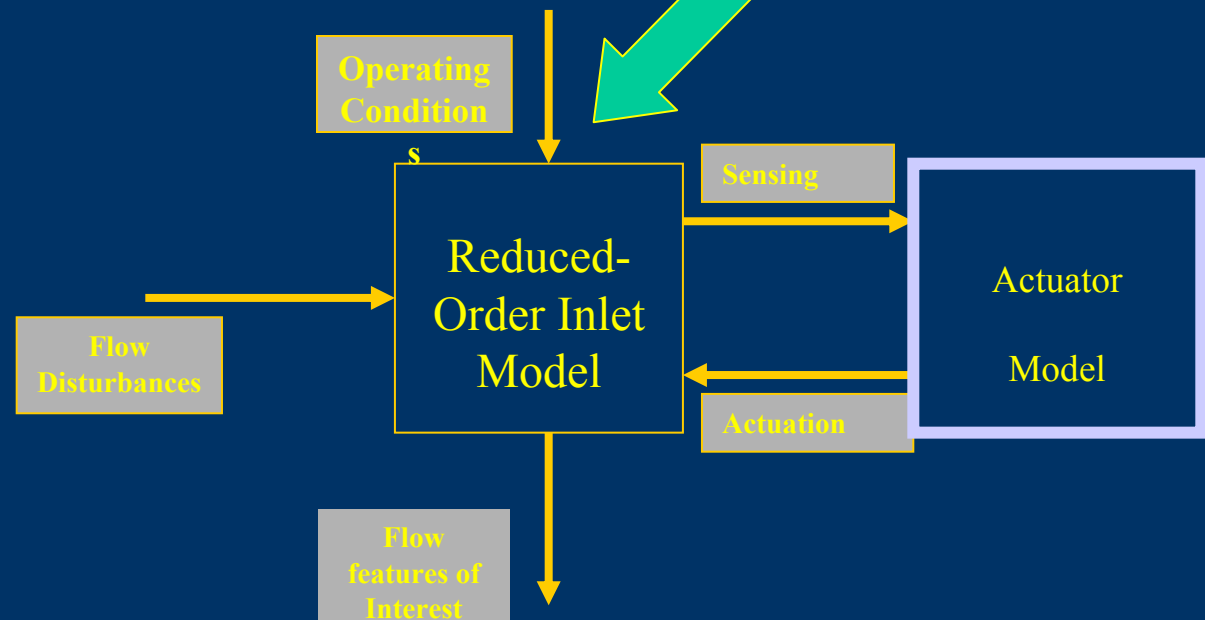
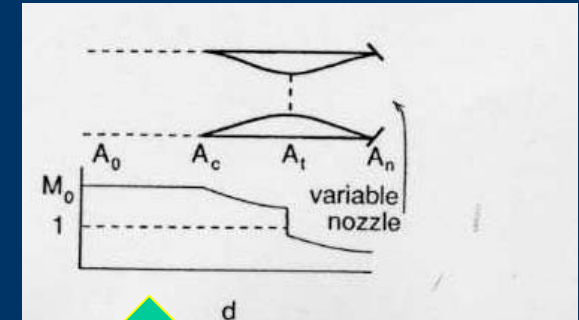
## Signal Wire in an Integrated Circuit



- Assess wiring impact on IC performance
- Wire Model must preserve terminal behavior

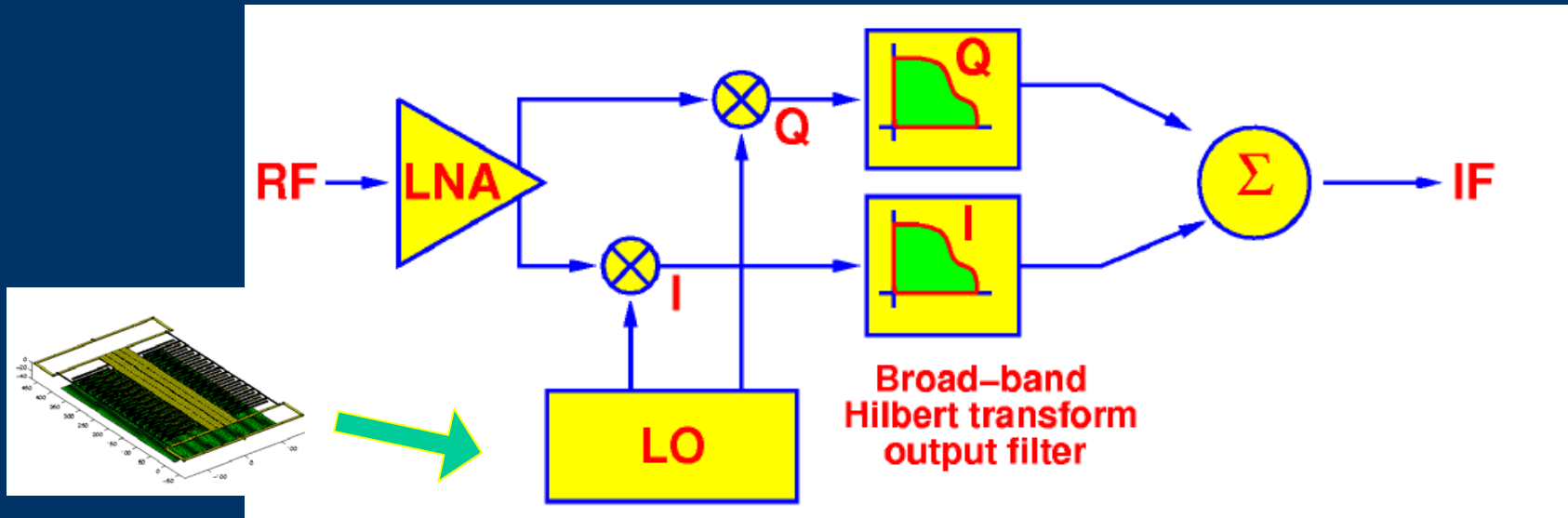
## Application Example

- Generate Low-order models directly from Navier-Stokes Equation based physical simulators.
- Reduced model must preserve instabilities.



