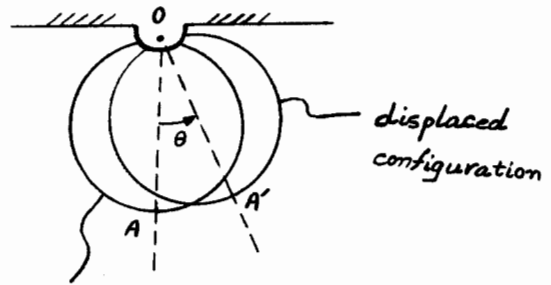


Problem 1

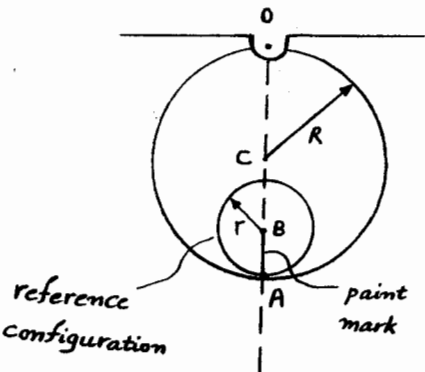
Practice Problems No. 1

Comparing the orientation of OA to OA' , the angular velocity of the ring would be:

$$\underline{\omega}_{ring} = \dot{\theta} \hat{e}_z$$

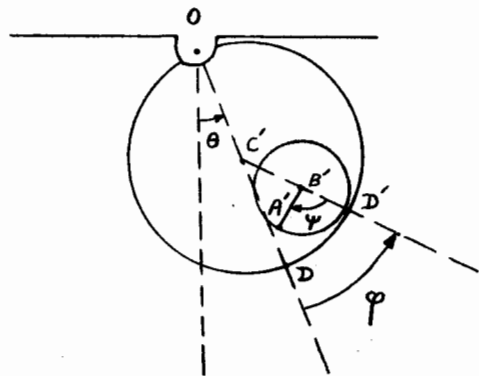


reference configuration



reference configuration

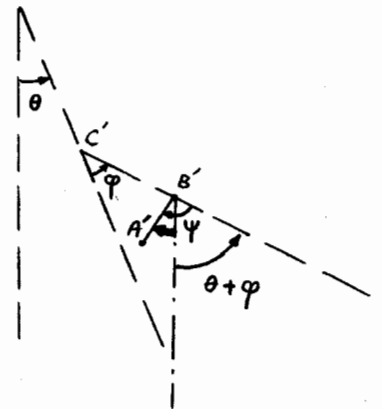
paint mark



displaced configuration

Comparing BA to $B'A'$,

$$\begin{aligned} \underline{\omega}_{disk} &= -[\dot{\psi} - (\dot{\theta} + \dot{\phi})] \hat{e}_z \\ &= (\dot{\theta} + \dot{\phi} - \dot{\psi}) \hat{e}_z \end{aligned}$$

