

16.001 Unified Engineering Materials and Structures

Concept and uses of stress at a point

Reading assignments: CDL: 4.2-4.7

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- 1 Mathematical preliminaries
 - Indicial notation and summation convention
 - Vectors
 - Transformation of basis

- 2 Concept and uses of stress at a point
 - Stress at a point
 - Stress tensor

Indicial notation and summation convention

A convenient way to write complicated expressions involving vectors and tensor.

Definitions

Free index: A subscript index $i = 1, 3, \dots$ will be denoted a *free index* if it is not repeated in the same additive term where the index appears. *Free* means that the index represents **all** the values in its range.

- Latin indices will range from 1 to 3, ($i, j, k, \dots = 1, 2, 3$),
- greek indices will range from 1 to 2, ($\alpha, \beta, \gamma, \dots = 1, 2$).

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Examples

- 1 a_{i1} implies a_{11}, a_{21}, a_{31} . (one free index)
- 2 $x_\alpha y_\beta$ implies $x_1 y_1, x_1 y_2, x_2 y_1, x_2 y_2$ (two free indices).
- 3 a_{ij} implies $a_{11}, a_{12}, a_{13}, a_{21}, a_{22}, a_{23}, a_{31}, a_{32}, a_{33}$ (two free indices implies 9 values).
- 4 $\frac{\partial \sigma_{ij}}{\partial x_j} + b_i = 0$ has a free index (i), therefore it represents three equations:

$$\frac{\partial \sigma_{1j}}{\partial x_j} + b_1 = 0, \quad \frac{\partial \sigma_{2j}}{\partial x_j} + b_2 = 0, \quad \frac{\partial \sigma_{3j}}{\partial x_j} + b_3 = 0$$

Indicial notation and summation convention, continued

Definitions

Summation convention: When a *repeated index* is found in an expression (inside an additive term) the summation of the terms ranging all the possible values of the indices is implied, i.e.:

$$a_i b_i = \sum_{i=1}^3 a_i b_i = a_1 b_1 + a_2 b_2 + a_3 b_3$$

Note that the choice of index is immaterial:

$$a_i b_i = a_k b_k$$

Examples

① $a_{ii} = a_{11} + a_{22} + a_{33}$

② $t_i = \sigma_{ij} n_j$ implies the **three** equations (why?):

$$t_1 = \sigma_{11} n_1 + \sigma_{12} n_2 + \sigma_{13} n_3$$

$$t_2 = \sigma_{21} n_1 + \sigma_{22} n_2 + \sigma_{23} n_3$$

$$t_3 = \sigma_{31} n_1 + \sigma_{32} n_2 + \sigma_{33} n_3$$

Other important rules about indicial notation:

- 1 An index cannot appear more than twice in a single additive term, it's either free or repeated only once.

$$a_i = b_{ij}c_jd_j \text{ is INCORRECT}$$

- 2 In an equation the *lhs* and *rhs*, as well as all the terms on both sides must have the same free indices
 - $a_i b_k = c_{ij} d_{kj}$ free indices i, k , CORRECT
 - $a_i b_k = c_{ij} d_{kj} + e_i f_{jj} + g_k p_i q_r$ INCORRECT, second term is missing free index k and third term has extra free index r

Definition of a basis in \mathbb{R}^3

A *basis* in \mathbb{R}^3 is given by any set of linearly independent vectors \mathbf{e}_i , $(\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3)$. From now on, we will assume that these basis vectors are orthonormal, i.e., they have a unit length and they are orthogonal with respect to each other. This can be expressed using dot products:

$$\begin{aligned}\mathbf{e}_1 \cdot \mathbf{e}_1 &= 1, \mathbf{e}_2 \cdot \mathbf{e}_2 = 1, \mathbf{e}_3 \cdot \mathbf{e}_3 = 1, \\ \mathbf{e}_1 \cdot \mathbf{e}_2 &= 0, \mathbf{e}_1 \cdot \mathbf{e}_3 = 0, \mathbf{e}_2 \cdot \mathbf{e}_3 = 0, \dots\end{aligned}$$