# Application Examples

## Micromechanical Resonators in a Wireless transceiver

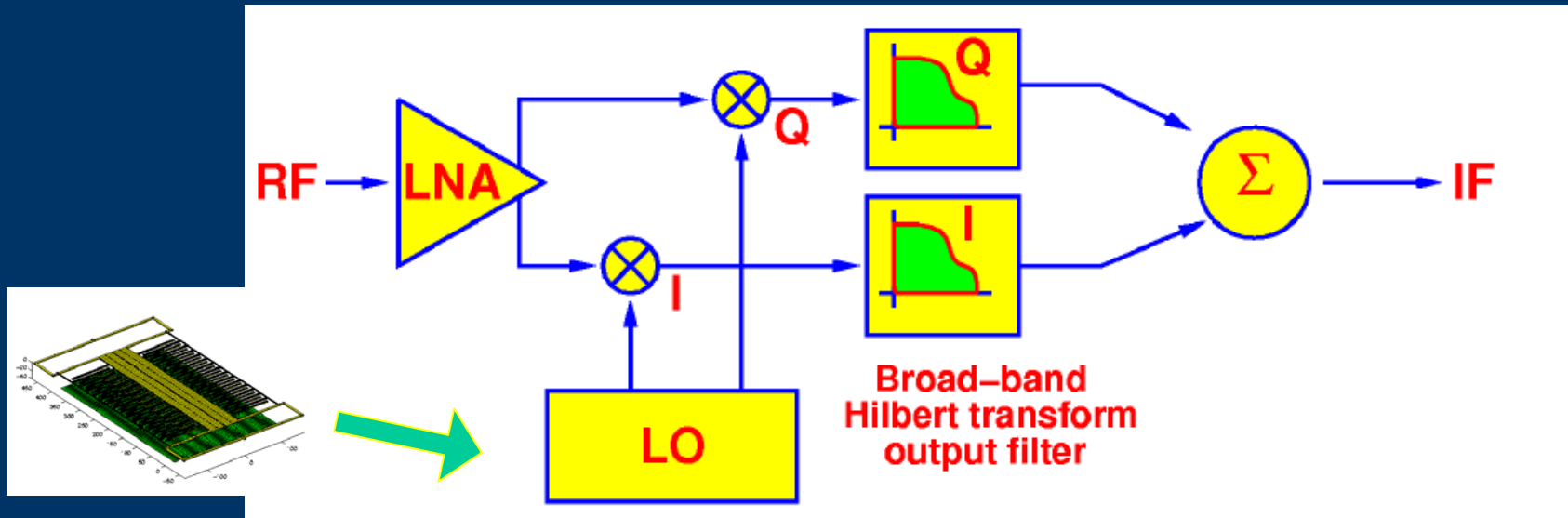


RF Front end with micromachined resonators for the oscillator

- What is system performance (noise, distortion, etc).
- Will poly-substrate separation (changes Q) matter?
- How tight must manufacturing tolerances be?

# Application Examples

## Micromechanical Resonators in a Wireless transceiver (cont)



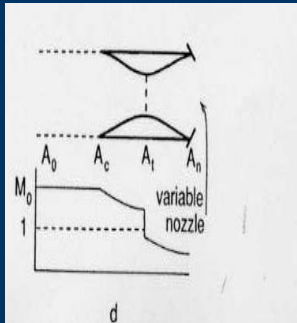
**Need to simulate ENTIRE system with dynamically accurate macromodels for all the components**

- Devices have Well-defined Inputs and Outputs
  - Signal Transmission on a Wire
    - Left and right end Voltages and Currents
  - Jet Engine Nozzle Design
    - Nozzle in and out flow, Nozzle size
  - Microresonator
    - Comb finger voltages and currents
- Dynamics is important
  - Internal state must be somehow represented

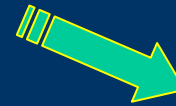


# Application Examples

## Traditional Approach to Generating Models

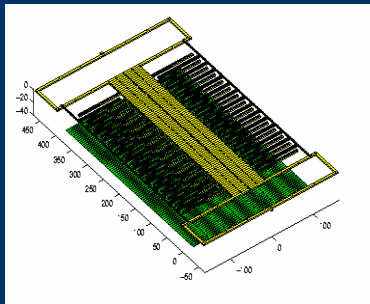


6 months...

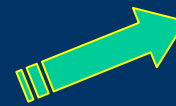


$$\frac{dx_r(t)}{dt} = F(x_r(t)) + b_r u(t)$$

$$y(t) = c_r^T x_r(t)$$



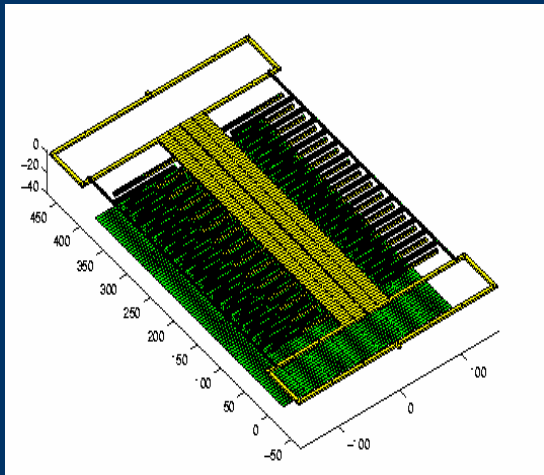
6 months...