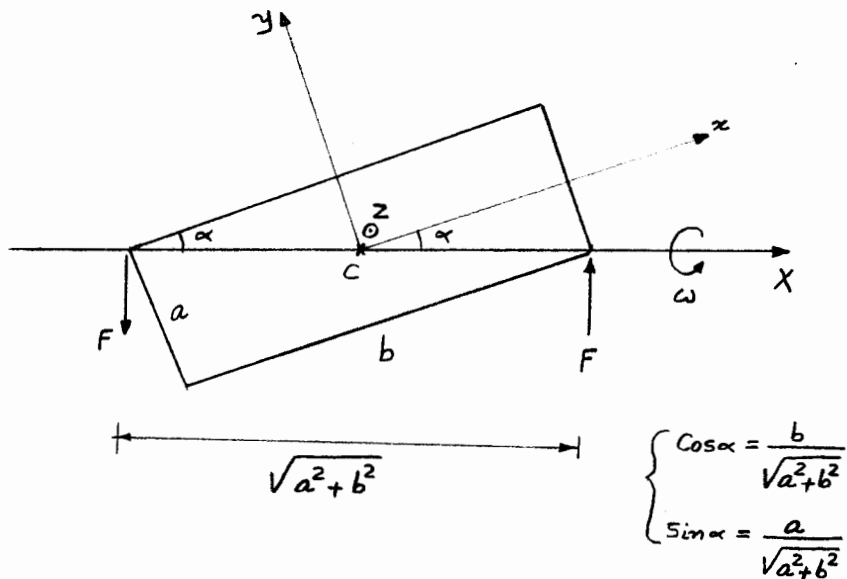


$$\text{No slip} \Rightarrow \begin{cases} \underline{v}_{D'}|_{disk} = \underline{v}_{D'}|_{ring} \\ A'D' = DD' \end{cases} \Rightarrow r\psi = R\phi \Rightarrow \psi = \frac{R}{r}\phi$$

$$\underline{\omega}_{disk} = \left[\dot{\theta} + \left(1 - \frac{R}{r}\right)\dot{\phi} \right] \hat{e}_z$$

Problem 2

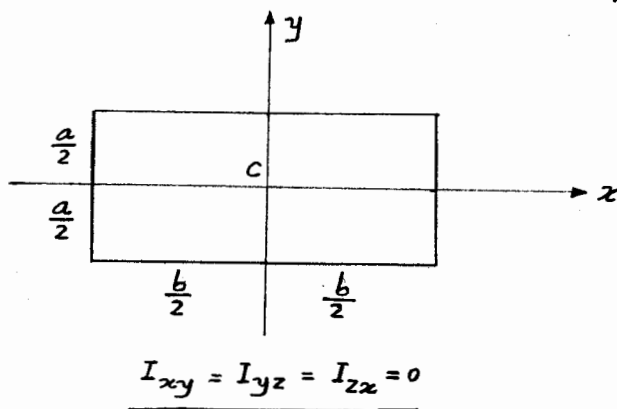
Introducing xyz coordinate system which is fixed to the plate and rotates with ω about X axis:



(a) Since $\underline{v}_C = 0$, forces on the bearings are equal and in opposite directions.

$$I_x = \int (\rho dV) (y^2 + z^2) \quad z^2 \approx 0 \text{ (thin plate)}$$

$$= \int_{-\frac{a}{2}}^{\frac{a}{2}} \int_{-\frac{b}{2}}^{\frac{b}{2}} \rho dx dy (y^2) = \rho b \frac{a^3}{12} = \frac{1}{12} M a^2$$



$$I_y = \frac{1}{12} M b^2 \quad I_z = \frac{1}{12} M (a^2 + b^2)$$

$$I_{xy} = I_{yz} = I_{zx} = 0$$

$$\underline{\omega}_{plate} = \omega \hat{e}_X = \omega \cos \alpha \hat{e}_x - \omega \sin \alpha \hat{e}_y \quad \rightarrow \quad \begin{cases} \omega_x = \omega \cos \alpha \\ \omega_y = -\omega \sin \alpha \\ \omega_z = 0 \end{cases}$$

$$\underline{H}_C = [I]_C \underline{\omega} \quad \Rightarrow \quad \underline{H}_C = I_x \omega_x \hat{e}_x + I_y \omega_y \hat{e}_y$$

$$\underline{z}_C = \frac{d\underline{H}_C}{dt}$$

$$\frac{d\underline{H}_C}{dt} = I_x \omega_x \frac{d\hat{e}_x}{dt} + I_y \omega_y \frac{d\hat{e}_y}{dt} \quad \begin{matrix} \omega_x \hat{e}_x = \omega \sin \alpha \hat{e}_z \\ \omega_y \hat{e}_y = \omega \cos \alpha \hat{e}_z \end{matrix}$$

