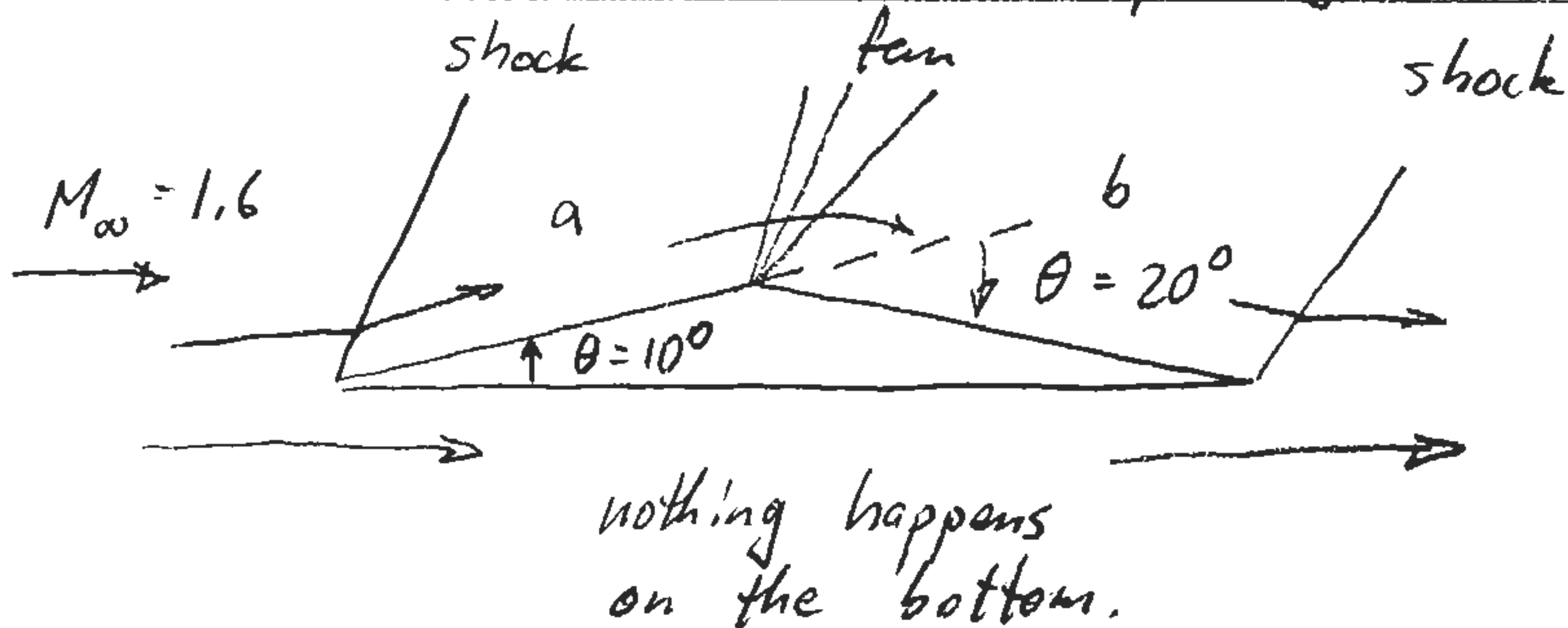


Flow sketch



a) Oblique Shock. $M_1 = 1.6, \theta = 10^\circ \rightarrow \beta = 51^\circ$ (chart, Anderson p. 513)

$$M_{n1} = M_1 \sin 51^\circ = 1.6 \cdot 0.777 = 1.243$$

$$P_2/P_1 = f(M_{n1}) = 1.636 \quad (\text{Anderson eq. 9.16, or Appendix B table.})$$

$$\text{Since } p_1 = p_\infty \rightarrow p_2 = \boxed{1.636 p_\infty = p_a}$$

$$M_{n2} = f(M_{n1}) = 0.817 \quad (\text{eq. 9.14})$$

$$\boxed{M_2 = M_a = \frac{M_{n2}}{\sin(51^\circ - 10^\circ)} = 1.245}$$

b) Expansion fan. $M_1 = M_a = 1.245, \nu(M_1) = 4.7^\circ$

$$\nu(M_2) = \nu(M_1) + 20^\circ = 24.7^\circ \rightarrow M_2 = 1.94$$

$$p_2 = p_{02} \left[1 + \frac{\gamma-1}{2} M_2^2 \right]^{-\frac{\gamma}{\gamma-1}}$$

$$p_{02} = p_{01} = p_{0a} \quad (\text{behind oblique shock.})$$

$$p_{0a}/p_{0\infty} = f(M_{n1}) = 0.987$$

$$p_{0a} = 0.987 p_{0\infty} = 0.987 p_\infty \left[1 + \frac{\gamma-1}{2} M_\infty^2 \right]^{\frac{\gamma}{\gamma-1}} = 4.195 p_\infty$$

$$\therefore p_2 = 4.195 p_\infty \left[1 + \frac{\gamma-1}{2} M_2^2 \right]^{-\frac{\gamma}{\gamma-1}} = \boxed{0.588 p_\infty = p_b}$$