Model for the  
System Simulator

# The Numerical Macromodeling or Model Reduction Paradigm

Generate a Reduced-Order Model Directly from  
3-D Geometry and Physics



**Automatic**



$$\frac{dx_r(t)}{dt} = F(x_r(t)) + b_r u(t)$$

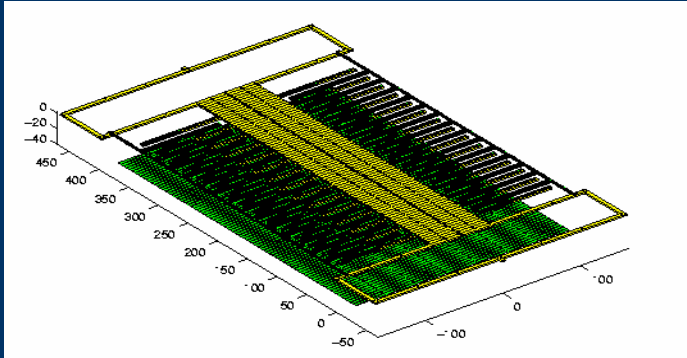
$$y(t) = c_r^T x_r(t)$$

Cheap to evaluate model  
which captures  
input (u)/output(y)  
behavior

Complicated Geometry,  
Coupled Electrostatics,  
Fluids, Elastics

# The Numerical Macromodeling or Model Reduction Paradigm

## What's Needed



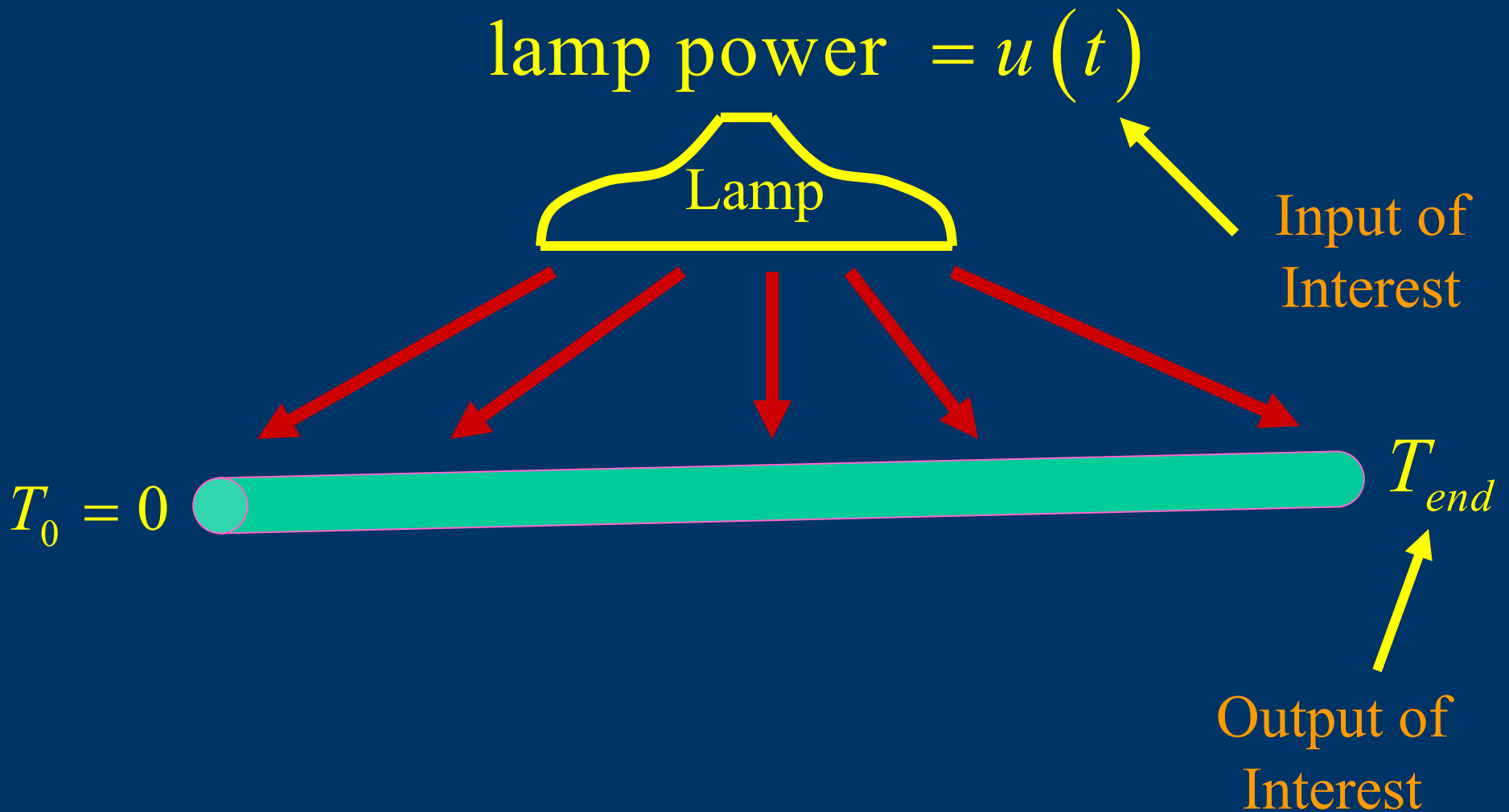
$$\frac{dx_r(t)}{dt} = F(x_r(t)) + b_r u(t)$$

$$y(t) = c_r^T x_r(t)$$

- Fast Solvers for complicated 3-D geometries
  - (Fast enough to solve ENTIRE devices)
  - for fluids, electrostatics, mechanics, ...
- Approaches for coupled domain analysis
  - Multilevel-Newton methods
- Automatic extraction of reduced order models

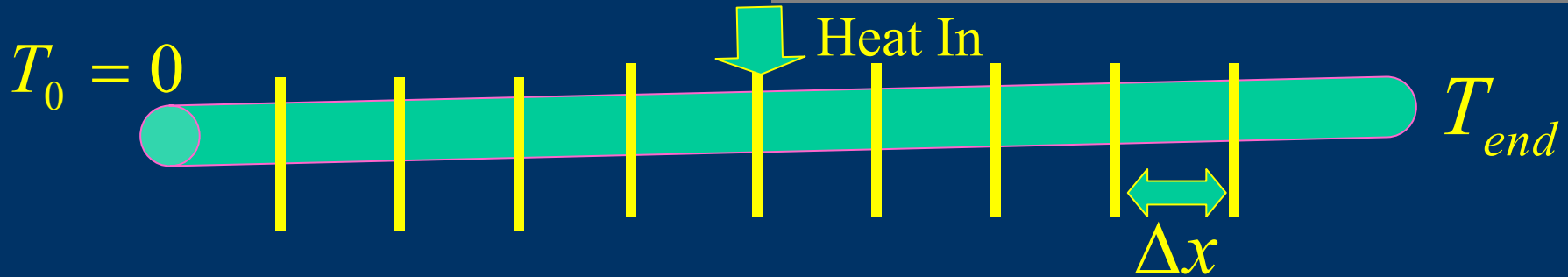
# Demonstration Example

## Heat Conducting Bar



# Demonstration Example

## Basic Equations



- Temperature Differential Equation

$$\underbrace{\gamma}_{\text{specific heat}} \frac{\partial T(x, t)}{\partial t} - \underbrace{\kappa}_{\text{thermal conductivity}} \frac{\partial^2 T(x, t)}{\partial x^2} = h(x) \underbrace{u(t)}_{\text{scalar input}}$$

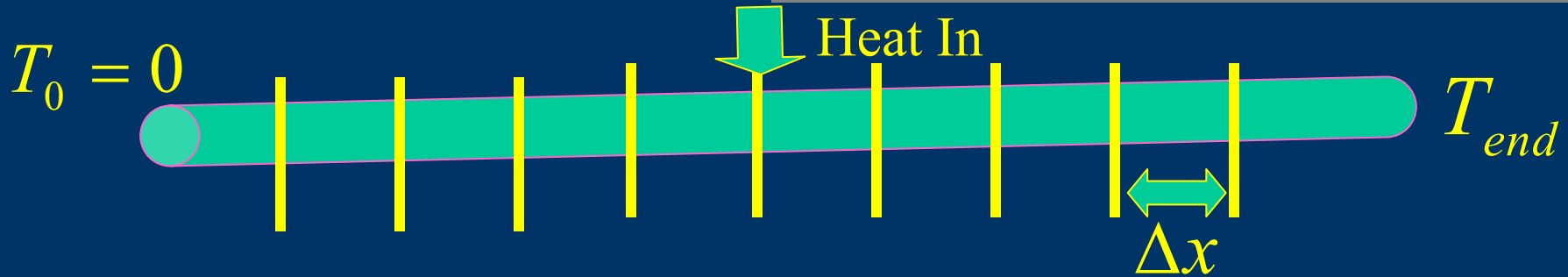
- Spatial Discretization (except at end)

$$\gamma \frac{d\hat{T}_i}{dt} - \frac{\kappa}{(\Delta x)^2} \left( \hat{T}_{i+1} - 2\hat{T}_i + \hat{T}_{i-1} \right) = h(x_i) u(t)$$

# Demonstration Example

## Heat Conducting Bar

### Input-Output Discrete Equations



$$\gamma \frac{d\hat{T}_i}{dt} - \frac{\kappa}{(\Delta x)^2} (\hat{T}_{i+1} - 2\hat{T}_i + \hat{T}_{i-1}) = h(x_i)u(t) \quad i \in [1, \dots, N-1]$$