The Kronecker Delta

Using indicial notation we can write these expressions in very succinct form as follows:

$$\mathbf{e}_i \cdot \mathbf{e}_j = \delta_{ij}$$

In the last expression the symbol δ_{ij} is defined as the *Kronecker delta*:

$$\delta_{ij} = \begin{cases} 1 & \text{if } i = j, \\ 0 & \text{if } i \neq j \end{cases}$$

Examples

$$\begin{aligned}
 a_i \delta_{ij} &= a_1 \delta_{11} + a_2 \delta_{21} + a_3 \delta_{31}, \\
 &\quad a_1 \delta_{12} + a_2 \delta_{22} + a_3 \delta_{32}, \\
 &\quad a_1 \delta_{13} + a_2 \delta_{23} + a_3 \delta_{33} \\
 &= a_1 1 + a_2 0 + a_3 0, \\
 &\quad a_1 0 + a_2 1 + a_3 0, \\
 &\quad a_1 0 + a_2 0 + a_3 \\
 &= a_1, \\
 &\quad a_2, \\
 &\quad a_3
 \end{aligned}$$

or more succinctly: $a_i \delta_{ij} = a_j$, i.e., the Kronecker delta can be thought of an “index replacer”.

Definition of a vector

A **vector** \mathbf{v} will be represented as:

$$\mathbf{v} = v_i \mathbf{e}_i = v_1 \mathbf{e}_1 + v_2 \mathbf{e}_2 + v_3 \mathbf{e}_3$$

The v_i are the *components* of \mathbf{v} in the basis \mathbf{e}_i . These components are the projections of the vector on the basis vectors:

$$\mathbf{v} = v_j \mathbf{e}_j$$

Taking the dot product with basis vector \mathbf{e}_i :

$$\mathbf{v} \cdot \mathbf{e}_i = v_j (\mathbf{e}_j \cdot \mathbf{e}_i) = v_j \delta_{ji} = v_i$$

Transformation of basis

Given two bases $\mathbf{e}_i, \tilde{\mathbf{e}}_k$ and a vector \mathbf{v} whose components in each of these bases are v_i and \tilde{v}_k , respectively, we seek to express the components in basis in terms of the components in the other basis. Since the vector is unique:

$$\mathbf{v} = \tilde{v}_m \tilde{\mathbf{e}}_m = v_n \mathbf{e}_n$$

Taking the dot product with $\tilde{\mathbf{e}}_i$:

$$\mathbf{v} \cdot \tilde{\mathbf{e}}_i = \tilde{v}_m (\tilde{\mathbf{e}}_m \cdot \tilde{\mathbf{e}}_i) = v_n (\mathbf{e}_n \cdot \tilde{\mathbf{e}}_i)$$

But $\tilde{v}_m (\tilde{\mathbf{e}}_m \cdot \tilde{\mathbf{e}}_i) = \tilde{v}_m \delta_{mi} = \tilde{v}_i$ from which we obtain:

$$\tilde{v}_i = \mathbf{v} \cdot \tilde{\mathbf{e}}_i = v_j (\mathbf{e}_j \cdot \tilde{\mathbf{e}}_i)$$

Note that $(\mathbf{e}_j \cdot \tilde{\mathbf{e}}_i)$ are the *direction cosines* of the basis vectors of one basis on the other basis:

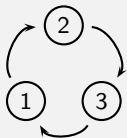
$$\mathbf{e}_j \cdot \tilde{\mathbf{e}}_i = \|\mathbf{e}_j\| \|\tilde{\mathbf{e}}_i\| \widehat{\cos \mathbf{e}_j \tilde{\mathbf{e}}_i}$$

Permutation tensor, cross product I

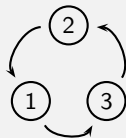
Definition

Permutation Tensor:

$$\epsilon_{mnp} = \begin{cases} 0 & \text{when any two indices are equal} \\ 1 & \text{when } mnp \text{ is an even permutation of } 1, 2, 3 \\ -1 & \text{when } mnp \text{ is an odd permutation of } 1, 2, 3 \end{cases}$$



