

**10.40 Thermodynamics**  
**Problem Set 1**

**Fall 2003**

Problem 3.15 Text

**Solution:**

(a)

Balloon inflation is modeled by a chamber of air pushing against a spring.

The pressure in the gas space is given by

$$P - P_i = k(L - L_i)$$

where  $k = \text{constant} = 5 \text{ bar/m}$

Figure removed. Please see “Tester, J. W., and Michael Modell. *Thermodynamics and Its Applications*. Upper Saddle River, NJ: Prentice Hall PTR, 1997, p. 64. Fig. P3.15.”

$P_i = 1 \text{ bar}$ ;  $T_i = 300 \text{ K}$ ;  $L_i = 0.15 \text{ m}$ ; piston area,  $A = 0.02 \text{ m}^2$ ;  $C_v = 20.9 \text{ J/mol K}$ ,  $C_p = 29.2 \text{ J/mol K}$

What is the air temperature ( $T_f$ ) when  $L = 0.6 \text{ m}$ ?

Assumptions:

Ideal gas

$$C_p, C_v \neq f(T)$$

Adiabatic process

Quasi-static process (air is well mixed)

Subscript	System
i	initial air in cylinder
f	final air in cylinder
in	air entering cylinder

Solution:

Choose air in cylinder as system. It is a simple system since it has no inertial or body forces, so  $\underline{E} \rightarrow \underline{U}$ . It is an open system due to the influx of air. The air in is from a constant pressure and temperature reservoir, so it has a constant enthalpy ( $H_{in}$ ).

First Law for an Open System

$$\begin{aligned} d\underline{U} &= \cancel{\delta Q} + \delta W + H_{in} \delta n_{in} - \cancel{H_{out} \delta n_{out}}, \quad \delta n_{in} = dN \\ &= 0 \qquad \qquad \qquad = 0 \end{aligned} \tag{1}$$

$$\begin{aligned} \delta W &= -Pd\underline{V} \quad \text{work done by the system on the environment} \\ &= -[P_i + k(L - L_i)]AdL \end{aligned}$$

$$\begin{aligned} \int_{U_i}^{U_f} d\underline{U} &= \int_{L_i}^{L_f} -[P_i + k(L - L_i)]AdL + \int_{N_i}^{N_f} H_{in} dN \\ \underline{U}_f - \underline{U}_i &= -A \left[ P_i(L_f - L_i) + \frac{k}{2}(L_f - L_i)^2 \right] + H_{in}(N_f - N_i) \\ \underline{U} &= NU \\ A \left[ P_i(L_f - L_i) + \frac{k}{2}(L_f - L_i)^2 \right] &= N_f(H_{in} - U_f) - N_i(H_{in} - U_i) \end{aligned} \quad (2)$$

From Table 3.2

$$\begin{aligned} H_{in} &= \int_{T_0}^{T_{in}} C_p dT + H_0 = C_p(T_{in} - T_0) + H_0 \\ U_f &= \int_{T_0}^{T_f} C_v dT + U_0 = C_v(T_f - T_0) + U_0 \end{aligned} \quad (3)$$

Inserting equation (3) into the RHS terms of equation (2) and simplifying,

$$N_f(H_{in} - U_f) = N_f(C_p T_{in} - C_v T_f) + N_f \left[ \underbrace{(C_v - C_p)T_0}_{-RT_0} + \underbrace{H_0 - U_0}_{RT_0} \right] \quad (4)$$

$$\text{Similarly, } H_{in} - U_i = N_i(C_p T_{in} - C_v T_i)$$

$$T_{in} = T_i$$

$$\begin{aligned} N_i(C_p T_{in} - C_v T_i) &= N_i RT_i = P_i \underline{V}_i = P_i A L_i \\ N_f &= P_f A L_f / RT_f \end{aligned} \quad (5)$$

Inserting equations (4) and (5) into (2) and canceling terms,

$$A \left[ P_i(L_f - L_i) + \frac{k}{2}(L_f - L_i)^2 \right] = \frac{P_f A L_f}{RT_f} (C_p T_{in} - C_v T_f) - P_i A L_i \quad (6)$$

$$T_f = \frac{T_{in} P_f L_f \left( \frac{C_p}{R} \right)}{P_i L_f + \frac{k}{2} (L_f - L_i)^2 + \frac{P_f L_f C_v}{R}}$$

plugging in  $P_f = P_i + k(L_f - L_i) = 3.25 \text{ bar}$   
yields  $T_f = 345 \text{ K}$

(7)