

2.092/2.093

FINITE ELEMENT OF SOLIDS AND FLUIDS I

FALL 2009

Homework 6- solution

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Assigned: Session 14
Due: Session 16

Problem 1 (20 points):

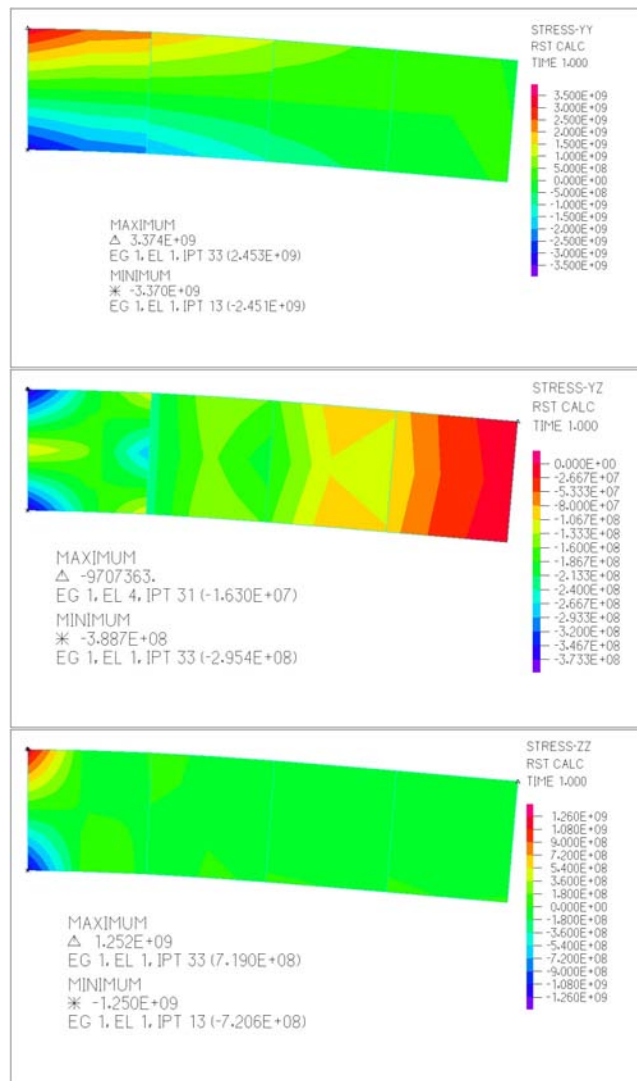


Figure 1: Stress distributions with 1×4 mesh

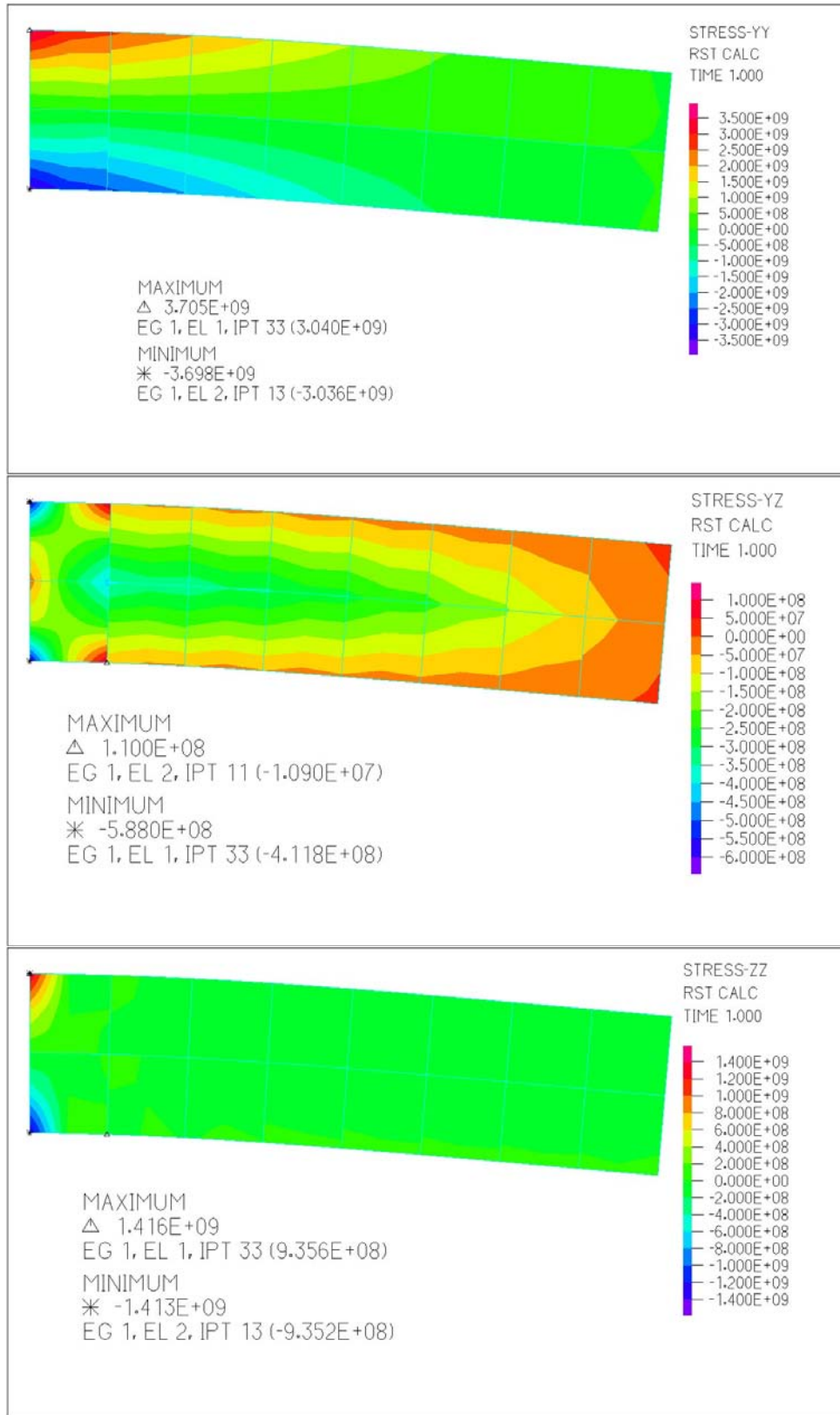


Figure 2: Stress distributions with 2x8 mesh

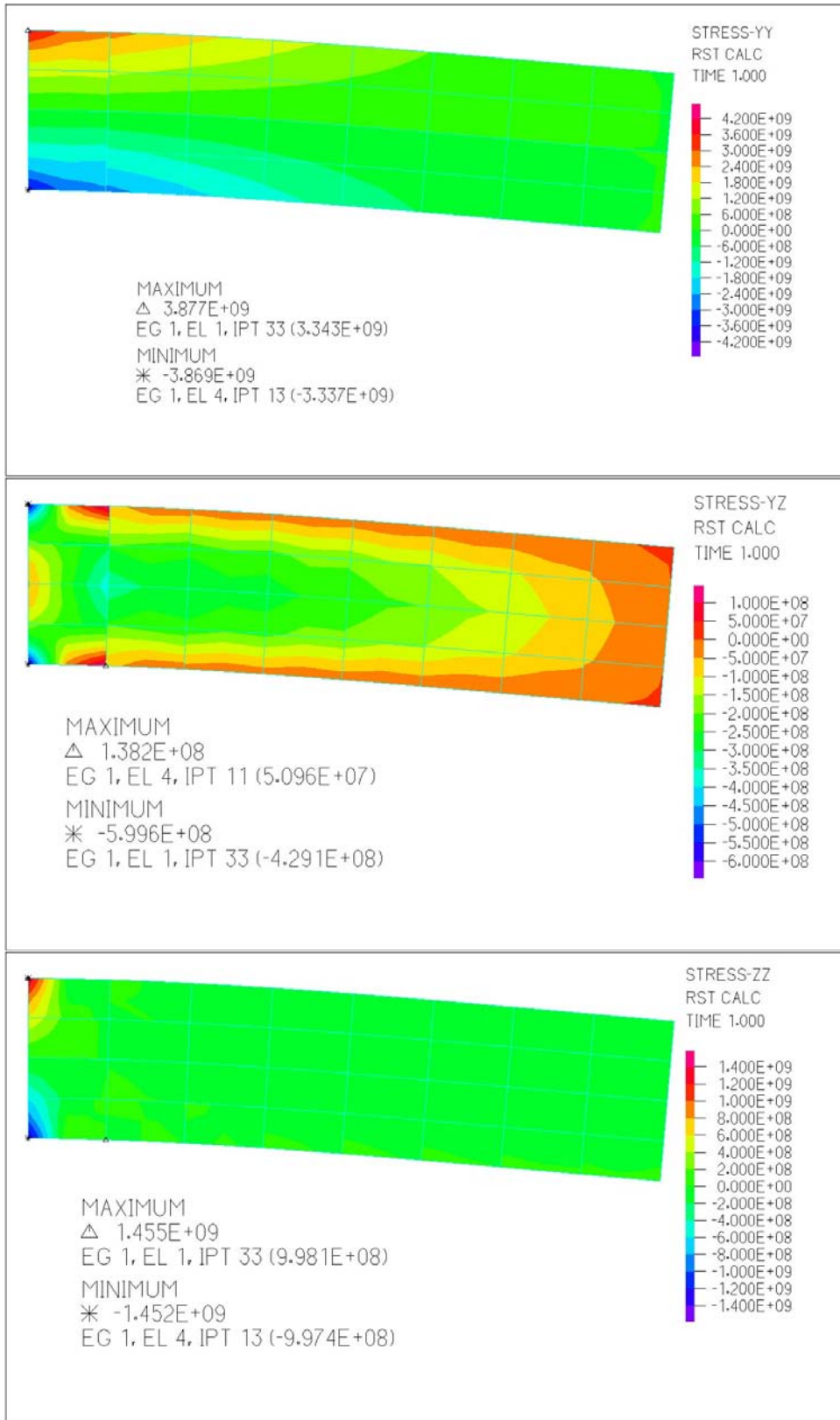


Figure 3: Stress distributions with 4x8 mesh

