

Problem Set No. 6Problem 1

(a)

generalized coordinates for the rolling disk:

$$q_1 = x_p, \quad q_2 = y_p, \quad q_3 = \alpha, \quad q_4 = \varphi$$

$$\text{rolling constraints: } (1) \begin{cases} \dot{x}_p - R\dot{\varphi} \sin\alpha = 0 & \rightarrow a_{11} = 1, a_{12} = 0, a_{13} = 0, a_{14} = -R \sin\alpha \\ (2) \begin{cases} \dot{y}_p - R\dot{\varphi} \cos\alpha = 0 & \rightarrow a_{21} = 0, a_{22} = 1, a_{23} = 0, a_{24} = -R \cos\alpha \end{cases} \end{cases}$$

$$\left. \begin{aligned} \frac{\partial a_{14}}{\partial q_3} = -R \cos\alpha \neq \frac{\partial a_{13}}{\partial q_4} = 0 \\ \frac{\partial a_{24}}{\partial q_3} = R \sin\alpha \neq \frac{\partial a_{23}}{\partial q_4} = 0 \end{aligned} \right\} \Rightarrow (1) \& (2) \text{ are not integrable.}$$

→ Constraints are nonholonomic.

(b) Assume that there are integrating factors: $c_1(q_i, t), c_2(q_i, t), i=1, \dots, 4$

$$\Rightarrow \tilde{a}_{11} = c_1, \quad \tilde{a}_{12} = 0, \quad \tilde{a}_{13} = 0, \quad \tilde{a}_{14} = -R c_1 \sin\alpha$$

$$\tilde{a}_{21} = 0, \quad \tilde{a}_{22} = c_2, \quad \tilde{a}_{23} = 0, \quad \tilde{a}_{24} = -R c_2 \cos\alpha$$

necessary conditions for the constraints to be integrable:

$$\frac{\partial \tilde{a}_{12}}{\partial q_i} = 0 \rightarrow \frac{\partial \tilde{a}_{1i}}{\partial q_2} = 0 \rightarrow \frac{\partial c_1}{\partial y_p} = 0$$

$$\frac{\partial \tilde{a}_{13}}{\partial q_i} = 0 \rightarrow \frac{\partial \tilde{a}_{1i}}{\partial q_3} = 0 \rightarrow \frac{\partial c_1}{\partial \alpha} = 0 \quad \& \quad -R \sin\alpha \frac{\partial c_1}{\partial \alpha} - R c_1 \cos\alpha = 0 \rightarrow \underline{c_1 = 0}$$

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$$\frac{\partial \tilde{a}_{23}}{\partial q_i} = 0 \quad \rightarrow \quad \frac{\partial \tilde{a}_{2i}}{\partial q_3} = 0 \quad \rightarrow \quad \frac{\partial C_2}{\partial \alpha} = 0 \quad \& \quad -R \cos \alpha \frac{\partial C_2}{\partial \alpha} + R C_2 \sin \alpha = 0$$

$$\rightarrow \quad \underline{C_2 = 0}$$

\Rightarrow no non trivial integrating factors exist.

two rolling constraints cannot be integrated to holonomic constraints.

generalized coordinates assuming

vehicle B does not rotate:

$$q_1 = x_A \quad q_2 = y_A \quad q_3 = \theta$$

$$q_4 = x_B \quad q_5 = y_B$$

Constraint: The orientation of vehicle A must always be toward vehicle B:

$$\underline{v}_A \times \underline{r}_{AB} = \underline{0}$$

$$(\dot{x}_A \underline{i} + \dot{y}_A \underline{j}) \times (x_B \underline{i} + y_B \underline{j}) = \underline{0}$$