Even permutation



Odd permutation

Examples

$$\epsilon_{123} = \epsilon_{231} = \epsilon_{312} = 1, \quad \epsilon_{213} = \epsilon_{321} = \epsilon_{132} = -1, \quad \epsilon_{112} = \epsilon_{233} = \epsilon_{222} = \dots = 0,$$

Source and usefulness: Cross products of cartesian basis vectors

$$\mathbf{e}_1 \times \mathbf{e}_1 = \mathbf{0}, \quad \mathbf{e}_2 \times \mathbf{e}_2 = \mathbf{0}, \quad \mathbf{e}_3 \times \mathbf{e}_3 = \mathbf{0},$$

$$\mathbf{e}_1 \times \mathbf{e}_2 = (+1)\mathbf{e}_3, \quad \mathbf{e}_2 \times \mathbf{e}_3 = (+1)\mathbf{e}_1, \quad \mathbf{e}_3 \times \mathbf{e}_1 = (+1)\mathbf{e}_2,$$

$$\mathbf{e}_1 \times \mathbf{e}_3 = (-1)\mathbf{e}_2, \quad \mathbf{e}_3 \times \mathbf{e}_2 = (-1)\mathbf{e}_1, \quad \mathbf{e}_2 \times \mathbf{e}_1 = (-1)\mathbf{e}_3,$$

These can all be captured by

$$\mathbf{e}_i \times \mathbf{e}_j = \epsilon_{ijk} \mathbf{e}_k$$

Permutation tensor, cross product III

Example

$$\mathbf{e}_1 \times \mathbf{e}_2 = \underbrace{\epsilon_{121}}_{=0} \mathbf{e}_1 + \underbrace{\epsilon_{122}}_{=0} \mathbf{e}_2 + \underbrace{\epsilon_{123}}_{=1} \mathbf{e}_3 = \mathbf{e}_3$$

We can also observe that

$$(\mathbf{e}_i \times \mathbf{e}_j) \cdot \mathbf{e}_k = (\epsilon_{ijl} \mathbf{e}_l) \cdot \mathbf{e}_k = \epsilon_{ijl} \delta_{kl} = \epsilon_{ijk}$$

The permutation tensor can be used to express some of the other familiar vector operations involving cross products.

Cross product of two vectors

$$\mathbf{v} \times \mathbf{w} = (v_i \mathbf{e}_i) \times (w_j \mathbf{e}_j) = v_i w_j (\mathbf{e}_i \times \mathbf{e}_j) = v_i w_j \epsilon_{ijk} \mathbf{e}_k$$

Mixed or triple product of three vectors

$$(\mathbf{v} \times \mathbf{w}) \cdot \mathbf{u} = v_i w_j \epsilon_{ijl} \mathbf{e}_l \cdot (u_k \mathbf{e}_k) = v_i w_j u_k \epsilon_{ijk}$$

Since the triple product can also be obtained from the determinant of the 3×3 matrix made of the components of the three vectors (either arranged in row or column form), we can use this to express the determinant of a 3×3 matrix A with components a_{ij} as follows. Assign the rows of the matrix to the components of the vectors above as follows: $v_i = a_{1i}$, $w_j = a_{2j}$, $u_k = a_{3k}$, then:

Determinant of a 3×3 Matrix

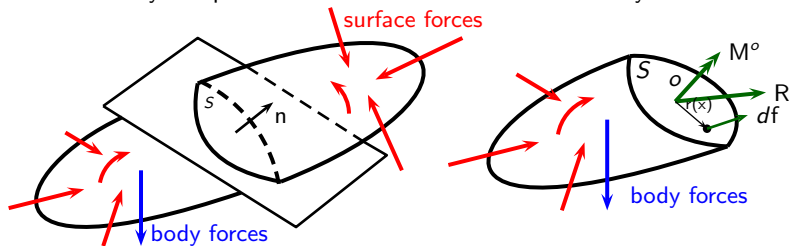
$$|A| = a_{1i} a_{2j} a_{3k} \epsilon_{ijk}$$

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Internal forces and equilibrium I

Consider a body in equilibrium under external surface and body forces:



We imagine a cut through the body with a plane defining the surface internal surface S with normal \mathbf{n} . The FBD on the right shows the resultant internal force (\mathbf{R}) and moment (\mathbf{M}^o) required from equilibrium for the left part. We assume that these are provided by the collective action of infinitesimal pointwise forces $d\mathbf{f}$ acting on the points of S .