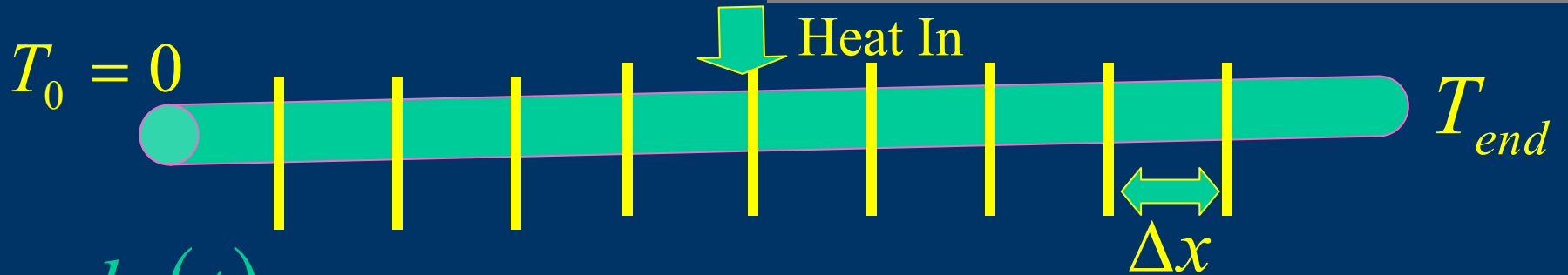
$$\gamma \frac{d\hat{T}_i}{dt} - \frac{\kappa}{(\Delta x)^2} (\hat{T}_N - \hat{T}_{N-1}) = h(x_N)u(t)$$

$$T_{end} = \hat{T}_N$$

# Demonstration Example

## Heat Conducting Bar

### State-Space Description



$$\frac{dx(t)}{dt} = \underbrace{A}_{N \times N} x(t) + \underbrace{b}_{N \times 1} \underbrace{u(t)}_{\text{scalar input}}$$

$$y(t) = \underbrace{c^T}_{N \times 1} x(t)$$

*scalar output*

Given the right scaling

$$A = \begin{bmatrix} 2 & -1 & 0 & \dots & 0 \\ -1 & 2 & \ddots & \ddots & \vdots \\ 0 & \ddots & \ddots & \ddots & 0 \\ \vdots & \ddots & \ddots & 2 & -1 \\ 0 & \dots & 0 & -1 & 1 \end{bmatrix}$$

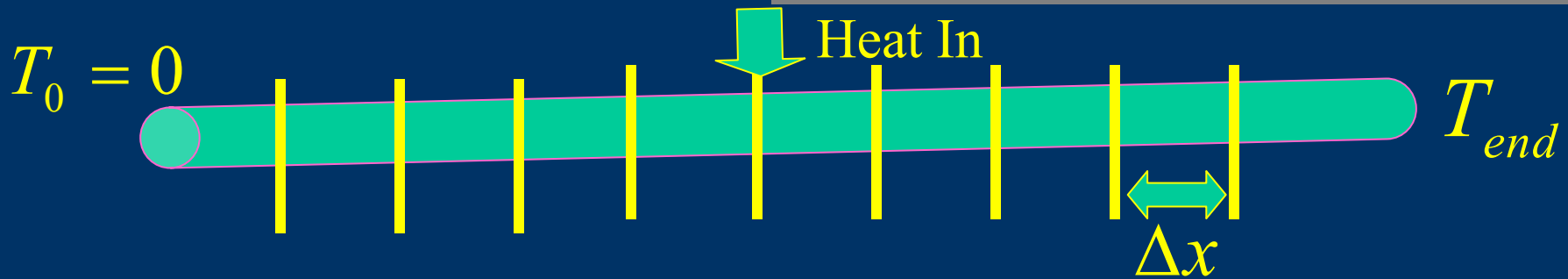
$$b = \begin{bmatrix} h(x_1) \\ h(x_2) \\ \vdots \\ \vdots \\ h(x_N) \end{bmatrix}$$

$$c = \begin{bmatrix} 0 \\ \vdots \\ \vdots \\ 0 \\ 1 \end{bmatrix}$$

# Demonstration Example

## Heat Conducting Bar

Temperature Dependent Thermal Conductivity



- Temperature Differential Equation

$$\underbrace{\gamma}_{\text{specific heat}} \frac{\partial T(x,t)}{\partial t} - \underbrace{\kappa(T(x,t))}_{\text{thermal conductivity}} \frac{\partial^2 T(x,t)}{\partial x^2} = h(x)$$

- Simple Spatial Discretization (not at ends)

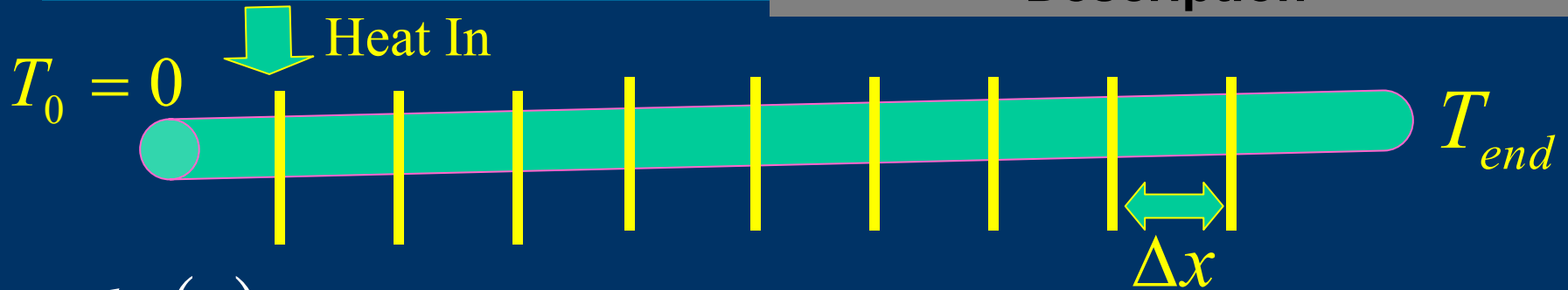
$$\gamma \frac{d\hat{T}_i}{dt} - \frac{\kappa(\hat{T}_i)}{(\Delta x)^2} \left( \hat{T}_{i+1} - 2\hat{T}_i + \hat{T}_{i-1} \right) = 0$$



# Demonstration Example

## Heat Conducting Bar

### Nonlinear State-Space Description



$$\frac{dx(t)}{dt} = F(x(t)) + \underbrace{\underset{N \times 1}{\mathbf{b}}}_{\text{scalar input}} \underbrace{u(t)}_{\text{scalar input}} \quad \underbrace{y(t)}_{\text{scalar output}} = \underbrace{\underset{N \times 1}{\mathbf{c}}^T}_{\text{scalar output}} x(t)$$

$$F(x) = \begin{bmatrix} 2\kappa(x_1) & -\kappa(x_1) & 0 & \cdots & 0 \\ -\kappa(x_2) & 2\kappa(x_2) & \ddots & \ddots & \vdots \\ 0 & \ddots & \ddots & \ddots & 0 \\ \vdots & \ddots & \ddots & 2\kappa(x_{N-1}) & -\kappa(x_{N-1}) \\ 0 & \cdots & 0 & -\kappa(x_N) & \kappa(x_N) \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_N \end{bmatrix} \quad \mathbf{b} = \begin{bmatrix} 1 \\ 0 \\ \vdots \\ \vdots \\ 0 \end{bmatrix} \quad \mathbf{c} = \begin{bmatrix} 0 \\ \vdots \\ \vdots \\ 0 \\ 1 \end{bmatrix}$$

## No Dynamics (Steady-State) Case

- Original System - Single Input/Output

$$0 = \underbrace{A}_{N \times N} x + \underbrace{b}_{N \times 1} \underbrace{u}_{\text{scalar input}} \quad \underbrace{y}_{\text{scalar output}} = \underbrace{c}_{N \times 1}^T x$$

- Reduced System

$$y = - \underbrace{c^T A^{-1} b}_{1 \times 1} u$$

- Satisfies Reduced Model Criteria
  - Cheap to evaluate
  - Exactly reproduces I/O Behavior

