

Application Tasks

6. (a) Let's start with the 3-D orthotropic stress-strain relations:

$$\begin{pmatrix} \epsilon_{11} \\ \epsilon_{22} \\ \epsilon_{33} \\ \epsilon_{23} \\ \epsilon_{13} \\ \epsilon_{12} \end{pmatrix} = \begin{bmatrix} 1/E_1 & -\nu_{12}/E_1 & -\nu_{13}/E_1 & 0 & 0 & 0 \\ -\nu_{12}/E_1 & 1/E_2 & -\nu_{23}/E_2 & 0 & 0 & 0 \\ -\nu_{13}/E_1 & -\nu_{23}/E_2 & 1/E_3 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1/2G_{23} & 0 & 0 \\ 0 & 0 & 0 & 0 & 1/2G_{13} & 0 \\ 0 & 0 & 0 & 0 & 0 & 1/2G_{12} \end{bmatrix} \begin{pmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{23} \\ \sigma_{13} \\ \sigma_{12} \end{pmatrix}$$

— (1)

For a plane stress state:

$$\sigma_{33} = \sigma_{13} = \sigma_{23} = 0 \quad (2)$$

Using (2) in (1) gives:

$$\epsilon_{11} = 1/E_1 \sigma_{11} - \nu_{12}/E_1 \sigma_{22} \quad (3)$$

$$\epsilon_{22} = -\nu_{12}/E_1 \sigma_{11} + 1/E_2 \sigma_{22} \quad (4)$$

$$\epsilon_{33} = -\nu_{13}/E_1 \sigma_{11} - \nu_{23}/E_2 \sigma_{22} \quad (5)$$

$$\epsilon_{23} = 0 \quad (6)$$

$$\epsilon_{13} = 0 \quad (7)$$

$$\epsilon_{12} = 1/2G_{12} \sigma_{12} \quad (8)$$

(2)

The plane stress elasticity tensor can be written as:

$$\sigma_{\alpha\beta} = E_{\alpha\beta\gamma\delta} \epsilon_{\gamma\delta} \quad (9)$$

Since the Greek subscripts indicate 1, 2, equation (9) relates  $\sigma_{11}$ ,  $\sigma_{22}$  and  $\sigma_{12}$  to  $\epsilon_{11}$ ,  $\epsilon_{22}$  and  $\epsilon_{12}$ . Thus, we only need to consider equations (3), (4), and (8). In matrix form, these equations can be expressed as:

$$\begin{Bmatrix} \epsilon_{11} \\ \epsilon_{22} \\ \epsilon_{12} \end{Bmatrix} = \begin{bmatrix} 1/E_1 & -\nu_{12}/E_1 & 0 \\ -\nu_{21}/E_1 & 1/E_2 & 0 \\ 0 & 0 & 1/2G_{12} \end{bmatrix} \begin{Bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{Bmatrix} \quad (10)$$

\*Note: This is the matrix of the plane stress compliance tensor. The inverse of this is the elasticity tensor.

Thus, from equation (10), we need to know:

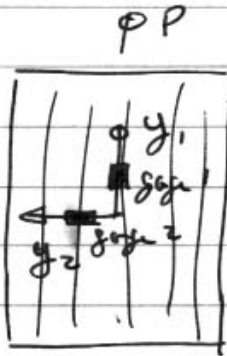
$$\boxed{E_1 \quad E_2 \quad \nu_{12} \quad G_{12}}$$

(b) From part (a), there are four material properties that need to be determined. They are determined from the two tests using the stress-strain behavior,

(3)

assuming that is within the linear region.