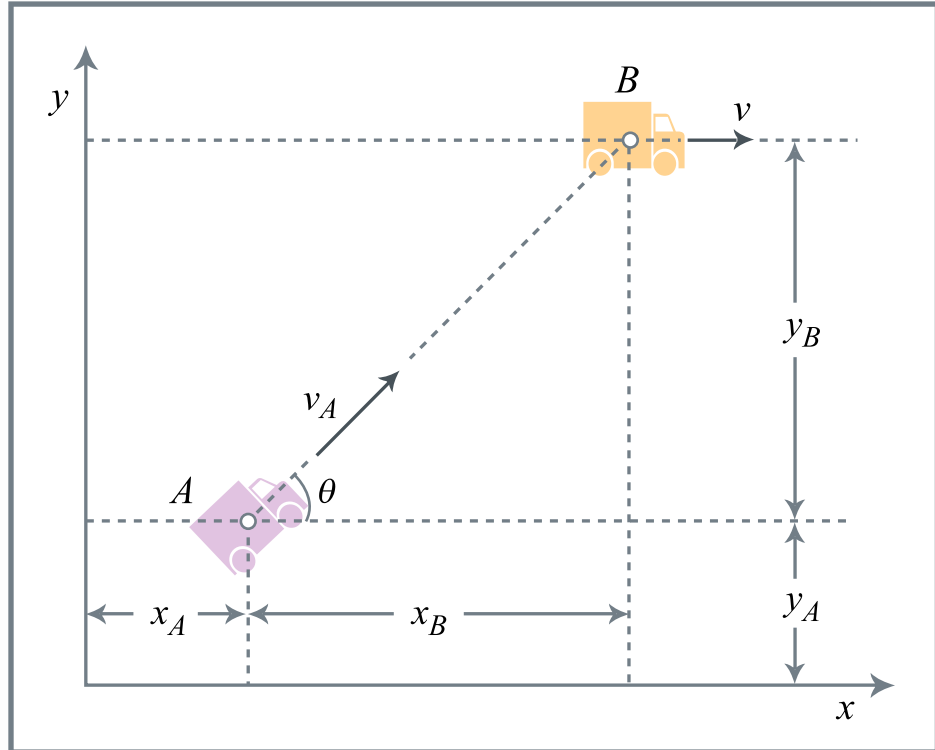


Figure by OCW. After 4.3/4 in Baruh, H. Analytical Dynamics. Boston MA: McGraw-Hill, 1999.

$$\dot{x}_A y_B - \dot{y}_A x_B = 0 \quad \text{nonholonomic scleronomic constraint}$$

check the integrability:

$$a_{11} = y_B, \quad a_{12} = -x_B, \quad a_{13} = a_{14} = a_{15} = 0$$

$$\frac{\partial a_{11}}{\partial q_5} = 1 \neq \frac{\partial a_{15}}{\partial q_1} = 0 \quad \rightarrow \quad \text{constraint is not integrable.}$$

If we allow for an integrating factor: $C(q_i, t)$, $i=1, \dots, 5$

$$\rightarrow \quad \tilde{a}_{11} = C y_B, \quad \tilde{a}_{12} = -C x_B, \quad \tilde{a}_{13} = \tilde{a}_{14} = \tilde{a}_{15} = 0$$

$$\frac{\partial \tilde{a}_{14}}{\partial q_i} = 0 \quad \rightarrow \quad \frac{\partial \tilde{a}_{1i}}{\partial q_4} = 0 \quad \rightarrow \quad \frac{\partial C}{\partial x_B} = 0 \quad \& \quad -\frac{\partial C}{\partial x_B} x_B - C = 0 \quad \rightarrow \quad \underline{C = 0}$$

\Rightarrow constraint is truly nonholonomic.

Problem 3

For point P:

$$\begin{cases} x = (L_1 + L_2 \sin \theta) \cos \varphi \\ y = (L_1 + L_2 \sin \theta) \sin \varphi \\ z = -L_2 \cos \theta \end{cases}$$

Virtual displacement:

$$\begin{cases} \delta x = L_2 \cos \theta \cos \varphi \delta \theta - (L_1 + L_2 \sin \theta) \sin \varphi \delta \varphi \\ \delta y = L_2 \cos \theta \sin \varphi \delta \theta + (L_1 + L_2 \sin \theta) \cos \varphi \delta \varphi \\ \delta z = L_2 \sin \theta \delta \theta \end{cases}$$

In the case of constant $\dot{\varphi}$ ($\delta \varphi = 0$):

$$\begin{aligned} x^2 + y^2 + z^2 &= OP^2 = L_1^2 + L_2^2 - 2L_1L_2 \cos(90 + \theta) \\ &= L_1^2 + L_2^2 + 2L_1L_2 \sin \theta \end{aligned}$$

virtual displacement:

$$2x \delta x + 2y \delta y + 2z \delta z = 2L_1L_2 \cos \theta \delta \theta$$

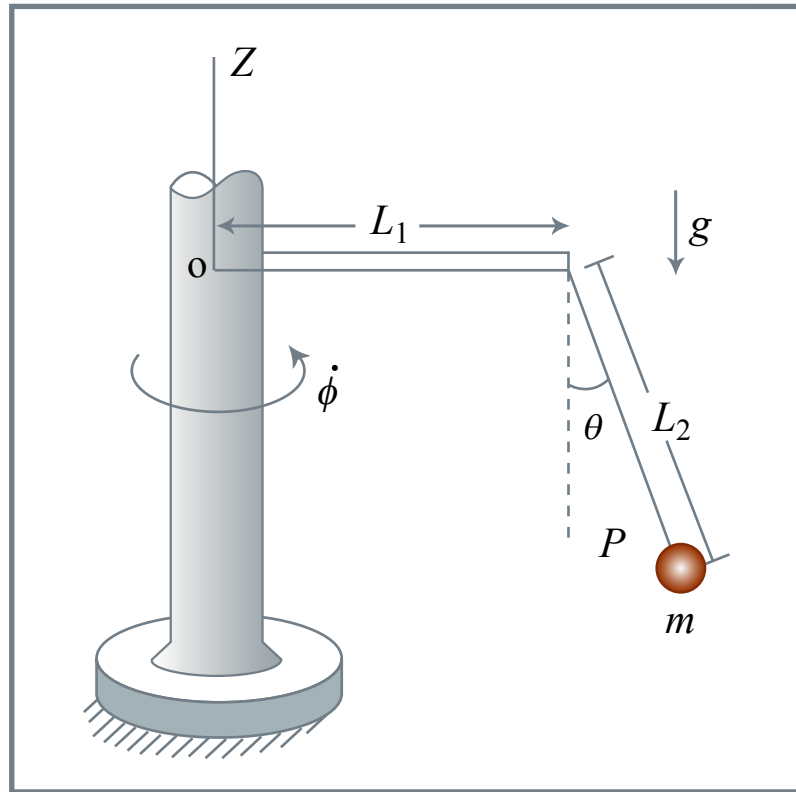
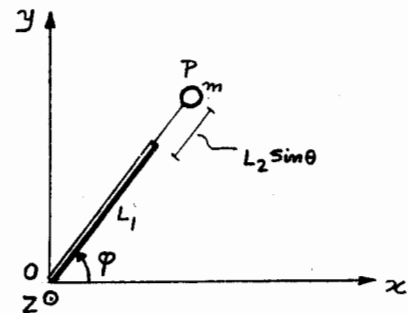
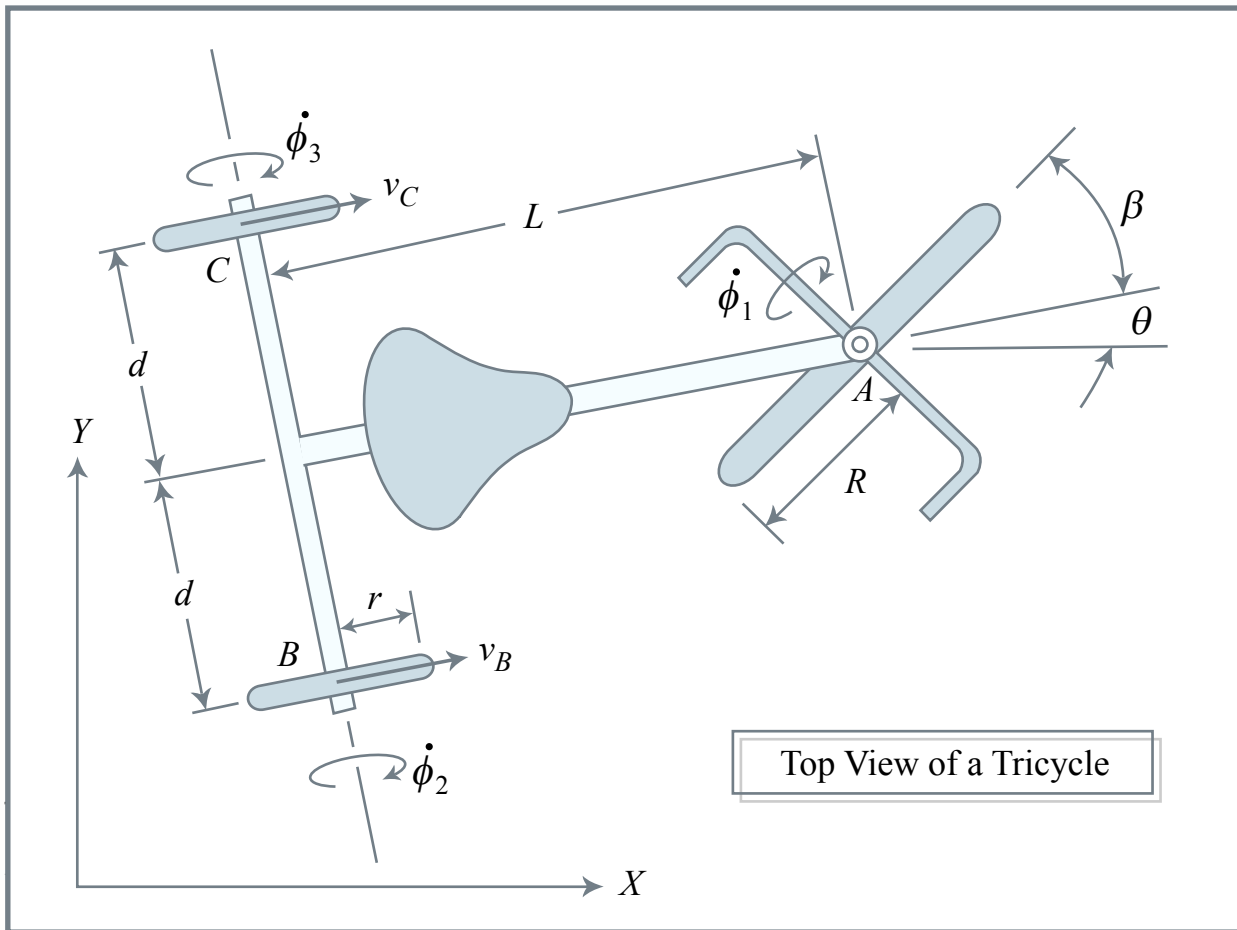


Figure by OCW.