$$= \left(\frac{1}{12} M a^2 \omega^2 \cos \alpha \sin \alpha - \frac{1}{12} M b^2 \omega^2 \sin \alpha \cos \alpha \right) \hat{e}_z = \frac{1}{12} M \omega^2 \sin \alpha \cos \alpha (a^2 - b^2) \hat{e}_z$$

$$\underline{z}_C = F \sqrt{a^2 + b^2} \hat{e}_z$$

$$\therefore F\sqrt{a^2+b^2} = \frac{1}{12} M\omega^2 \frac{ab}{a^2+b^2} (a^2-b^2)$$

$$\Rightarrow F = \frac{1}{12} M\omega^2 ab \frac{a^2-b^2}{(a^2+b^2)^{3/2}}$$

Note that force F rotates about X axis and is always in xy plane.

(b)

$$KE = \frac{1}{2} \{\omega\}^t [I]_c \{\omega\} + \frac{1}{2} M \cancel{v_c \cdot v_c} \rightarrow 0$$

$$= \frac{1}{2} (I_x \omega_x^2 + I_y \omega_y^2)$$

$$= \frac{1}{2} \left(\frac{1}{12} M a^2 \omega^2 \cos^2 \alpha + \frac{1}{12} M b^2 \omega^2 \sin^2 \alpha \right)$$

$$= \frac{1}{12} M \omega^2 \frac{a^2 b^2}{a^2 + b^2}$$

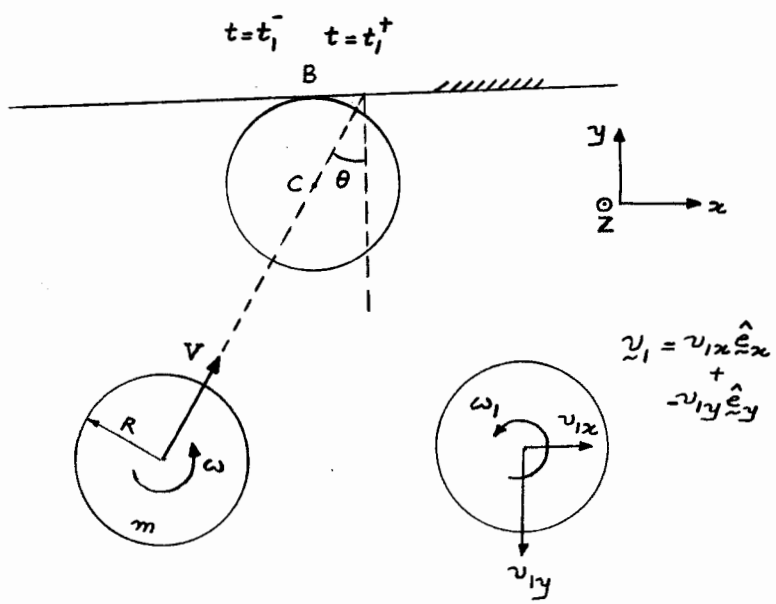
kinetic energy of the rotating plate

Problem 3

Assume the disk collides with the wall at point B at $t=t_1$.

An impulse acts on the disk at $t=t_1$ ($\Delta P_x, \Delta P_y$).

Collision in the normal direction (y) is elastic so the magnitude of the velocity in the normal direction is conserved:



$$v_{1y}|_{t=t_1^+} = v_y|_{t=t_1^-} \Rightarrow v_{1y}|_{t=t_1^+} = V \cos \theta$$

\therefore $v_{1y} = V \cos \theta$

No slip occurs at the wall. $\Rightarrow v_{Bx}|_{t=t_1^+} = 0 \Rightarrow [v_{Cx} + (\omega \times \underline{CB})_x]_{t=t_1^+} = 0$

$\Rightarrow v_{1x} - \omega_1 R = 0 \Rightarrow$ $v_{1x} = R \omega_1$

Angular momentum about point B: $\underline{\tau}_B = \frac{d}{dt} \underline{H}_B + \underline{r}_B \times \underline{P}$

