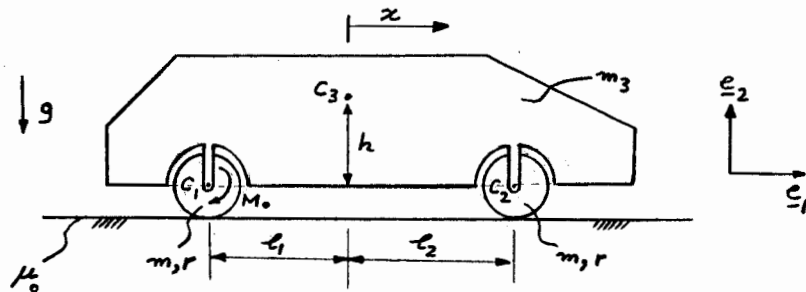

 Problem Set No. 5

Problem 1


(a) # DOF = 1

$$\omega|_{\text{body of the car}} = 0$$

$$\omega|_{\text{each wheel}} = \omega$$

$$I|_{\text{each wheel}} = \frac{1}{2}mr^2$$

Assuming pure rolling, $v_{C3} = v_{C1} = v_{C2} = \dot{x} = r\omega$

Kinetic Energy: $T = \frac{1}{2}m_3 v_{C3}^2 + 4 \left[\frac{1}{2}m\dot{x}^2 + \frac{1}{2} \left(\frac{1}{2}mr^2 \right) \omega^2 \right]$

$$T = \frac{1}{2}(m_3 + 6m)\dot{x}^2$$

$$\dot{T} = \frac{d}{dt}(\text{work of external forces}) = \text{power of external forces}$$

There is no displacement in the direction of gravity and normal forces.

Velocity at the point of attack of friction force is $\underline{0}$.

So the only thing that does work in this system is M_0 .

$$\therefore \dot{T} = \frac{d}{dt}(M_0\varphi) = M_0\omega = M_0\frac{\dot{x}}{r}$$

$$\dot{T} = (m_3 + 6m)\dot{x}\ddot{x}$$

$$\Rightarrow$$

$$\ddot{x} = \frac{M_0}{r(6m + m_3)}$$

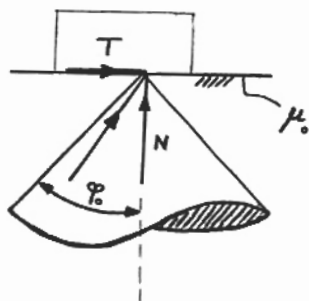
acceleration of
the car

Problem 1

(b) One can show the no slip condition geometrically:

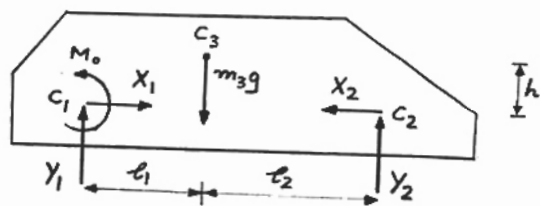
resultant reaction force must be within the cone.

$$\tan \varphi_0 = \mu_0$$



We need to find the forces acting on the rear wheel, then check whether the dynamical condition of rolling is satisfied.

Angular mom. about c_2 :



FBD for the body of the car

$$\frac{M}{C_2} = \dot{H}_{C_2} + \underline{v}_{C_2} \times \underline{P} = 0 \quad (v_{C_2} \parallel P)$$

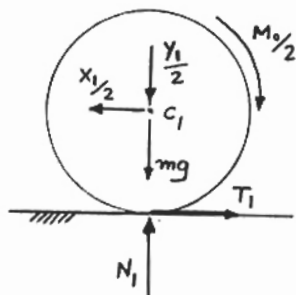
$$\underline{H}_{C_2} = \underline{H}_{C_3} + \underline{P} \times \underline{r}_{C_3 C_2} = \underline{I}_{C_3} \omega + m_3 \underline{v}_{C_3} \times \underline{r}_{C_3 C_2} = 0 \quad (\omega = 0) = -m_3 h \dot{x} \underline{e}_3$$

$$\underline{M}_{C_2} = [-Y_1(-l_1 + l_2) + m_3 g l_2 + M_0] \underline{e}_3$$

$$\therefore m_3 h \ddot{x} = Y_1(l_1 + l_2) - m_3 g l_2 - M_0$$

$$\rightarrow Y_1 = \frac{m_3(h\ddot{x} + g l_2) + M_0}{l_1 + l_2} = \frac{m_3 g l_2}{l_1 + l_2} + M_0 \frac{6mr + m_3(r+h)}{r(l_1 + l_2)(6m + m_3)} \quad (1)$$

