## 16.06 Principles of Automatic Control Lecture 26

From last time, we had plant and compensator

$$G(s) = \frac{1}{(1+s/0.5)(1+s)(1+s/2)}$$
$$K(s) = 9\frac{1+s}{1+s/8}$$

The closed-loop step response has 45% overshoot, when 37% expected. Why? Look at Bode plot of  $H = \frac{KG}{(1+KG)}$ :



Because of low  $K_p$ , D.C. gain of H is 0.9, which increases effective  $M_r$  by factor of 1/0.9.

## Lag Compensator

Consider the plant

$$G(s) = \frac{1}{s(s+10)}$$

In a unity feedback control system



Suppose we use a proportional controller

$$K(s) = 141$$

For this controller,

$$\omega_c = 10 \,\mathrm{r/s}$$
  
PM =45°

and the overshoot in response to a unit step is

$$M_p = 23\%$$

Suppose that we find the response of the closed-loop system satisfactory, except that the velocity constant  $K_v = 14.1$  is lower than desired ( $K_v = 100$ ). How might we improve the response?

Look at Bode plot:



Placing  $K_v$  constrain on Bode plot shows that we must somehow make slope steeper for a bit to achieve the requirement, if we want crossover behavior to be similar. We do this with a *lag* compensator:



On order to achieve our design goals, we need the lag ratio a/b to be the amount of additional low frequency gain required. In our case,

$$\frac{a}{b} = \frac{100}{14.1} = 7.1$$

We also need

 $a \ll \omega_c$ 

So that not too much phase lag is added at crossover. It's common to use

$$a = \omega_c/10$$

which ensures  $< 6^{\circ}$  of phase lag will be added at crossover.

So the new Compensator is

$$K(s) = 141 \, \frac{s+1}{s+0.14}$$

How well does the new compensator work? Compare step responses, error response to ramp inputs (see plots).





Note that although the steady-state error to a ramp input is reduced, there is a long tail to the response. Why? Look at Root locus:



Note the constant pole near lag zero. Pole has small residue, but a long time constant.

time constant.

This behavior is very typical of systems with lag of PI control. To eliminate, must increase bandwidth (crossover frequency), which is not always desirable.

## PI Control

PI (proportional-integral control) is used when the type of the system must be increased, say, from type 0 to type 1.

**Example:** Consider a system that performs adequately with unity feedback



where

$$G(s) = 100 \frac{1}{(1+s/1)(1+s/200)},$$

but we desire a type 1 system with velocity constant  $K_v = 100$ . Look at problem on Bode plot:



So the compensator is

$$K(s) = \frac{3}{s} + 1 = \frac{s+3}{s}$$

Note that error pole will be near s = -3. To speed up error response, use

$$K(s) = \frac{s+10}{s} \qquad (\Rightarrow K_v = 1000)$$

which will result in pole near s = -10.

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