## No Dynamics Case

- Original System - Single Input/Output

$$0 = F(x) + \underbrace{b}_{N \times 1} \underbrace{u}_{\text{scalar input}} \quad \underbrace{y}_{\text{scalar output}} = \underbrace{c}_{N \times 1}^T x$$

- Reduced System

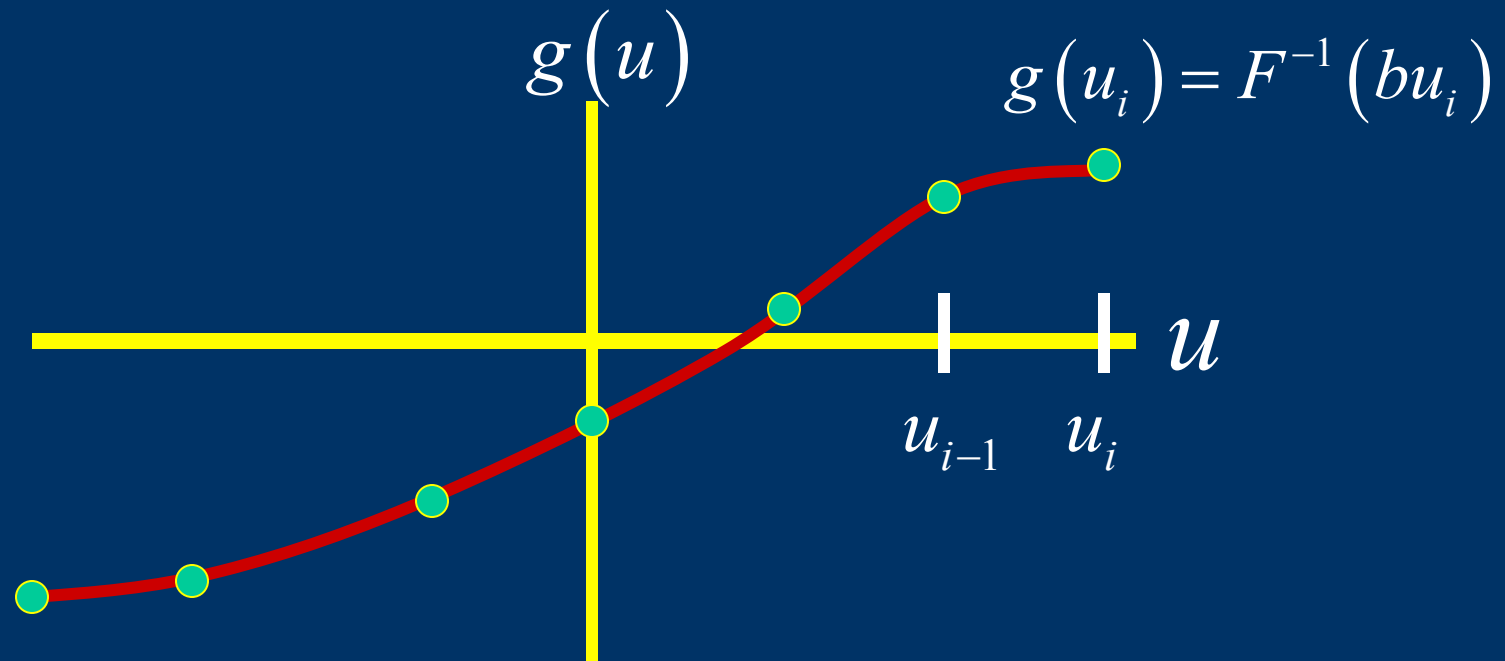
$$y = g(u)$$

- Is “g(u)” a reduced-order model?
  - Depends how we represent g!

## No Dynamics Case

### Representation of Reduced Model

- Could use an interpolated table of data



- Table is a reduced order model
  - Cheap to evaluate
  - Accurate if enough points used

## No Dynamics Case

- Linear Case, one solve, one inner product
  - Solve  $Ax = b \Rightarrow x = A^{-1}b$
  - Form  $c^T x = c^T A^{-1}b$
- Nonlinear Case (if an interpolated table is used)
  - Solve
$$F(x_i) = bu_i \Rightarrow x_i = F^{-1}(bu_i) \text{ for } i = 1, \dots, \#samples$$
  - Form  $c^T x_i = c^T F^{-1}(bu_i) \text{ for } i = 1, \dots, \#samples$
- Nonlinear Reduction adds a representation problem to model reduction

## Dynamic Linear case

- Original Dynamical System - Single Input/Output

$$\frac{dx(t)}{dt} = \underbrace{A}_{N \times N} x(t) + \underbrace{b}_{N \times 1} \underbrace{u(t)}_{\substack{\text{scalar} \\ \text{input}}} \quad \underbrace{y(t)}_{\substack{\text{scalar} \\ \text{output}}} = \underbrace{c}_{N \times 1}^T x(t)$$

- Reduced Dynamical System

$$\frac{dx_r(t)}{dt} = \underbrace{A_r}_{q \times q} x_r(t) + \underbrace{b_r}_{q \times 1} \underbrace{u(t)}_{\substack{\text{scalar} \\ \text{input}}} \quad \underbrace{y_r(t)}_{\substack{\text{scalar} \\ \text{output}}} = \underbrace{c_r}_{q \times 1}^T x_r(t)$$

- $q \ll N$ , but input/output behavior preserved

# Reminder about Eigenanalysis

Consider an ODE:  $\frac{dx(t)}{dt} = Ax(t) + bu(t), \quad x(0) = 0$

Eigendecomposition:  $A = \underbrace{\begin{bmatrix} \vdots & \vdots & \vdots \\ E_1 & E_2 & E_N \\ \vdots & \vdots & \vdots \end{bmatrix}}_E \begin{bmatrix} \lambda_1 & 0 & 0 \\ 0 & \ddots & 0 \\ 0 & 0 & \lambda_N \end{bmatrix} \begin{bmatrix} \vdots & \vdots & \vdots \\ E_1 & E_2 & E_N \\ \vdots & \vdots & \vdots \end{bmatrix}^{-1}$

Change of variables:  $Ew(t) = x(t) \Leftrightarrow w(t) = E^{-1}x(t)$

Substituting:  $\frac{dEw(t)}{dt} = AEw(t) + bu(t), \quad Ew(0) = 0$

Multiply by  $E^{-1}$ :  $\frac{dw(t)}{dt} = E^{-1}AEw(t) + E^{-1}bu(t)$

# Reminder about Eigenanalysis Cont.

## Decoupled Equations

$$\frac{d}{dt} \begin{bmatrix} w_1 \\ \vdots \\ w_N \end{bmatrix} = \begin{bmatrix} \lambda_1 & 0 & 0 \\ 0 & \ddots & 0 \\ 0 & 0 & \lambda_N \end{bmatrix} \begin{bmatrix} w_1 \\ \vdots \\ w_N \end{bmatrix} + \underbrace{\begin{bmatrix} (E^{-1}b)_1 \\ \vdots \\ (E^{-1}b)_N \end{bmatrix}}_{\tilde{b}} u(t)$$

## Output Equation

$$y(t) = c^T x(t) = c^T E w(t) = \underbrace{(E^T c)^T}_{\tilde{c}} w(t)$$



# Reminder about Eigenanalysis Cont.

## Solving Decoupled Equations

$$w_i(t) = \int_0^t e^{\lambda_i(t-\tau)} \tilde{b}_i u(\tau) d\tau$$

Assuming Zero  
Initial Conditions

## Output Equation

$$y(t) = \sum_{i=1}^N \tilde{c}_i w_i(t)$$

## Dynamic Linear Case

$$\begin{bmatrix} \dot{w}_1 \\ \vdots \\ \dot{w}_q \end{bmatrix} = \begin{bmatrix} \lambda_1 & 0 & 0 \\ 0 & \ddots & 0 \\ 0 & 0 & \lambda_q \end{bmatrix} \begin{bmatrix} w_1 \\ \vdots \\ w_q \end{bmatrix} + \begin{bmatrix} \tilde{b}_1 \\ \vdots \\ \tilde{b}_q \end{bmatrix} u(t)$$

## Output Equation

$$y(t) = \sum_{i=1}^q \tilde{c}_i w_i(t)$$

Why?