Test A



Applied Load/Stress

$$\sigma_{11} = 450 \text{ MPa}$$

$$\sigma_{22} = 0$$

$$\sigma_{12} = 0$$



Strain readings

$$\text{(Gauge 1)} \quad \epsilon_{11} = 2600 \times 10^{-6}$$

$$\text{(Gauge 2)} \quad \epsilon_{22} = -850 \times 10^{-6}$$

Using the known stress and strains in equation (10) gives:

$$\epsilon_{11} = \frac{1}{E_1} (450 \text{ MPa}) = 2600 \times 10^{-6}$$

$$\Rightarrow E_1 = \frac{450 \times 10^6 \text{ Pa}}{2600 \times 10^{-6}}$$

$$\Rightarrow E_1 = 173 \times 10^9 \text{ Pa}$$

$$\Rightarrow \boxed{E_1 = 173 \text{ GPa}}$$

$$\epsilon_{22} = -\frac{\nu_{12}}{E_1} (450 \text{ MPa}) = -850 \times 10^{-6}$$

$$\Rightarrow \nu_{12} = 850 \times 10^{-6} \times \frac{173 \times 10^9 \text{ Pa}}{450 \times 10^6 \text{ Pa}}$$

$$\Rightarrow \boxed{\nu_{12} = 0.327}$$

(4)

\*Note: Since the load/stress in this case is applied along the material principal axis,  $E_1$  and  $\nu_{12}$  can be obtained directly from definitions:

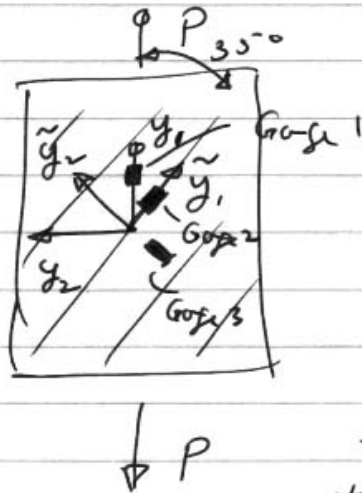
→ Definition of  $\nu_{12}$ :  $\nu_{12} = -\frac{\epsilon_{22}}{\epsilon_{11}}$  (for  $\sigma_{11}$  applied only)

$\Rightarrow \nu_{12} = \frac{-(-850 \times 10^{-6})}{2600 \times 10^{-6}} = 0.327 \checkmark$

→ Definition of  $E_1$ :  $E_1 = \frac{\sigma_{11}}{\epsilon_{11}}$  (for  $\sigma_{11}$  applied only)

$\Rightarrow E_1 = \frac{450 \times 10^6 \text{ Pa}}{2600 \times 10^{-6}} = 173 \text{ GPa} \checkmark$

Test B



We need to setup two sets of axes. Place one aligned with the applied load. Place the other aligned with the fibers (and other two strain gauges). We will need

to do transformations between the two.

(5)

With these axis systems, we summarize what we know:

$$\text{Applied Load/stress: } \sigma_{11} = 200 \text{ MPa}$$

$$\sigma_{22} = 0$$

$$\sigma_{12} = 0$$

$$\text{measured strains: } \epsilon_{11} = 11900 \times 10^{-6} \text{ (gage 1)}$$

$$\text{(gage 2)} \quad \tilde{\epsilon}_{11} = 650 \times 10^{-6}$$

$$\text{(gage 3)} \quad \tilde{\epsilon}_{22} = 10100 \times 10^{-6}$$

Now we need to transform  $\epsilon$  and  $\sigma$  in the loading axis to  $\tilde{\epsilon}$  and  $\tilde{\sigma}$  in the material axis in order to find the unknown quantities (or moduli):

$$\epsilon_{22}, \epsilon_{12}, \tilde{\sigma}_{11}, \tilde{\sigma}_{22}, \tilde{\sigma}_{12}, \tilde{\epsilon}_{12}$$

and to use the stress-strain relations in material axes (as expressed in equation (10)) and rewritten here for the "transformed axis" case:

$$\begin{Bmatrix} \tilde{\epsilon}_{11} \\ \tilde{\epsilon}_{22} \\ \tilde{\epsilon}_{12} \end{Bmatrix} = \begin{bmatrix} 1/E_1 & -\nu_{12}/E_1 & 0 \\ -\nu_{12}/E_1 & 1/E_2 & 0 \\ 0 & 0 & 1/2G_{12} \end{bmatrix} \begin{Bmatrix} \tilde{\sigma}_{11} \\ \tilde{\sigma}_{22} \\ \tilde{\sigma}_{12} \end{Bmatrix} \quad (11)$$