$\underline{\tau}_B = 0 \rightarrow \frac{d}{dt} \underline{H}_B = 0 \rightarrow \underline{H}_B|_{t=t_1^-} = \underline{H}_B|_{t=t_1^+}$ ($\underline{H}_B = \underline{H}_C + \underline{BC} \times \underline{P}$)

$\rightarrow \frac{1}{2} m R^2 \omega + m V R \sin \theta = \frac{1}{2} m R^2 \omega_1 + m R v_{1x}$ ($\omega_1 = v_{1x}/R$)

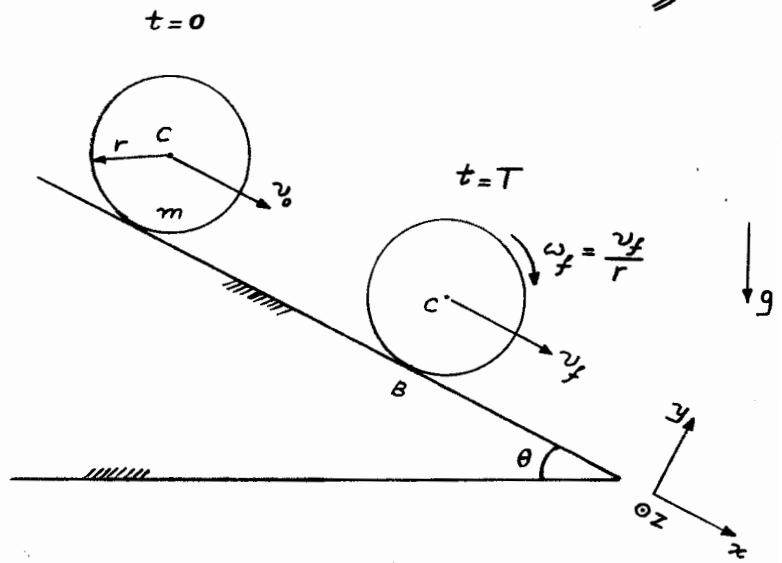
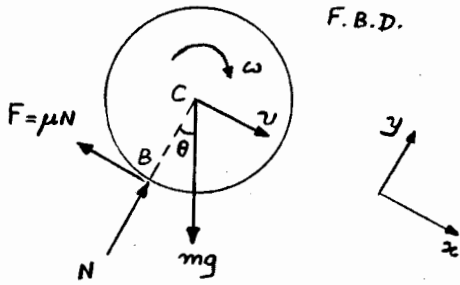
$v_{1x} = \frac{2}{3} V \sin \theta + \frac{1}{3} R \omega$

$v_1 = \sqrt{v_{1x}^2 + v_{1y}^2} = \sqrt{(\frac{2}{3} V \sin \theta + \frac{1}{3} R \omega)^2 + (V \cos \theta)^2}$

velocity of the center of the disk after collision

Problem 4

$0 < t < T$



End of the period of slipping $t=T$.

Linear mom. : $m \frac{dv}{dt} = F$

in x direction : $m \frac{dv}{dt} = mg \sin \theta - F$ ($F = \mu N$ during the period of slipping)

in y direction : $0 = mg \cos \theta - N \rightarrow N = mg \cos \theta$

$\therefore m \frac{dv}{dt} = mg (\sin \theta - \mu \cos \theta) \rightarrow \int_0^T dv = \int_0^T g (\sin \theta - \mu \cos \theta) dt$

$\Rightarrow v_f - v_0 = gT (\sin \theta - \mu \cos \theta)$

Ang. mom. about point C : $\tau_c = \frac{d}{dt} H_c$

$\left. \begin{aligned} \tau_c &= -\mu N r \hat{e}_z = -\mu mg \cos \theta r \hat{e}_z \\ \frac{d}{dt} H_c &= -I_c \frac{d\omega}{dt} \hat{e}_z = -\frac{2}{5} m r^2 \frac{d\omega}{dt} \hat{e}_z \end{aligned} \right\} \Rightarrow \mu g \cos \theta = \frac{2}{5} r \frac{d\omega}{dt}$