After 4.4/7 in Baruh, H. *Analytical Dynamics*. Boston MA: McGraw-Hill, 1999.



Top View



Top View of a Tricycle

Figure by OCW. After 6.8 in Ginsberg, J. H. *Advanced Engineering Dynamics*. 2nd ed. New York: Cambridge University Press, 1998. wheels do not slip:

$$\begin{cases} v_A = R\dot{\phi}_1 \\ v_B = r\dot{\phi}_2 \\ v_C = r\dot{\phi}_3 \end{cases}$$

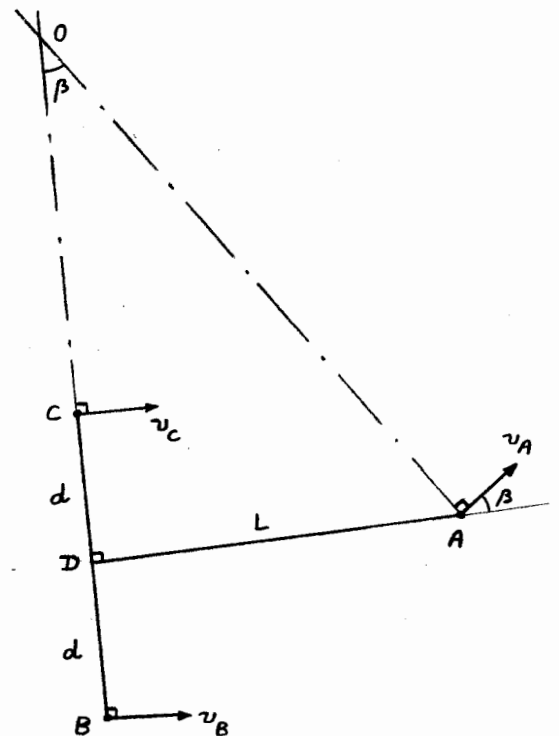
Points A, B, and C are on the frame as well.

One can find the instantaneous axis of rotation for the frame:

$$\omega_{frame} = \dot{\theta} \mathbf{k}$$

$$OA = \frac{L}{\sin\beta}, \quad OC = \frac{L}{\tan\beta} - d, \quad OB = \frac{L}{\tan\beta} + d$$

$$v_A = OA \cdot \dot{\theta} \quad v_C = OC \cdot \dot{\theta} \quad v_B = OB \cdot \dot{\theta}$$



Problem 4

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$$v_A = R\dot{\varphi}_1 = \frac{L}{\sin\beta} \dot{\theta} \quad \rightarrow \quad \underline{L\dot{\theta} = R\dot{\varphi}_1 \sin\beta}$$

$$\left. \begin{aligned} v_B = r\dot{\varphi}_2 &= \left(\frac{L}{\tan\beta} + d\right) \dot{\theta} \\ v_C = r\dot{\varphi}_3 &= \left(\frac{L}{\tan\beta} - d\right) \dot{\theta} \end{aligned} \right\} \Rightarrow \begin{aligned} \underline{r(\dot{\varphi}_2 - \dot{\varphi}_3)} &= 2d\dot{\theta} \\ r(\dot{\varphi}_2 + \dot{\varphi}_3) &= \frac{2L\dot{\theta}}{\tan\beta} = \frac{2R\dot{\varphi}_1 \sin\beta}{\tan\beta} = 2R\dot{\varphi}_1 \cos\beta \end{aligned}$$

$$\rightarrow \underline{r(\dot{\varphi}_2 + \dot{\varphi}_3) = 2R\dot{\varphi}_1 \cos\beta}$$

velocity constraints

To find the number of degrees of freedom:

Each wheel is like a vertical disk rolling without slip on a 2-D plane so it has 2 degrees of freedom. We have the following constraints:

- 1- $AB = \text{Const.}$
- 2- $AC = \text{Const.}$
- 3- $BC = \text{Const.}$

4- wheel B has to be perpendicular to the shaft BC. Satisfying the no slip constraint forces the wheel C to be perpendicular to the shaft BC as well.

$$\therefore \# \text{ DOF} = 3 \times 2 - 4 = 2$$

Note that we need 4 generalized coordinate to define the system.