\mathbf{R} and \mathbf{M}^o are therefore obtained as the following integrals:

$$\mathbf{R} = \int_S d\mathbf{f}$$
$$\mathbf{M}^o = \int_S \mathbf{r} \times d\mathbf{f}$$

It should be clear that \mathbf{t} is a force per unit area defined at each point of surface S obtained as the limiting value of the resultant force \mathbf{f} acting on a (finite) surface area element ΔS when this tends to zero.

Definition

The *stress vector* at a point on ΔS is defined as:

$$\mathbf{t} = \lim_{\Delta S \rightarrow 0} \frac{\mathbf{f}}{\Delta S} \quad (1)$$

Notes:

- The integral of the stress vector in the area defines the resultant internal force necessary to keep the left side in equilibrium (as we discussed before)

$$\mathbf{R} = \int_S \mathbf{t} dS$$

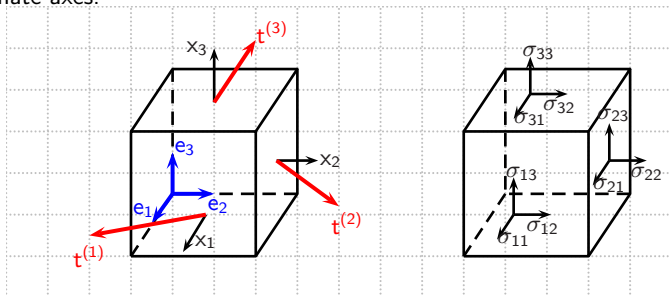
- Similarly, the resultant internal moment vector \mathbf{M}^o with respect to a point o is given by:

$$\mathbf{M}^o = \int_S \mathbf{r} \times \mathbf{t} dS$$

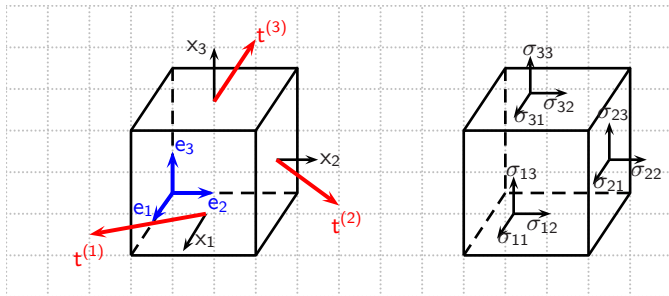
Stress Vector II

In other words, the internal force and moment vectors are the equipollent force system corresponding to the collective action of the continuously distributed stress vectors \mathbf{t} on the cut surface.

If the cut had gone through the same point under consideration but along a plane with a different normal, the stress vector would have been different. Let's consider the three stress vectors $\mathbf{t}^{(i)}$ acting on the planes normal to the coordinate axes.



Stress Vector III



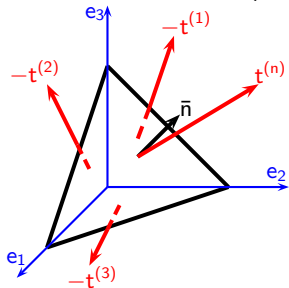
Let's also decompose each $\mathbf{t}^{(i)}$ in its three components in the coordinate system \mathbf{e}_i (this can be done for any vector) as (see Figure):

$$\mathbf{t}^{(i)} = \sigma_{ij} \mathbf{e}_j \quad (2)$$

σ_{ij} is the component of the stress vector $\mathbf{t}^{(i)}$ along the \mathbf{e}_j direction.