The second order transformation are:

$$\tilde{\epsilon}_{\alpha\beta} = l_{\alpha 20} l_{\beta 20} \epsilon_{00} \quad ; \quad \tilde{\sigma}_{\alpha\beta} = l_{\alpha 20} l_{\beta 20} \sigma_{00}$$

(6)

And can be written in matrix form as:

$$\begin{Bmatrix} \tilde{\epsilon}_{11} \\ \tilde{\epsilon}_{22} \\ \tilde{\epsilon}_{12} \end{Bmatrix} = \begin{bmatrix} \cos^2 \theta & \sin^2 \theta & 2 \sin \theta \cos \theta \\ \sin^2 \theta & \cos^2 \theta & -2 \sin \theta \cos \theta \\ -\sin \theta \cos \theta & \sin \theta \cos \theta & \cos^2 \theta - \sin^2 \theta \end{bmatrix} \begin{Bmatrix} \epsilon_{11} \\ \epsilon_{22} \\ \epsilon_{12} \end{Bmatrix} \quad (12)$$

$$\begin{Bmatrix} \tilde{\sigma}_{11} \\ \tilde{\sigma}_{22} \\ \tilde{\sigma}_{12} \end{Bmatrix} = \begin{bmatrix} \cos^2 \theta & \sin^2 \theta & 2 \sin \theta \cos \theta \\ \sin^2 \theta & \cos^2 \theta & -2 \sin \theta \cos \theta \\ -\sin \theta \cos \theta & \sin \theta \cos \theta & \cos^2 \theta - \sin^2 \theta \end{bmatrix} \begin{Bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{Bmatrix} \quad (13)$$

Noting that our transformation from  $\epsilon_a$  to  $\tilde{\epsilon}_a$  is clockwise, we use  $\theta = -35^\circ$ , so we get:

$$\begin{Bmatrix} \tilde{\sigma}_{11} \\ \tilde{\sigma}_{22} \\ \tilde{\sigma}_{12} \end{Bmatrix} = \begin{bmatrix} 0.671 & 0.329 & 0.940 \\ 0.329 & 0.671 & -0.940 \\ -0.470 & 0.470 & 0.342 \end{bmatrix} \begin{Bmatrix} 200 \text{ MPa} \\ 0 \\ 0 \end{Bmatrix}$$

$$\Rightarrow \tilde{\sigma}_{11} = 134.2 \text{ MPa} \quad (14a)$$

$$\tilde{\sigma}_{22} = 65.8 \text{ MPa} \quad (14b)$$

$$\tilde{\sigma}_{12} = -94.0 \text{ MPa} \quad (14c)$$

Similarly for the strain using equation (12):

$$\begin{Bmatrix} 650 \times 10^{-6} \\ 10100 \times 10^{-6} \\ \epsilon_{12} \end{Bmatrix} = \begin{bmatrix} 0.671 & 0.329 & 0.940 \\ 0.329 & 0.671 & -0.940 \\ -0.470 & 0.470 & 0.342 \end{bmatrix} \begin{Bmatrix} 11900 \times 10^{-6} \\ 0 \\ 0 \end{Bmatrix} \quad (15)$$

(7)

$$\Rightarrow 650 \times 10^{-6} = 7985 \times 10^{-6} + 0.329 \epsilon_{22} + 0.94 \epsilon_{12} \quad (16)$$

$$10,000 \times 10^{-6} = 3915 \times 10^{-6} + 0.671 \epsilon_{22} - 0.94 \epsilon_{12} \quad (17)$$

$$\tilde{\epsilon}_{12} = -5543 \times 10^{-6} + 0.47 \epsilon_{22} + 0.342 \epsilon_{12} \quad (18)$$