FBD for one of the rear wheels:



Linear mom. in vertical direction:

$$m a_{C_1|y} = N_1 - \frac{Y_1}{2} - mg \rightarrow N_1 = mg + \frac{Y_1}{2} \quad (2)$$

Angular mom. about C_1 :

$$(\underline{T}_1 r - \frac{M_0}{2}) \underline{e}_3 = \dot{H}_{C_1} + \underline{v}_{C_1} \times \underline{P} = 0$$

Problem 1

$$(b) \quad \underline{H}_{C_1} = \underline{I}_{C_1} \underline{\omega} = -\left(\frac{1}{2}mr^2\right) \dot{\alpha} \underline{e}_3 = -\frac{1}{2}mr\dot{\alpha} \underline{e}_3 \quad \leftarrow |\underline{v}_{C_1}|$$

$$\therefore mr\ddot{\alpha} = -2T_1r + M_0 \quad \rightarrow \quad T_1 = \frac{M_0 - mr\ddot{\alpha}}{2r} = M_0 \frac{5m+m_3}{2r(6m+m_3)} \quad (3)$$

We have so far assumed the kinematic condition of rolling. Rolling will, however, only hold as long as its dynamic condition is satisfied: $\frac{T_1}{N_1} \leq \mu_0$ (4)

(1), (2), (3), (4) \Rightarrow

$$M_0 \frac{5m+m_3}{2r(6m+m_3)} \leq \mu_0 g \frac{2m\ell_1 + (2m+m_3)\ell_2}{2(\ell_1+\ell_2)} + \mu_0 M_0 \frac{6mr+m_3(r+h)}{2r(\ell_1+\ell_2)(6m+m_3)}$$

$$\rightarrow M_0 \leq \frac{\mu_0 r g (6m+m_3) [2m\ell_1 + (2m+m_3)\ell_2]}{(5m+m_3)(\ell_1+\ell_2) - \mu_0 [6mr+m_3(r+h)]}$$

Largest M_0 for which car does not slip.

Problem 2

Introducing xyz coordinate system fixed to the ring and as a result rotating with Ω about Z axis:

$$\underline{v}_P = \frac{d}{dt} \underline{OP} = \frac{d}{dt} (\underline{OC} + \underline{CP}) = \underline{v}_C + \underline{v}_P|_{\text{rel. to } C}$$

$$\begin{aligned} \underline{v}_C &= \frac{d}{dt} \underline{OC} = \Omega \underline{e}_z \times \underline{OC} \\ &= \Omega \left(\frac{\sqrt{3}}{2} \underline{e}_z - \frac{1}{2} \underline{e}_y \right) \times (-a \underline{e}_y) \\ &= a \Omega \frac{\sqrt{3}}{2} \underline{e}_x \end{aligned}$$

$$\underline{v}_P|_{\text{rel. to } C} = \frac{d}{dt} \underline{CP} = \underline{\omega}|_{CP} \times \underline{CP}$$