Introduction of the Stress Tensor I

Different planes passing through the point with different normals will have different stress vectors $\mathbf{t}^{(n)}$. Is there a relation among them? To answer this invoke equilibrium of the (shrinking) tetrahedron of material:



Cauchy's tetrahedron:
equilibrium of a tetrahedron
shrinking to a point.

The area of the faces of the tetrahedron are ΔS_1 , ΔS_2 , ΔS_3 and ΔS .

We have used Newton's third law of action and reaction: $\mathbf{t}^{(-n)} = -\mathbf{t}^{(n)}$. To enforce equilibrium, we must consider the force vectors (stress vectors multiplied by respective areas) acting on each face of the tetrahedron:

$$\mathbf{t}^{(n)} \Delta S - \mathbf{t}^{(1)} \Delta S_1 - \mathbf{t}^{(2)} \Delta S_2 - \mathbf{t}^{(3)} \Delta S_3 = 0$$

Introduction of the Stress Tensor II

The surface area elements are related by the following formula: $\Delta S n_i = \Delta S_i$.
Proof in the following mathematical aside:

By virtue of Green's Theorem:

$$\int_V \nabla \phi dV = \int_S \mathbf{n} \phi dS$$

applied to the function $\phi = 1$, we get

$$0 = \int_S \mathbf{n} dS$$

which applied to our tetrahedron gives:

$$0 = \Delta S \mathbf{n} - \Delta S_1 \mathbf{e}_1 - \Delta S_2 \mathbf{e}_2 - \Delta S_3 \mathbf{e}_3$$

If we take the scalar product of this equation with \mathbf{e}_i , we obtain:

$$\Delta S (\mathbf{n} \cdot \mathbf{e}_i) = \Delta S_i$$

or

$$\boxed{\Delta S_i = \Delta S n_i}$$

Replace in equilibrium expression:

$$\Delta S(\mathbf{t}^{(n)} - \underbrace{n_1}_{(\mathbf{n} \cdot \mathbf{e}_1)} \mathbf{t}^{(1)} - \underbrace{n_2}_{(\mathbf{n} \cdot \mathbf{e}_2)} \mathbf{t}^{(2)} - \underbrace{n_3}_{(\mathbf{n} \cdot \mathbf{e}_3)} \mathbf{t}^{(3)}) = 0$$

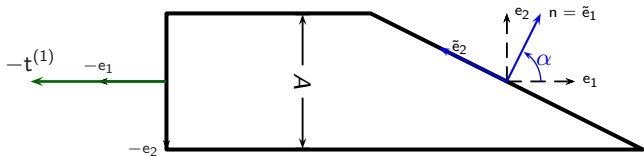
which can be written more simply (using summation convention) as:

$$\mathbf{t}^{(n)} = \underbrace{(\mathbf{n} \cdot \mathbf{e}_i)}_{n_i} \mathbf{t}^{(i)} \quad (3)$$

Introduction of the Stress Tensor IV

Example

Consider a cut at an angle α in a truss member of cross sectional area A and subject to a force of magnitude F . The bar is subject to a uniform uniaxial stress $\sigma = \frac{F}{A}$. The stress vector at any point on a plane of normal \mathbf{e}_1 is (see figure) $\mathbf{t}^{(1)} = \sigma \mathbf{e}_1$. $\mathbf{t}^{(2)} = \mathbf{t}^{(3)} = \mathbf{0}$. What is $\mathbf{t}^{(n)}$ where $\mathbf{n} = \tilde{\mathbf{e}}_1 = \cos(\alpha)\mathbf{e}_1 + \sin(\alpha)\mathbf{e}_2$?



$$\mathbf{t}^{(n)} = \mathbf{t}^{(\tilde{\mathbf{e}}_1)} = n_i \mathbf{t}^{(i)} = n_1 \mathbf{t}^{(1)} = \cos(\alpha) \sigma \mathbf{e}_1 = \frac{F}{\frac{A}{\cos(\alpha)}} \mathbf{e}_1$$

This gives us $\mathbf{t}^{(n)}$ in the basis \mathbf{e}_i . What about in basis $\tilde{\mathbf{e}}_i$?