- Certain modes are not affected by the input

$\tilde{b}_{k+1}, \dots, \tilde{b}_N$  are all small

- Certain modes do not affect the output

$\tilde{c}_{k+1}, \dots, \tilde{c}_N$  are all small

- Keep least negative evals (slowest modes)

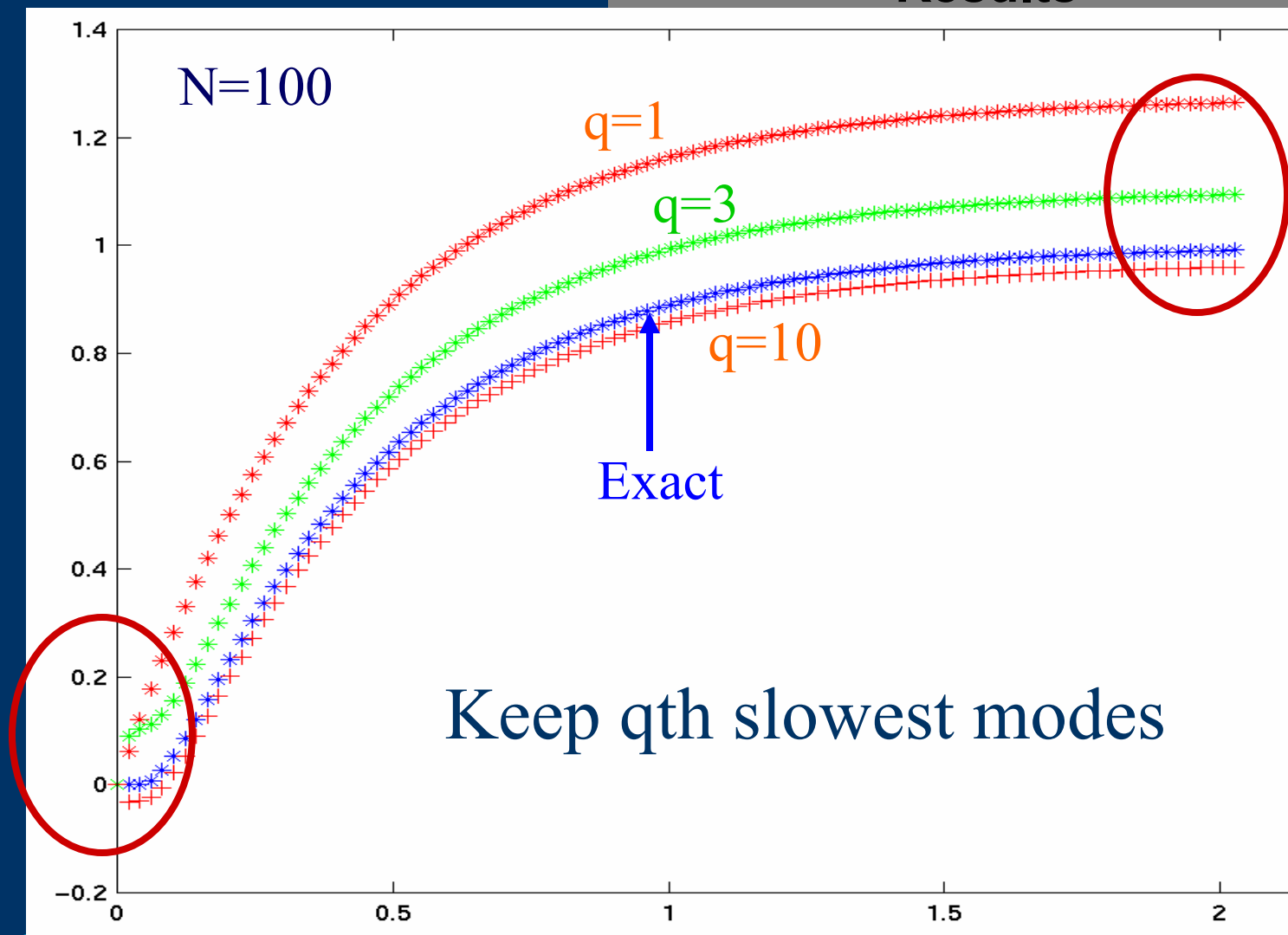
– Look at response to a constant input

$$w_i(t) = \int_0^t e^{\lambda_i(t-\tau)} \tilde{b}_i u d\tau = \frac{1}{\lambda_i} \underbrace{\left( \tilde{b}_i u - \tilde{b}_i u e^{\lambda_i t} \right)}_{\text{Small if } |\lambda_i| \text{ large}}$$

# Reduced models via mode truncation

## Dynamic Linear Case

### Heat Conducting bar Results



# An Aside on Transfer Functions – Laplace Transform

Consider an ODE:  $\frac{dx(t)}{dt} = Ax(t) + bu(t)$

Bilateral Laplace Transform:  $X(s) = \int_{-\infty}^{\infty} x(t)e^{-st} dt$

Key Transform Property:  $sX(s) = \int_{-\infty}^{\infty} \frac{dx(t)}{dt} e^{-st} dt$