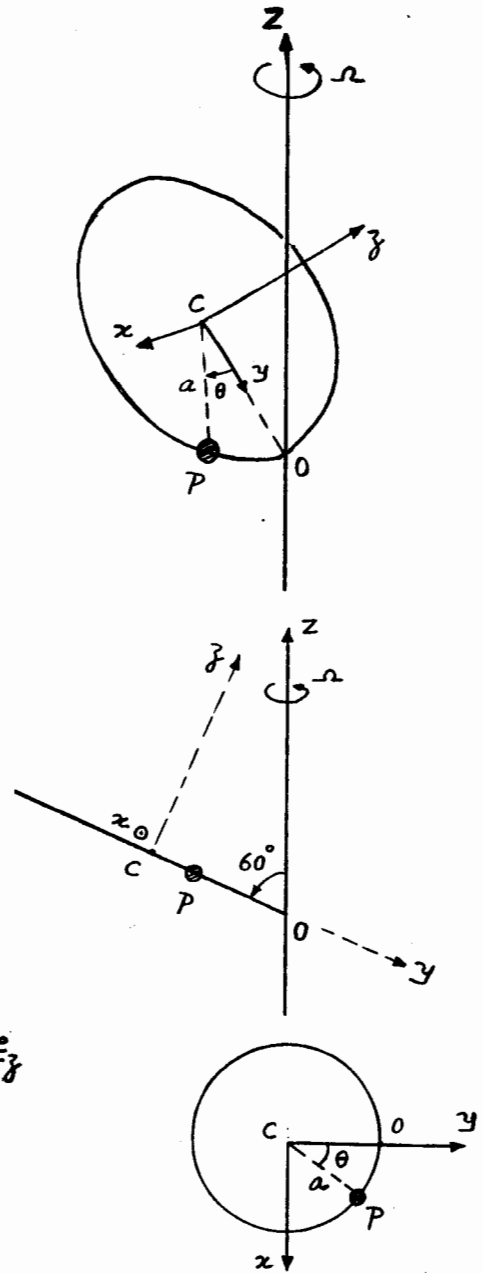
$$\underline{\omega}|_{CP} = \Omega \underline{e}_z - \dot{\theta} \underline{e}_z = -\frac{\Omega}{2} \underline{e}_y + \left(\Omega \frac{\sqrt{3}}{2} - \dot{\theta} \right) \underline{e}_z$$

$$\underline{CP} = a \sin \theta \underline{e}_x + a \cos \theta \underline{e}_y$$

$$\begin{aligned} \Rightarrow \underline{v}_P|_{\text{rel. to } C} &= \left[-\frac{\Omega}{2} \underline{e}_y + \left(\Omega \frac{\sqrt{3}}{2} - \dot{\theta} \right) \underline{e}_z \right] \times \left[a \sin \theta \underline{e}_x + a \cos \theta \underline{e}_y \right] \\ &= \frac{\Omega}{2} a \sin \theta \underline{e}_z + a \sin \theta \left(\Omega \frac{\sqrt{3}}{2} - \dot{\theta} \right) \underline{e}_y - a \cos \theta \left(\Omega \frac{\sqrt{3}}{2} - \dot{\theta} \right) \underline{e}_x \end{aligned}$$

$$\therefore \underline{v}_P = \left[a \Omega \frac{\sqrt{3}}{2} (1 - \cos \theta) + a \dot{\theta} \cos \theta \right] \underline{e}_x + \left[a \sin \theta \left(\Omega \frac{\sqrt{3}}{2} - \dot{\theta} \right) \right] \underline{e}_y + \left[a \frac{\Omega}{2} \sin \theta \right] \underline{e}_z$$

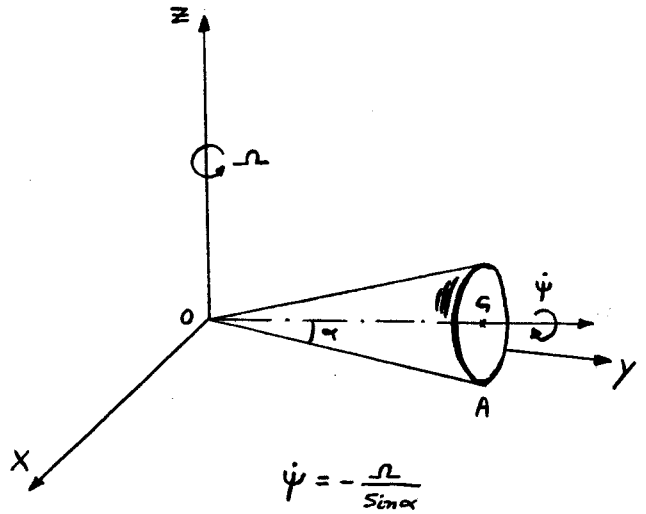
Velocity of P in terms of θ and $\dot{\theta}$



Problem 3

xyz rotates with Ω about z axis.

$$\begin{aligned} \underline{\omega}_{\text{Cone}} &= -\Omega \cot \alpha \underline{e}_{OA} \\ &= -\Omega \cot \alpha (-\cos \alpha \underline{e}_z + \sin \alpha \underline{e}_y) \\ &= -\Omega \cos \alpha \underline{e}_y + \Omega \frac{\cos^2 \alpha}{\sin \alpha} \underline{e}_z \end{aligned}$$