$$\begin{aligned} \mathbf{t}^{(\tilde{\mathbf{e}}_1)} &= \left(\mathbf{t}^{(\tilde{\mathbf{e}}_1)} \cdot \tilde{\mathbf{e}}_1 \right) \tilde{\mathbf{e}}_1 + \left(\mathbf{t}^{(\tilde{\mathbf{e}}_1)} \cdot \tilde{\mathbf{e}}_2 \right) \tilde{\mathbf{e}}_2 = \left(\mathbf{t}^{(\tilde{\mathbf{e}}_1)} \cdot \tilde{\mathbf{e}}_i \right) \tilde{\mathbf{e}}_i \\ &= (\cos(\alpha) \sigma \mathbf{e}_1 \cdot \tilde{\mathbf{e}}_1) \tilde{\mathbf{e}}_1 + (\cos(\alpha) \sigma \mathbf{e}_1 \cdot \tilde{\mathbf{e}}_2) \tilde{\mathbf{e}}_2 \\ &= \sigma \cos^2(\alpha) \tilde{\mathbf{e}}_1 + \sigma \cos(\alpha)(-\sin(\alpha)) \tilde{\mathbf{e}}_2 \end{aligned}$$

Definition of stress tensor I

Going back to Eqn. (3), we can also pull \mathbf{n} as a “common factor” and create a new type of mathematical expression (tensor product):

$$\mathbf{t}^{(n)} = \mathbf{n} \cdot (\mathbf{e}_1 \mathbf{t}^{(1)} + \mathbf{e}_2 \mathbf{t}^{(2)} + \mathbf{e}_3 \mathbf{t}^{(3)}) = \mathbf{n} \cdot (\mathbf{e}_1 \otimes \mathbf{t}^{(1)} + \mathbf{e}_2 \otimes \mathbf{t}^{(2)} + \mathbf{e}_3 \otimes \mathbf{t}^{(3)}) \quad (4)$$

The factor in parenthesis is the definition of the *Cauchy stress tensor* σ :

Definition

$$\sigma = \mathbf{e}_1 \otimes \mathbf{t}^{(1)} + \mathbf{e}_2 \otimes \mathbf{t}^{(2)} + \mathbf{e}_3 \otimes \mathbf{t}^{(3)} = \mathbf{e}_i \otimes \mathbf{t}^{(i)}$$

(5)

$$\mathbf{t}^{(n)} = \mathbf{n} \cdot \sigma$$

Note these are tensorial expressions (independent of the vector and tensor components in a particular coordinate system). To obtain the tensorial

Definition of stress tensor II

components in our rectangular system we replace the expressions of $\mathbf{t}^{(i)}$ from Eqn.(2)

Definition (Stress tensor representation in cartesian coordinate basis \mathbf{e}_i)

$$\boldsymbol{\sigma} = \mathbf{e}_i \underbrace{\sigma_{ij} \mathbf{e}_j}_{\mathbf{t}^{(i)}} = \sigma_{ij} \mathbf{e}_i \mathbf{e}_j \quad (6)$$

where

$$\sigma_{ij} = \begin{pmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{32} & \sigma_{33} \end{pmatrix}$$

are the components of the stress tensor $\boldsymbol{\sigma}$ in the cartesian coordinate system \mathbf{e}_i . Note that σ_{ij} represent the cartesian components of the stress vectors acting on the planes with normals \mathbf{e}_i , i.e. $\mathbf{t}^{(i)}$

Definition of stress tensor III

The cartesian components of the stress vector on the plane with normal \mathbf{n} can be obtained by noticing that:

$$\mathbf{t}^{(\mathbf{n})} = \mathbf{n} \cdot \sigma_{ij} \mathbf{e}_i \mathbf{e}_j = \sigma_{ij} (\mathbf{n} \cdot \mathbf{e}_i) \mathbf{e}_j = (\sigma_{ij} n_i) \mathbf{e}_j \quad (7)$$

$$t_j^{(\mathbf{n})} = t_j(\mathbf{n}) = \sigma_{ij} n_i \quad (8)$$

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