Solving for  $\epsilon_{22}$ ,  $\epsilon_{12}$  and  $\tilde{\epsilon}_{12}$  via equations (16), (17), and (18), we first add equation (16) and (17):

$$10750 \times 10^{-6} = 12000 \times 10^{-6} + \epsilon_{22}$$

$$\Rightarrow \epsilon_{22} = -1250 \times 10^{-6} \quad (19a)$$

(NOTE: This is the same as using the invariant sum  $\epsilon_{11} + \epsilon_{22}$  [see Problem 23])

Use this result in (16) to get:

$$650 \times 10^{-6} = 7985 \times 10^{-6} - 411 \times 10^{-6} + 0.94 \epsilon_{12}$$

$$\Rightarrow 0.94 \epsilon_{12} = -6924 \times 10^{-6}$$

$$\Rightarrow \epsilon_{12} = -7366 \times 10^{-6} \quad (19b)$$

And finally in (18):

$$\tilde{\epsilon}_{12} = -5543 \times 10^{-6} + 0.47(-1250 \times 10^{-6}) + 0.342(-7366 \times 10^{-6})$$

$$\Rightarrow \tilde{\epsilon}_{12} = -8650 \times 10^{-6} \quad (19c)$$

Now use the results of (19a-c) with the stress-strain equations in transformed axes (1) to get:

(8)

$$\tilde{\epsilon}_{11} = 650 \times 10^{-6} = \frac{1}{E_1} (134.2 \text{ MPa}) - \frac{\nu_{12}}{E_1} (65.8 \text{ MPa}) \quad (2)$$

We already know  $E_1$  (173 GPa) and  $\nu_{12}$  (0.327),  
so we check our results via this equation:

$$\frac{134.2 \text{ MPa}}{173 \text{ GPa}} - \frac{0.327}{173 \text{ GPa}} (65.8 \text{ MPa}) \stackrel{?}{=} 650 \times 10^{-6}$$

$$651 \times 10^{-6} \stackrel{?}{=} 650 \times 10^{-6} \quad \checkmark$$

Good

Note: Not exact but we expect some difference due to difference in tubing. Actually,  $1 \times 10^{-6}$  is extremely small for experimental deviation.

The next two parts of the matrix equation give:

$$\tilde{\epsilon}_{22} = 10100 \times 10^{-6} = -\frac{\nu_{12}}{E_1} (134.2 \text{ MPa}) + \frac{1}{E_2} (65.8 \text{ MPa})$$

$$\Rightarrow E_2 = \frac{65.8 \text{ MPa}}{10100 \times 10^{-6} + \frac{0.327}{173 \text{ GPa}} (134.2 \text{ MPa})}$$

$$\Rightarrow \boxed{E_2 = 6.36 \text{ GPa}}$$

And:

$$\tilde{\epsilon}_{12} = \frac{1}{2G_{12}} (-94.0 \text{ MPa}) = -8650 \times 10^{-4}$$

$$\Rightarrow \boxed{G_{12} = 5.43 \text{ GPa}}$$



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Summarizing:

$$E_1 = 173 \text{ GPa}$$

$$E_2 = 6.36 \text{ GPa}$$

$$\nu_{12} = 0.327$$

$$G_{12} = 5.43 \text{ GPa}$$

(c) Note that we used the transformation equation for  $\tilde{E}_{11}$ , not to obtain an elastic constant but to check if the relationship held. This shows that we do have more strain gauges than needed and we can therefore eliminate one gauge. However, we cannot choose any gauge. We can eliminate Gauge 2 in Test A since  $\nu_{12}$  can be obtained from Test B via the  $\tilde{E}_{11}$  equation (equation (20)). We cannot, however, eliminate any of the gauges in Test B as we would not have enough data to obtain  $\tilde{E}_{12}$  and thus  $G_{12}$ .