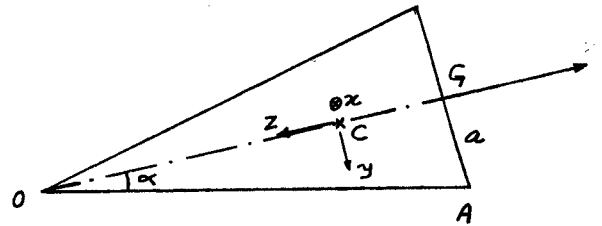
(a)

Angular momentum of the cone

about the tip O: $\underline{H}_O = \underline{I}_O \underline{\omega}$ ($\underline{v}_O = \underline{0}$)

$$\underline{I}_O = \begin{bmatrix} I_1 & 0 & 0 \\ 0 & I_1 & 0 \\ 0 & 0 & I_3 \end{bmatrix}$$

$$\underline{\omega} = \begin{cases} \omega_x = 0 \\ \omega_y = -\Omega \cos \alpha \\ \omega_z = \Omega \frac{\cos^2 \alpha}{\sin \alpha} \end{cases}$$



$$a = \sqrt{\frac{10}{3} \frac{I_3}{M}}$$

$$\therefore \underline{H}_O = \underline{I}_O \underline{\omega} = -I_1 \Omega \cos \alpha \underline{e}_y + I_3 \Omega \frac{\cos^2 \alpha}{\sin \alpha} \underline{e}_z$$

(b)

$$\underline{P} = M \underline{v}_C = M (\dot{\theta} \underline{e}_z \times \underline{OC}) = M \dot{\theta} \left| \frac{3}{4} |OG| \right| \cos \alpha \underline{e}_x = \frac{3}{4} M \Omega a \frac{\cos^2 \alpha}{\sin \alpha} \underline{e}_x$$

$$\underline{F} = \dot{\underline{P}} = \frac{3}{4} M \Omega a \frac{\cos^2 \alpha}{\sin \alpha} \frac{d\underline{e}_x}{dt} \quad \begin{aligned} \Omega \underline{e}_z \times \underline{e}_x &= \Omega (-\cos \alpha \underline{e}_y - \sin \alpha \underline{e}_z) \times \underline{e}_x \\ &= \Omega (\cos \alpha \underline{e}_z - \sin \alpha \underline{e}_y) = \Omega \underline{e}_{AO} \end{aligned}$$

$$\underline{F} = \frac{3}{4} M \Omega^2 a \frac{\cos^2 \alpha}{\sin \alpha} \underline{e}_{AO}$$

total required force

Problem 3

(b)

$$\underline{M}_0 = \dot{\underline{H}}_0 \quad (\underline{v}_0 = \underline{0})$$

$$\dot{\underline{H}}_0 = \dot{\underline{H}}_0 + \omega|_{xyz} \times \underline{H}_0 = \Omega \underline{e}_z \times \underline{H}_0$$

$$= \Omega \underline{e}_z \times \left(-I_1 \Omega \cos\alpha \underline{e}_y + I_3 \Omega \frac{\cos^2\alpha}{\sin\alpha} \underline{e}_z \right)$$

$$\Omega \underline{e}_z \times \underline{e}_y = \Omega (-\cos\alpha \underline{e}_y - \sin\alpha \underline{e}_z) \times \underline{e}_y = \Omega \sin\alpha \underline{e}_x$$

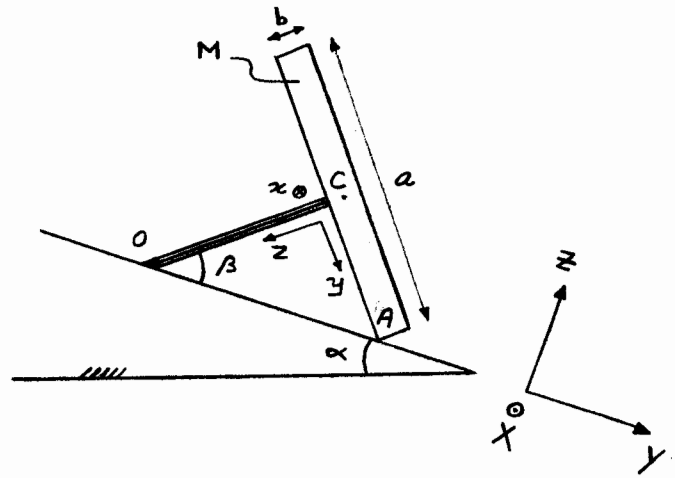
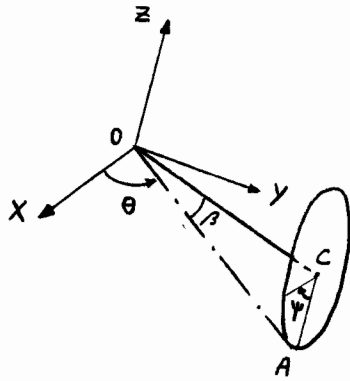
$$\Omega \underline{e}_z \times \underline{e}_z = \Omega (-\cos\alpha \underline{e}_y - \sin\alpha \underline{e}_z) \times \underline{e}_z = -\Omega \cos\alpha \underline{e}_x$$