Rewrite the ODE in transformed variables

$$sX(s) = AX(s) + bU(s) \quad Y(s) = c^T X(s)$$

$$\Rightarrow Y(s) = \underbrace{c^T (sI - A)^{-1} b}_{H(s)} U(s)$$

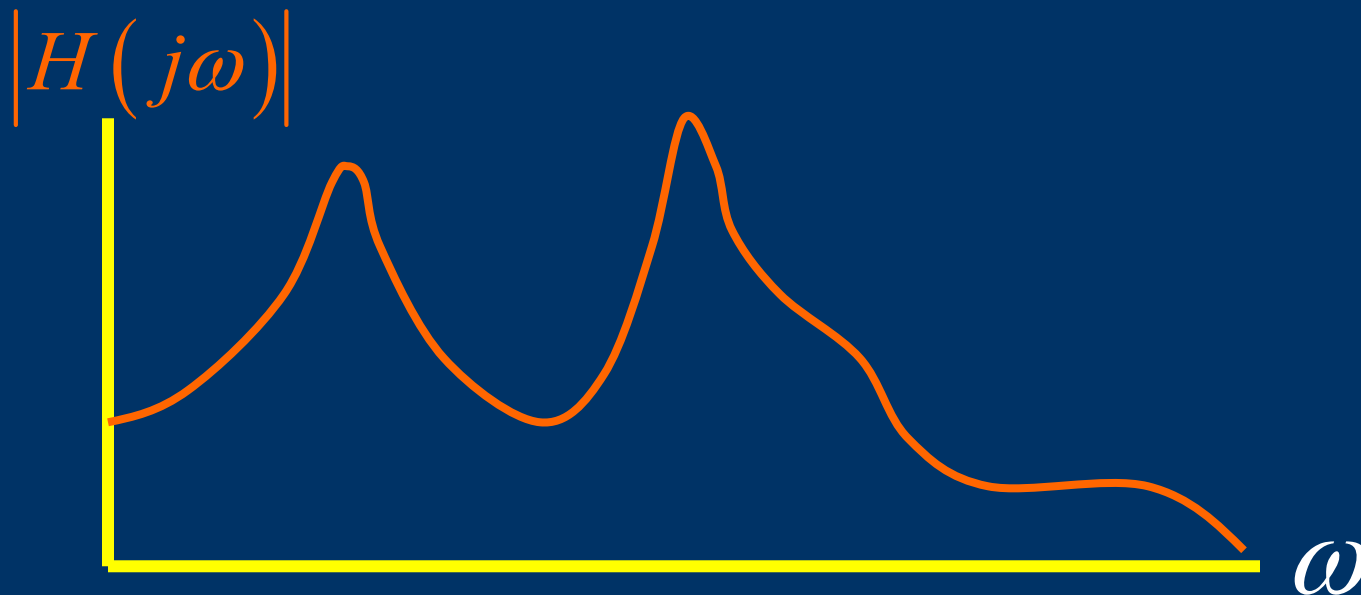
$H(s)$  ← Transfer Function

# An Aside on Transfer Functions – Meaning of $H(s)$

For Stable Systems,  $H(j\omega)$  is the frequency response

If  $u(t) = e^{j\omega t}$  ← Sinusoid

then  $y(t) = H(j\omega) e^{j\omega t}$  Sinusoid with shifted phase and amplitude



# An Aside on Transfer Functions – EigenAnalysis

## Transfer Function

$$H(s) = c^T (sI - A)^{-1} b$$

## Apply Eigendecomposition

$$H(s) = c^T E (sI - \lambda)^{-1} E^{-1} b$$

$$= \tilde{c}^T \begin{bmatrix} \frac{1}{s - \lambda_1} & 0 & 0 \\ 0 & \ddots & 0 \\ 0 & 0 & \frac{1}{s - \lambda_N} \end{bmatrix} \tilde{b} \Rightarrow H(s) = \sum_{i=1}^N \frac{\tilde{c}_i \tilde{b}_i}{s - \lambda_i}$$

### Original System Transfer Function

$$H(s) = \frac{\tilde{c}_1 \tilde{b}_1}{(s - \lambda_1)} + \dots + \frac{\tilde{c}_N \tilde{b}_N}{(s - \lambda_N)} = \underbrace{\frac{b_0 + b_1 s + \dots + b_{N-1} s^{N-1}}{1 + a_1 s + \dots + a_N s^N}}_{\text{Rational Function}}$$

### Reduced Model Transfer Function

$$H_r(s) = \frac{b_0^r + b_1^r s + \dots + b_{q-1}^r s^{q-1}}{\underbrace{1 + a_1^r s + \dots + a_q^r s^q}_{\text{Lower Order Rational Function}}}$$

Lower Order Rational Function

Model Reduction = Find a low order rational function matching  $H(s)$



# Rational Transfer Function Representation

Degrees of Freedom

## Dynamic Linear Case

### Reduced Model Dynamical System

$$\frac{dx_r(t)}{dt} = \underbrace{A_r}_{q \times q} x(t) + \underbrace{b_r}_{q \times 1} \underbrace{u(t)}_{\text{scalar input}} \quad \underbrace{y_r(t)}_{\text{scalar output}} = \underbrace{c_r^T}_{q \times 1} x_r(t)$$

$2q + q^2$   
coefficients

### Reduced Model Transfer Function

$$H_r(s) = \frac{b_0^r + b_1^r s + \dots + b_{q-1}^r s^{q-1}}{1 + a_1^r s + \dots + a_q^r s^q}$$

$2q$   
coefficients

# Rational Transfer Function Representation

## Dynamic Linear Case

Variable Changes Do not change transfer functions

## Reduced Model Transfer Function

$$\frac{dx_r(t)}{dt} = A_r x(t) + b_r u(t) \quad y_r(t) = c_r^T x_r(t)$$

$$\Rightarrow H(s) = c_r^T (sI - A_r)^{-1} b_r$$

## Similarity ( $x = Sw$ ) Transformed Transfer Function

$$\frac{dw_r(t)}{dt} = S^{-1} A_r S w(t) + S^{-1} b_r u(t) \quad y_r(t) = c_r^T S w_r(t)$$

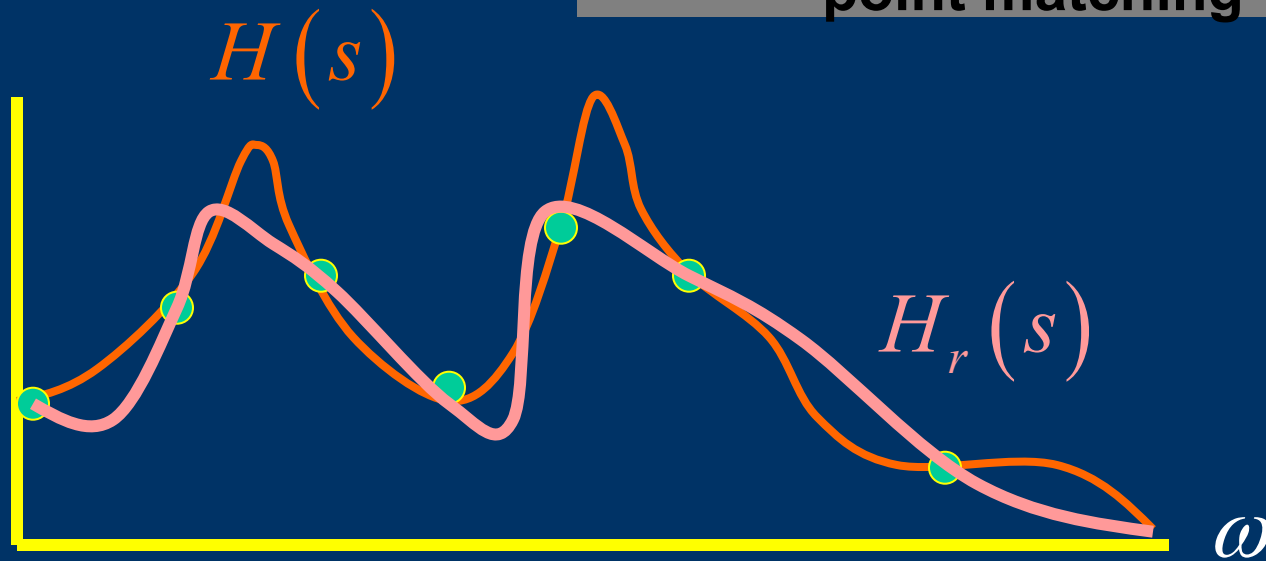
$$\Rightarrow H(s) = c_r^T S (sI - S^{-1} A_r S)^{-1} S^{-1} b_r = c_r^T (sI - A_r)^{-1} b_r$$

