

## 12.520 Lecture Notes 19

### Plates (continued)

Flexural equation:  $D \frac{d^4 w}{dx^4} + P \frac{d^2 w}{dx^2} + \Delta \rho g w = q(x)$

where  $D = \frac{Eh^3}{12(1-\nu^2)}$ .

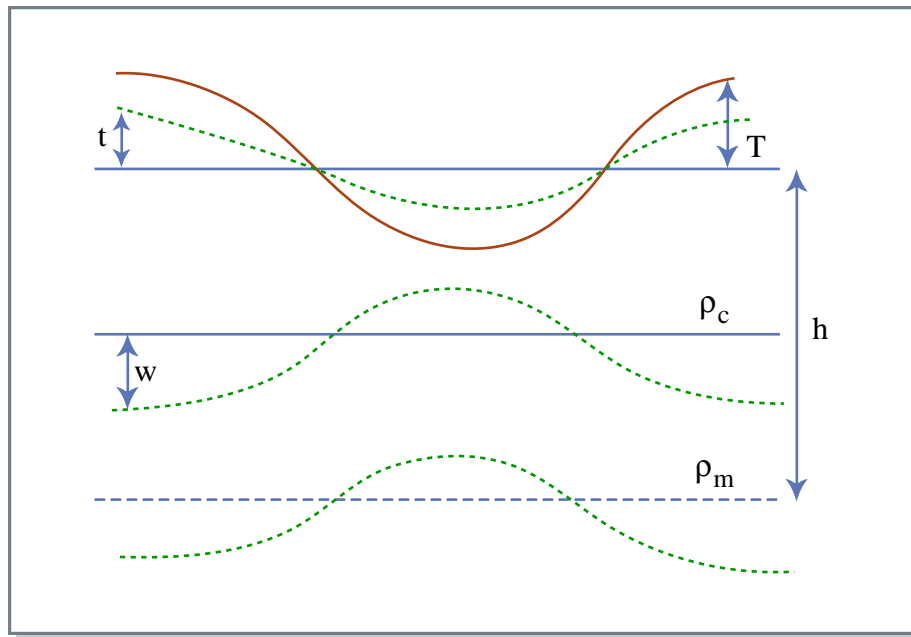


Figure 19.1  
Figure by MIT OCW.

$$T = T_0 \cos kx = T_0 \cos \frac{2\pi x}{\lambda}$$

“Harmonic” load

$$t = t_0 \cos kx, \quad w = w_0 \cos kx, \quad t_0 = T_0 - w_0$$

$$D \frac{d^4 w}{dx^4} + \rho_m g w = t_0 \rho_c g \cos kx \text{ when } P = 0$$

$$k^4 D w_0 \cos kx + \rho_m g w_0 \cos kx = t_0 \rho_c g \cos kx$$

$$w_0 = \frac{t_0}{\frac{\rho_m}{\rho_c} - 1 + \frac{Dk^4}{\rho_c g}}$$

Call  $2\pi \left( \frac{D}{\rho_c g} \right)^{1/4} \equiv \lambda_f$  flexural wavelength

For  $\lambda \neq \lambda_f$ ,  $(\lambda_f k)^4 = 1$ ,  $w_0 = \frac{t_0}{\frac{\rho_m}{\rho_c} - 1}$ ; isostasy

For  $\lambda = \lambda_f$ ,  $w_0 = 0 \Rightarrow$  uncompensated

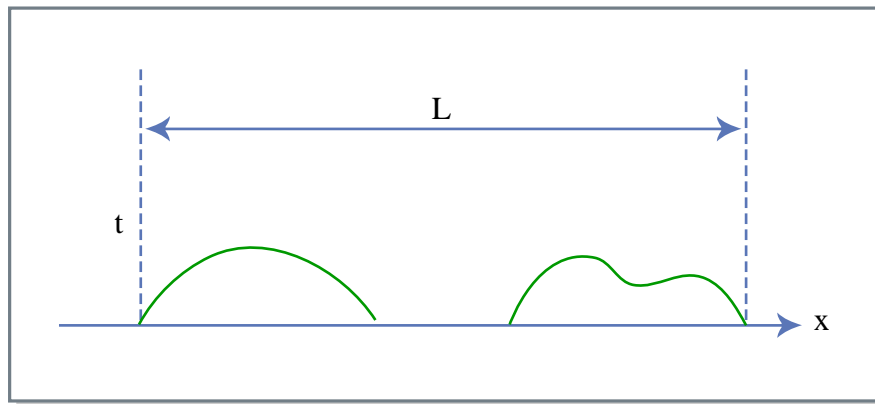


Figure 19.2  
Figure by MIT OCW.

$$t(x) = \sum_{n=0}^{\infty} \left[ t_n^c \cos \frac{2\pi n x}{L} + t_n^s \sin \frac{2\pi n x}{L} \right]$$

Find  $t_n^c, t_n^s$  Assume D, calculate  $w_n^s, w_n^c$

Synthesize  $w(x)$ , compare to observations.