$\Rightarrow \int_0^T d\omega = \int_0^T \frac{5}{2} \frac{\mu}{r} g \cos \theta dt \rightarrow \omega_f - 0 = \frac{5}{2} \mu g \cos \theta \frac{T}{r}$

No slip at $t=T \Rightarrow \underline{r\omega_f = v_f}$

$$\therefore v_f = \frac{5}{2} \mu g \cos \theta T$$

$$v_f - v_0 = g T (\sin \theta - \mu \cos \theta)$$

$$\Rightarrow T = \frac{2v_0}{g(7\mu \cos \theta - 2\sin \theta)}$$

time duration of slipping

$$v_f = \frac{5\mu \cos \theta v_0}{7\mu \cos \theta - 2\sin \theta}$$

velocity of the center of mass C
at the end of the period
of slipping

Note that $7\mu \cos \theta - 2\sin \theta$ has to be positive :

$$\underline{\tan \theta < 3.5\mu}$$

Problem 5

$$AB = 2a$$

Horizontal impulse ΔP at $t=0$.

During the impulse period, other forces do not have enough time to act:

Linear momentum: $m \frac{d\vec{v}_C}{dt} = \vec{F}$

$$m d\vec{v}_C = \vec{F} dt$$

$$m (\vec{v}_C|_{t=0^+} - 0) = \int_0^{0^+} \vec{F} dt = \Delta \vec{P} = \Delta P \hat{e}_x$$

$$\Rightarrow \vec{v}_C|_{t=0^+} = \frac{\Delta P}{m} \hat{e}_x \quad \rightarrow \quad \begin{cases} v_{Cx} = \frac{\Delta P}{m} \\ v_{Cy} = 0 \end{cases}, \text{ at } t=0^+$$

Ang. mom. about C: $\tau_C = \frac{d}{dt} H_C = I_C \frac{d\omega}{dt} \hat{e}_z$

$$\tau_C dt = I_C d\omega \hat{e}_z \quad \Rightarrow \quad \int_0^{0^+} \tau_C dt = I_C (\omega|_{t=0^+} - 0) \hat{e}_z$$

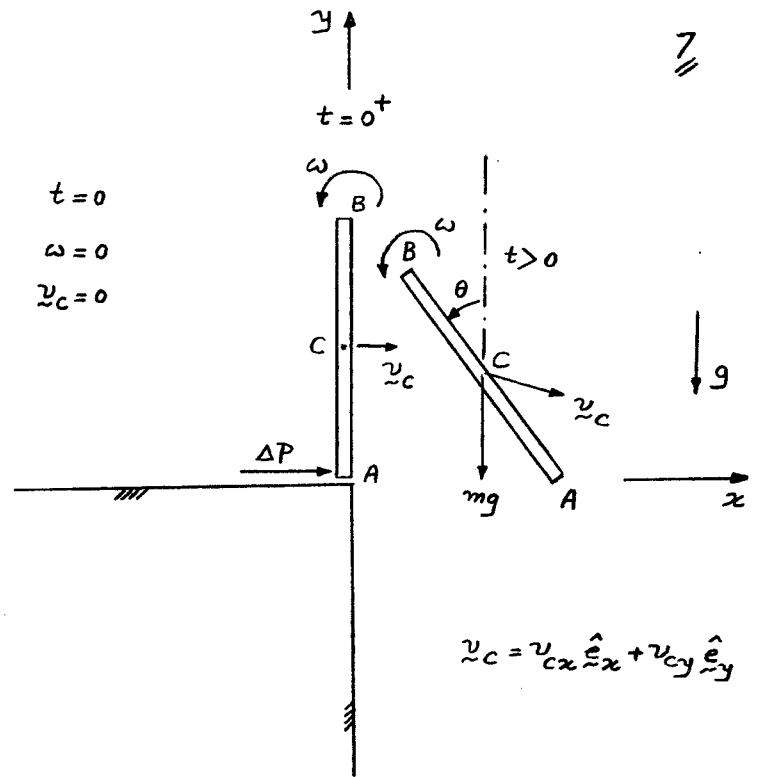
$$\Rightarrow \Delta P a = \frac{1}{12} m (2a)^2 [\omega|_{t=0^+} - 0] \quad \rightarrow \quad \omega|_{t=0^+} = \frac{3\Delta P}{ma}$$