$$\underline{M}_0 = -\Omega^2 \left(I_1 \sin\alpha \cos\alpha + I_3 \frac{\cos^3\alpha}{\sin\alpha} \right) \underline{e}_x$$

total required torque
about point 0

Problem 4

7



$$I_1 = I_2 = I_{xx} = I_{yy} = \frac{Ma^2}{16}$$

$$I_3 = I_{zz} = \frac{Ma^2}{8}$$

$$\underline{\omega}_{disk} = \dot{\theta} \underline{e}_z + \dot{\psi} \underline{e}_z$$

$$\underline{e}_z = -\cos\beta \cos\theta \underline{e}_x - \cos\beta \sin\theta \underline{e}_y - \sin\beta \underline{e}_z$$

$$\underline{\omega}_{disk} = -\dot{\psi} \cos\beta \cos\theta \underline{e}_x - \dot{\psi} \cos\beta \sin\theta \underline{e}_y + (\dot{\theta} - \dot{\psi} \sin\beta) \underline{e}_z$$

No slip : $\underline{v}_A|_{disk} = \underline{0}$

$$\underline{v}_A|_{disk} = \underline{v}_O + \underline{\omega}|_{disk} \times \underline{OA} = \underline{0}$$

$$\underline{OA} = \frac{a}{2\sin\beta} (\cos\theta \underline{e}_x + \sin\theta \underline{e}_y)$$

$$\underline{\omega}|_{disk} \times \underline{OA} = \underline{0} \implies \dot{\psi} = \frac{\dot{\theta}}{\sin\beta} \implies \underline{\omega}_{disk} = -\dot{\theta} \cot\beta \underline{e}_{OA}$$

$$T = \frac{1}{2} M \underline{v}_C \cdot \underline{v}_C + \frac{1}{2} \underline{\omega}^T \underline{I}_C \underline{\omega} = \frac{1}{2} M \underline{v}_C \cdot \underline{v}_C + \frac{1}{2} (I_1 \omega_x^2 + I_1 \omega_y^2 + I_3 \omega_z^2)$$

$$\underline{v}_C = \dot{\theta} \underline{e}_z \times \underline{OC} = \dot{\theta} \left(\frac{a}{2\tan\beta} + \frac{b}{2} \right) \cos\beta \underline{e}_x \implies \underline{v}_C \cdot \underline{v}_C = \dot{\theta}^2 \cos^2\beta \left(\frac{a \cos\beta}{2\sin\beta} + \frac{b}{2} \right)^2$$

$$\underline{\omega}_{disk} = -\dot{\theta} \cot\beta \underline{e}_{OA} = -\dot{\theta} \cot\beta (-\cos\beta \underline{e}_z + \sin\beta \underline{e}_y) \implies \omega_x = 0, \omega_y = -\dot{\theta} \cos\beta, \omega_z = \dot{\theta} \frac{\cos\beta}{\sin\beta}$$

$$\therefore T = \frac{1}{8} M \dot{\theta}^2 \cot^2\beta (a \cos\beta + b \sin\beta)^2 + \frac{1}{2} I_1 \dot{\theta}^2 \cos^2\beta + \frac{1}{2} I_3 \dot{\theta}^2 \frac{\cos^4\beta}{\sin^2\beta} = \frac{\cos^2\beta}{32} M a^2 \dot{\theta}^2 \left(1 + 6 \cot^2\beta + 4 \frac{b^2}{a^2} + 8 \frac{b}{a} \cot\beta \right)$$

kinetic energy of the disk