Note: If we neglect the oblique shock's loss: $p_{0b} = p_{0a} = p_{0\infty} \rightarrow p_2 = 0.596 p_\infty$ (not quite correct)

$$c) L' = L'_a + L'_b = (p_\infty - p_a) \frac{c}{2} + (p_\infty - p_b) \frac{c}{2} = (1 - 1.636) p_\infty \frac{c}{2} + (1 - 0.588) p_\infty \frac{c}{2}$$

$$\boxed{L' = -0.112 p_\infty c}$$

$$D' = D'_a + D'_b = -L'_a \tan 10^\circ + L'_b \tan 10^\circ = [-(1 - 1.636) + (1 - 0.588)] \tan 10^\circ p_\infty \frac{c}{2}$$

$$\boxed{D' = 0.0924 p_\infty c}$$

$$\text{Using } \frac{1}{2} \rho_\infty V_\infty^2 = \frac{\gamma}{2} p_\infty M_\infty^2 = 1.792 p_\infty \rightarrow$$

$$\boxed{C_L = L' / \frac{1}{2} \rho_\infty V_\infty^2 = -0.0625}$$

$$\boxed{C_D = D' / \frac{1}{2} \rho_\infty V_\infty^2 = 0.0516}$$

Flow out of tank through air hose:

$T_0 = 300\text{K}^\circ$
 $P_0 = 3.45 \times 10^5 \text{ Pa}$

smallest $A = A^*$ at sonic conditions

Required $\dot{m} = 0.01 \text{ kg/s} = \rho^* a^* A^*$

$$\rho^* = \rho_0 \left[1 + \frac{\gamma-1}{2} \right]^{-\frac{1}{\gamma-1}} = 0.634 \rho_0$$

but $h_0 = c_p T_0 = 1004 \text{ J/kg}^\circ\text{K} \cdot 300^\circ = 301200 \text{ m}^2/\text{s}^2$

$$a_0 = \sqrt{(\gamma-1)h_0} = 347.1 \text{ m/s}$$

$$\rho_0 = \gamma P_0 / (\gamma-1)h_0 = 4.01 \text{ kg/m}^3$$

So $\rho^* = 0.634 \rho_0 = 2.542 \text{ kg/m}^3$

$$a^* = a_0 \left[1 + \frac{\gamma-1}{2} \right]^{-\frac{1}{2}} = 316.8 \text{ m/s}$$

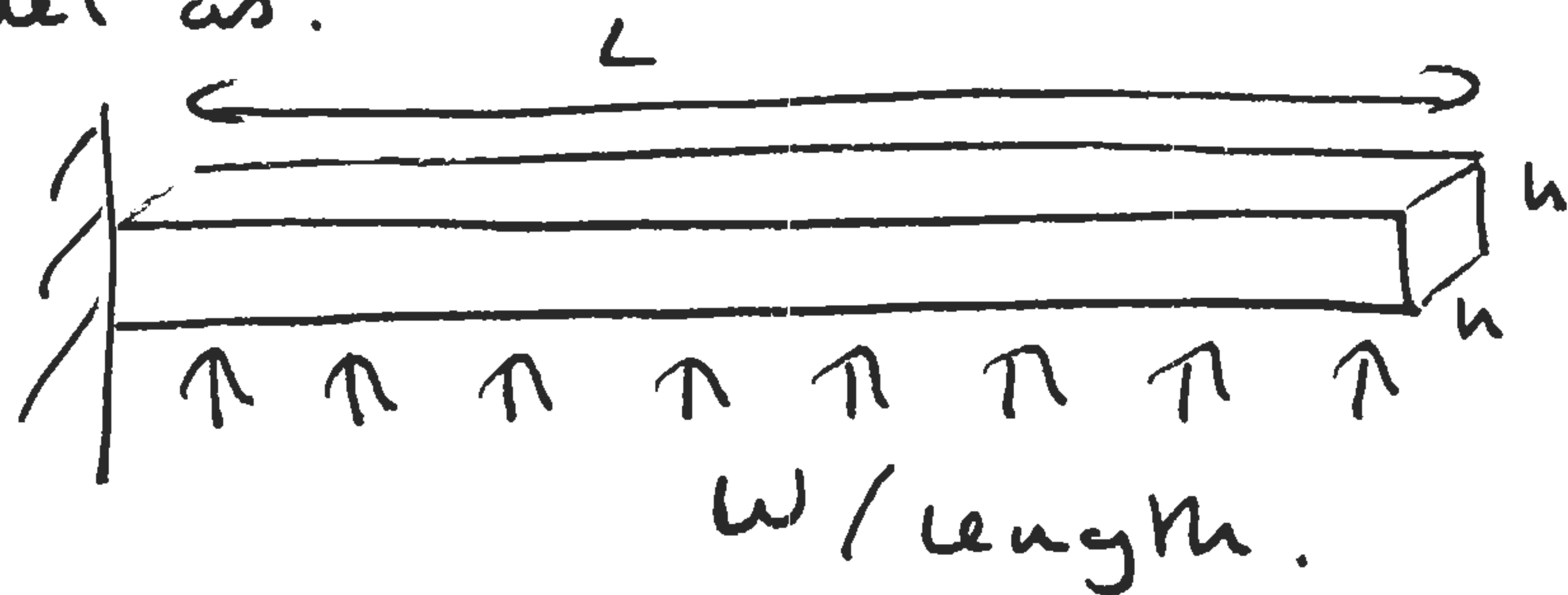
$$A^* = \frac{\dot{m}}{\rho^* a^*} = \frac{0.01}{2.542 \cdot 316.8} = 1.24 \times 10^{-5} \text{ m}^2 = 0.124 \text{ cm}^2$$