**Many Dynamical Systems have the same transfer function!!**

# Rational Transfer Function Representation

## Dynamic Linear Case

### Rational Function Fitting by point matching



- Can match  $2q$  points
- cross multiplying generates a linear system

For  $i = 1$  to  $2q$

$$\left(1 + a_1^r s_i + \cdots + a_q^r s_i^q\right) H(s_i) - \left(b_0^r + b_1^r s + \cdots + b_{q-1}^r s^{q-1}\right) = 0$$

# Rational Transfer Function Representation

Point Matching Matrix can be ill-conditioned

## Dynamic Linear Case

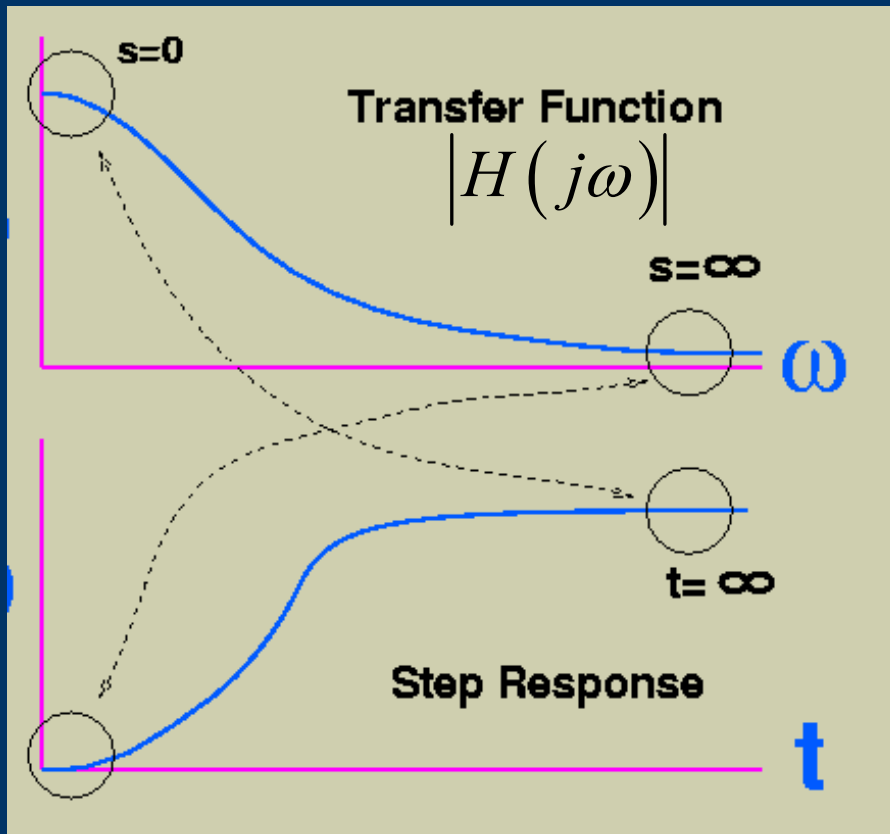
$$\begin{bmatrix} s_1 H(s_1) & s_1^2 H(s_1) & \cdots & -s_1^{q-1} \\ \vdots & \vdots & \cdots & -s_2^{q-1} \\ \vdots & \vdots & \cdots & \vdots \\ s_{2q} H(s_{2q}) & s_{2q}^2 H(s_{2q}) & \cdots & -s_{2q}^{q-1} \end{bmatrix} \begin{bmatrix} a_1 \\ a_2 \\ \vdots \\ b_{q-1} \end{bmatrix} = \begin{bmatrix} H(s_1) \\ H(s_2) \\ \vdots \\ H(s_{2q}) \end{bmatrix}$$

- Columns contain progressively higher powers of the test frequencies
- Must orthogonalize columns during construction

## Dynamic Linear Case

# Rational Transfer Function Representation

Importance of Fitting at low frequency



Correct Steady State behavior requires accurate match at low frequencies

# Rational Transfer Function Representation

## Taylor Series Expansion and Moments

### Dynamic Linear Case

## Original System Transfer Function Moments

$$H(s) = c^T (sI - A)^{-1} b = -c^T \underbrace{(I - sA^{-1})^{-1}}_{\text{Taylor Expand with respect to } s} A^{-1} b$$

Taylor Expand with respect to  $s$

$$H(s) = -c^T (I - sA^{-1})^{-1} A^{-1} b = \sum_{k=0}^{\infty} c^T A^{-(k+1)} b s^k$$

$$H(s) = \underbrace{c^T A^{-1} b}_{m_0} + \underbrace{c^T A^{-2} b}_{m_1} s + \underbrace{c^T A^{-3} b}_{m_2} s^2 + \dots = \sum_{k=0}^{\infty} m_k s^k$$

Moments

# Rational Transfer Function Representation

Moment Matching for accurate low frequency behavior

## Dynamic Linear Case

Reduced Model Matches Original Systems Moments

$$H_r(s) = \frac{b_0^r + b_1^r s + \dots + b_{q-1}^r s^{q-1}}{1 + a_1^r s + \dots + a_q^r s^q} = m_0 + m_1 s + \dots + m_{2q-1} s + \dots$$

Cross-Multiplying and Matching Terms

$$\begin{bmatrix} m_0 & m_1 & \dots & m_{k-1} \\ m_1 & \dots & \dots & \vdots \\ \vdots & \dots & \dots & m_{2q-3} \\ m_{k-1} & \dots & m_{2q-3} & m_{2q-2} \end{bmatrix} \begin{bmatrix} a_q \\ a_{q-1} \\ \vdots \\ a_1 \end{bmatrix} = \begin{bmatrix} m_q \\ m_{q+1} \\ \vdots \\ m_{2q-1} \end{bmatrix}$$

# Rational Transfer Function Representation

## Dynamic Linear Case

### Explicit Moment Matching Problem

System of equations extremely ill-conditioned

$$\begin{bmatrix} m_0 & m_1 & \cdots & m_{k-1} \\ m_1 & \cdots & \cdots & \vdots \\ \vdots & \cdots & \cdots & m_{2q-3} \\ m_{k-1} & \cdots & m_{2q-3} & m_{2q-2} \end{bmatrix} \begin{bmatrix} a_q \\ a_{q-1} \\ \vdots \\ a_1 \end{bmatrix} = \begin{bmatrix} m_q \\ m_{q+1} \\ \vdots \\ m_{2q-1} \end{bmatrix}$$

$$m_i = c^T A^{-i} b \approx \lambda_{A_{\max}} m_{i-1}$$

Columns become linearly dependent for large  $q$ !



# Rational Transfer Function Representation

## Dynamic Linear Case

Problems with explicit fitting methods

- Linear Systems for fitting ill-conditioned
  - Need specialized algorithms which avoid explicit fitting matrix construction
- Rational function must be converted to state-space
  - Needed by most simulation tools
  - Requires root finding procedure, very sensitive to parameter variation

# Summary

- Need For Model Reduction
  - Circuits, MEMS, Optics, Jet Engines
- Simple Example Problem
  - Heat Conducting bar example
- Steady-State Case (linear and nonlinear)
- Dynamic Linear Case
  - Truncating Eigenmodes
    - Loss correct steady state values
    - Select modes to delete
  - Rational Function Fitting
    - Generates ill-conditioned matrices