## Plate subject to an end load

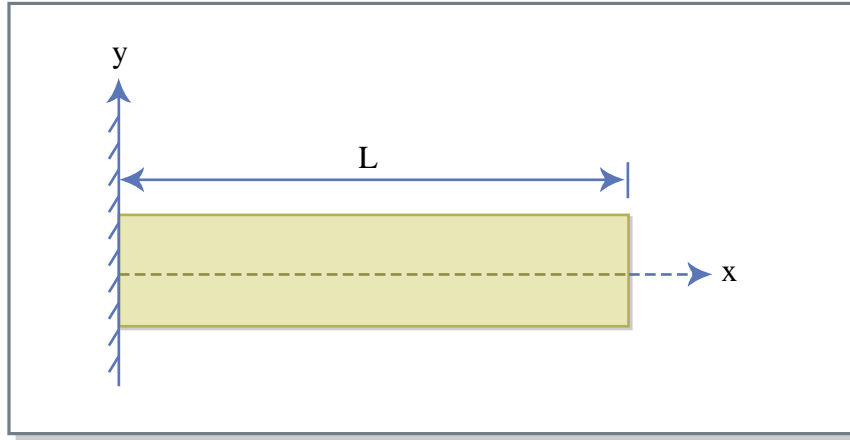


Figure 19.3  
Figure by MIT OCW.

Shear force:  $\frac{dV}{dx} = -q$

Since  $q = 0$ ,  $V = \text{const.} = V_a$

Bending moment:

$$\frac{dM}{dx} = V + P \frac{dw}{dx}$$

Since  $P = 0$ ,  $\frac{dM}{dx} = V \Rightarrow M = V_a x + \text{const} \Rightarrow 0$  at  $x = L$

$$M = V_a(x - L)$$

Displacement:

$$\frac{d^4 w}{dx^4} = 0 \Rightarrow \frac{d^3 w}{dx^3} = \text{const}$$

$$\text{But } M = -D \frac{d^2 w}{dx^2} = 0, \quad \frac{dM}{dx} = -D \frac{d^3 w}{dx^3} = V_a$$

$$\frac{d^3 w}{dx^3} = -\frac{V_a}{D}$$

$$\frac{d^2 w}{dx^2} = -\frac{V_a}{D}(x - L)$$

$$\text{Subject to } w, \frac{dw}{dx} = 0 \text{ at } x = 0$$

$$w = \frac{V_a x^2}{2D} \left( L - \frac{x}{3} \right) \Rightarrow \text{cubic displacement}$$

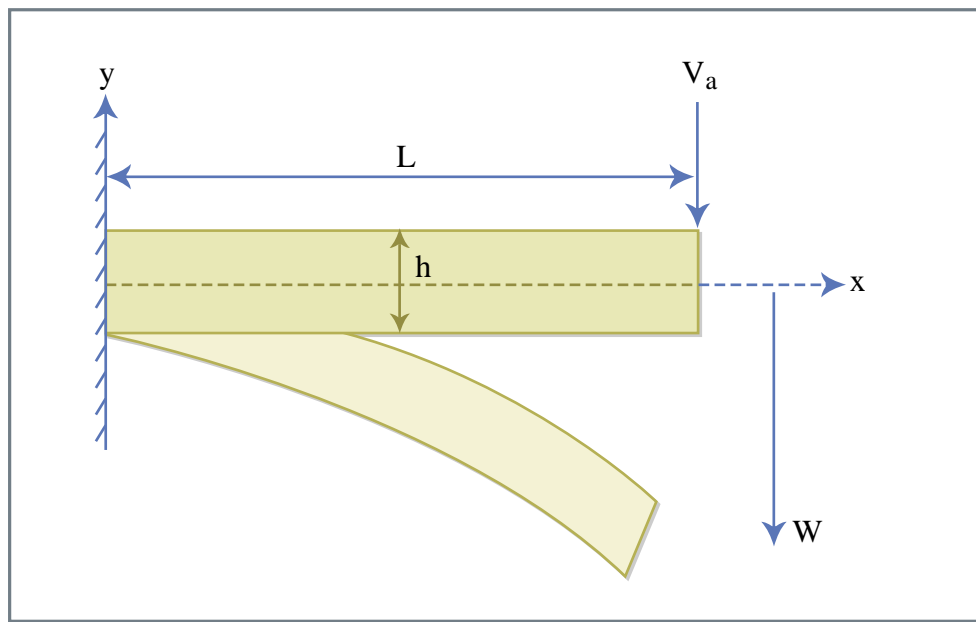


Figure 19.4

Figure by MIT OCW.

$$w = \frac{V_a}{2D} x^2 \left( L - \frac{x}{3} \right)$$

$$\text{Assumption } |\sigma_{xy}| = |\sigma_{xx}|$$

$$\sigma_{xx} = \frac{E}{1-\nu^2} \varepsilon_{xx}$$

$$\varepsilon_{xx} = -y \frac{d^2 w}{dx^2}$$

$$M = -D \frac{d^2 w}{dx^2}$$

$$\varepsilon_{xx} = \frac{y}{D} M$$

$$\sigma_{xx}^{\max} = \frac{E}{1-\nu^2} \frac{h}{2} \frac{1}{D} V_a L = \frac{6V_a L}{h^2} = \frac{6V_a}{h} \left( \frac{L}{h} \right)$$

$$\langle \sigma_{xy} \rangle = \frac{V_a}{h} = \frac{1}{6} \frac{h}{L} \sigma_{xx}^{\max}$$