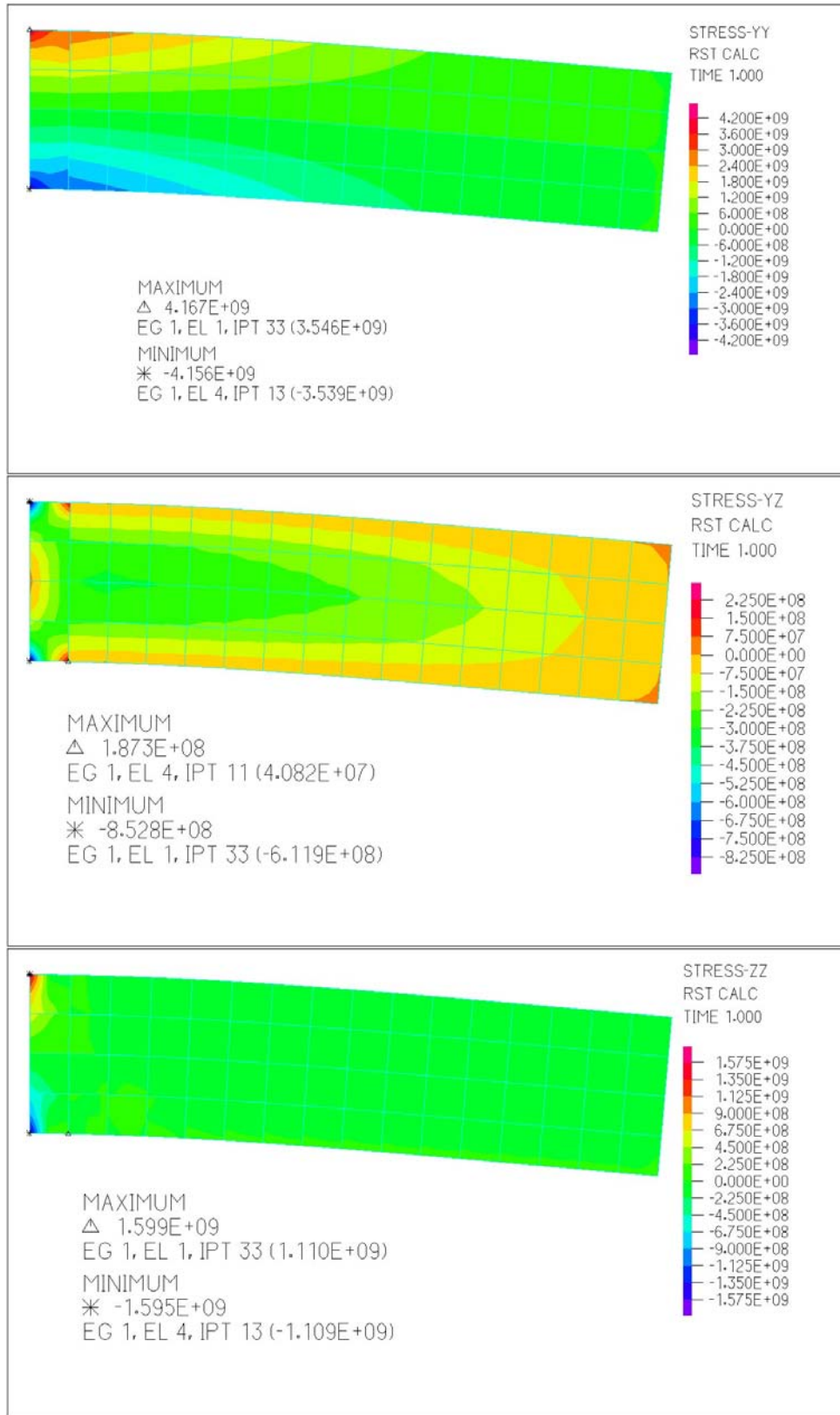


Figure 4: Stress distributions with 4x16 mesh

· Euler-Bernoulli Beam Theory

$$\text{Governing equation: } EI \frac{d^4 w}{dx^4} = p(x) = -\frac{p_0}{L} x.$$

$$\text{Boundary conditions: } w(0) = 0; \quad \frac{dw(0)}{dx} = 0; \quad \frac{d^2 w(L)}{dx^2} = 0; \quad \frac{d^3 w(L)}{dx^3} = 0.$$

After solving the governing equation with boundary conditions, we are able to obtain

$$\tau_{xx} = -Ez \frac{d^2 w}{dx^2} = Ez \left(\frac{p_0}{2EIL} \right) \left(\frac{x^3}{3} - L^2 x + \frac{2}{3} L^3 \right) \text{ and } \tau_{xx} = \frac{2p_0 L^2}{h^2} = 3.2 \times 10^9 \text{ N/m}^2 \text{ at } x=0,$$

$$z = h/2.$$

Table 1: Numerical results for maximum stresses

| Mesh | Max. stress τ_{yy} | Max. stress τ_{yz} | Max. stress τ_{zz} |
|------|-------------------------|-------------------------|-------------------------|
| 1×4 | 3.37E+09 | 9.71E+06 | 1.25E+09 |
| 2×8 | 3.71E+09 | 1.10E+08 | 1.42E+09 |
| 4×8 | 3.88E+09 | 1.38E+08 | 1.46E+09 |
| 4×16 | 4.17E+09 | 1.87E+08 | 1.60E+09 |

The finite analysis stress, τ_{yy} , appears to keep increasing to infinite stress at the top corner of the beam as the number of elements increases. This is because of the stress concentration caused by the geometry discontinuity. However, in real the stress never reach infinity since the material will yield before this point.