$\vec{v}_C|_{t=0^+}$ and $\omega|_{t=0^+}$ are the initial conditions for the next stage of the motion

which is free fall:

$$\tau_C = \frac{d}{dt} H_C, \quad \tau_C = 0 \quad \Rightarrow \quad I_C \omega = \text{const.} \quad \rightarrow \quad \omega = \text{const.} = \omega|_{t=0^+} = \frac{3\Delta P}{ma}$$

$$m \frac{d\vec{v}_C}{dt} = \vec{F} = -mg \hat{e}_y$$



$$\vec{v}_C = v_{Cx} \hat{e}_x + v_{Cy} \hat{e}_y$$

$$\therefore \begin{cases} m \frac{dv_{cx}}{dt} = 0 \rightarrow v_{cx} = \text{const.} = v_{cx}|_{t=0^+} = \frac{\Delta P}{m} \\ m \frac{dv_{cy}}{dt} = -mg \rightarrow v_{cy} = -gt + v_{cy}|_{t=0^+} = -gt \end{cases}$$

$$\begin{cases} v_{cx} = \frac{dx_c}{dt} = \frac{\Delta P}{m} \rightarrow x_c = \frac{\Delta P}{m} t + x_c|_{t=0} = \frac{\Delta P}{m} t \\ v_{cy} = \frac{dy_c}{dt} = -gt \rightarrow y_c = -g \frac{t^2}{2} + y_c|_{t=0} = -g \frac{t^2}{2} + a \end{cases}$$

$$\omega = \frac{d\theta}{dt} = \frac{3\Delta P}{ma} \rightarrow \theta = \frac{3\Delta P}{ma} t + \theta|_{t=0} = \frac{3\Delta P}{ma} t$$

$$\begin{cases} x_B = x_c - a \sin \theta = \frac{\Delta P}{m} t - a \sin\left(\frac{3\Delta P}{ma} t\right) \\ y_B = y_c + a \cos \theta = -g \frac{t^2}{2} + a \left[1 + \cos\left(\frac{3\Delta P}{ma} t\right)\right] \end{cases}$$

Point B clips the edge of the table ($x=0, y=0$):

$$\begin{cases} x_B = 0 = \frac{\Delta P}{m} t - a \sin\left(\frac{3\Delta P}{ma} t\right) = 0 \xrightarrow{X = \frac{\Delta P t}{ma}} X - \sin(3X) = 0 \rightarrow X = 0.76 = \frac{\Delta P t}{ma} \\ y_B = 0 = -g \frac{t^2}{2} + a \left[1 + \cos\left(\frac{3\Delta P}{ma} t\right)\right] = 0 \end{cases}$$

$$-g \frac{t^2}{2} + a [1 + \cos(3X)] = 0 \rightarrow g \frac{t^2}{2} = a (1 - 0.65) \rightarrow t = 0.84 \sqrt{\frac{a}{g}}$$

$$\Delta P t = 0.76 ma \rightarrow \Delta P = \frac{0.76 ma}{0.84 \sqrt{\frac{a}{g}}} = 0.91 m \sqrt{ga}$$

value of horizontal impulse

At this instant:

$$\begin{cases} \theta = \frac{3\Delta P}{ma} t = 3(0.76) = 2.28 = 130.6^\circ \\ x_B = y_B = 0 \end{cases}$$