$$r = 2 \text{ mm}$$

$$\text{diameter} = 4 \text{ mm} = 0.156 \text{ in.}$$

M19

Wing, designed for bending loads
Model as:



$$I = \frac{1}{12} h^4$$

$$\text{Max moment at root} = \frac{wL^2}{2}$$

$$\begin{aligned} \text{Max stress} &= \frac{Mz}{I} = \sigma_{\text{max}} = \frac{wL^2}{2} \cdot \frac{h}{2} \cdot \frac{12}{h^4} \\ &= \frac{3wL^2}{h^3} \quad \text{cannot exceed } \sigma_y \end{aligned}$$

$$\text{mass} = \rho AL = \rho h^2 L$$

$$h = \sqrt{\frac{m}{\rho L}}$$

substitute for h

$$\sigma_y = 3wL^2 \left(\frac{\rho L}{m} \right)^{3/2}$$

$$\therefore \text{minimum mass } m = (3wL^2)^{2/3} \frac{\rho L}{\sigma_y^{2/3}}$$

\therefore choose material with min

$$\text{or max } \sigma_y^{2/3} / \rho. \Leftarrow$$

For turbine disk $\sigma_{max} < \sigma_y$

$$\therefore \frac{3}{8} \rho v^2 = \sigma_y$$

choose material with max σ_y/ρ

	ρ	σ_y	σ_y/ρ	$\sigma_y^{2/3}/\rho$
6) Al 2024	2800	345	0.12	0.018
Al 7075	2800	495	0.18	0.022
Ti 6-4	4510	910	0.20	0.021
Ph 17-7 Ph.	8000	1435	0.18	0.016
steel	7800	260	0.03	0.005
			↓	↓
			<u>Ti</u>	<u>Al 7075</u>

Critical crack size for

$$\text{Al 7075} \quad a_c = \frac{1}{\pi} \left(\frac{K_{IC}}{0.4 \sigma_y} \right)^2 = \frac{1}{\pi} \left(\frac{24 \times 10^6}{0.4 \times 495 \times 10^6} \right)^2$$

$$a_c = 4.7 \text{ mm.} \quad - \text{detectable}$$

$$\text{Ti 6-4} \quad a_c = \frac{1}{\pi} \left(\frac{50 \times 10^6}{0.9 \times 910 \times 10^6} \right)^2 = 1.2 \text{ mm}$$

|
Small

d) Critical crack size for Ti 6-4 is small, harder to detect, more difficult to implement a damage tolerant design approach.

M20

$$\begin{aligned}\frac{da}{dN} &= 2.7 \times 10^{-12} (\Delta K)^{5.0} \\ &= 2.7 \times 10^{-12} (\gamma \Delta \sigma \sqrt{\pi a})^{5.0} \\ &= 2.7 \times 10^{-12} \gamma^5 \Delta \sigma^5 \pi^{2.5} a^{2.5}\end{aligned}$$

$$a_{cr.f} = \frac{1}{\pi} \left(\frac{K_{Ic}}{\sigma_{max}} \right)^2 \quad K_{Ic} = 44 \text{ MPa} \sqrt{\text{m}}$$

$$\sigma_{max} = \sigma_{mean} + 100 \text{ MPa} = 250 \text{ MPa}$$

$$a_{cr.f} = \frac{1}{\pi} \left(\frac{44}{250} \right)^2 = 9.9 \text{ mm}$$

$$N_f = \int_{a_0}^{a_f} \frac{da}{2.7 \times 10^{-12} \gamma^5 \Delta \sigma^5 \pi^{2.5} a^{2.5}}$$

$$N_f = \frac{1}{2.7 \times 10^{-12} (1.2)^5 (200)^5 \pi^{2.5}} \left[\frac{a^{-1.5}}{1.5} \right]_{9.9 \times 10^{-3}}^{3.0 \times 10^{-3}}$$

$$N_f = 89.88 \text{ cycles}$$

Inspect at more frequent intervals \rightarrow Maybe every 45 cycles