Problem 2 (20 points):

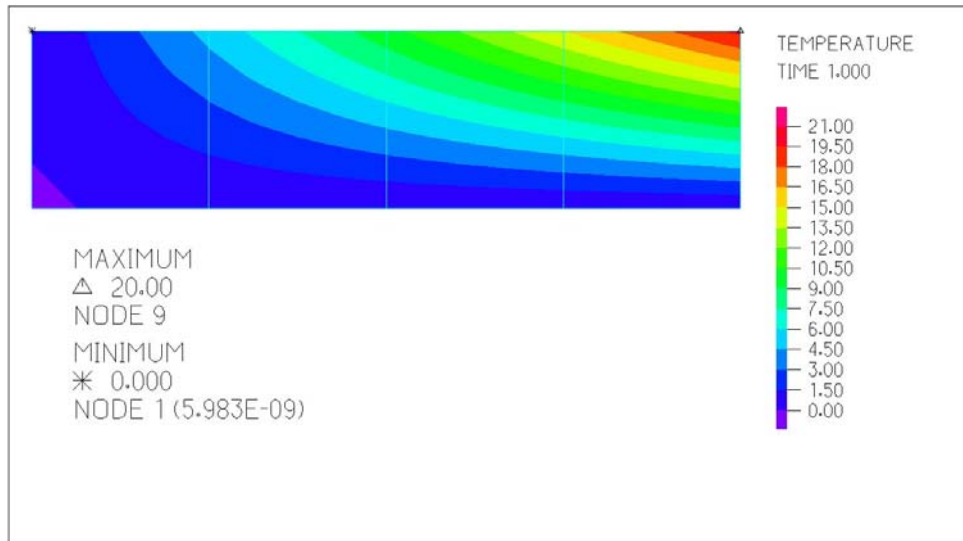


Figure 5: Temperature distribution with 1×4 mesh

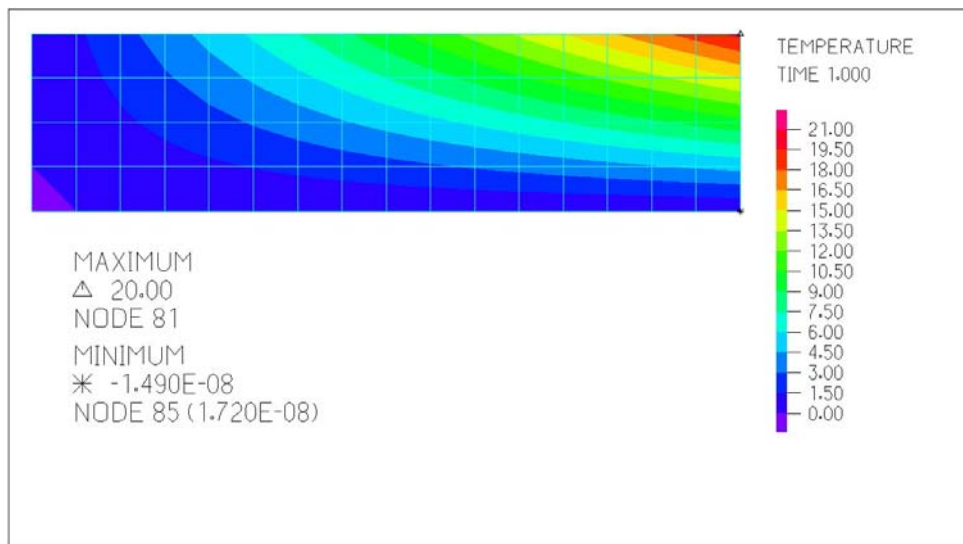


Figure 6: Temperature distribution with 4×16 mesh

The temperature distribution obtained from finite analysis seems to have converged even with a very coarse mesh.

Problem 3 (20 points):

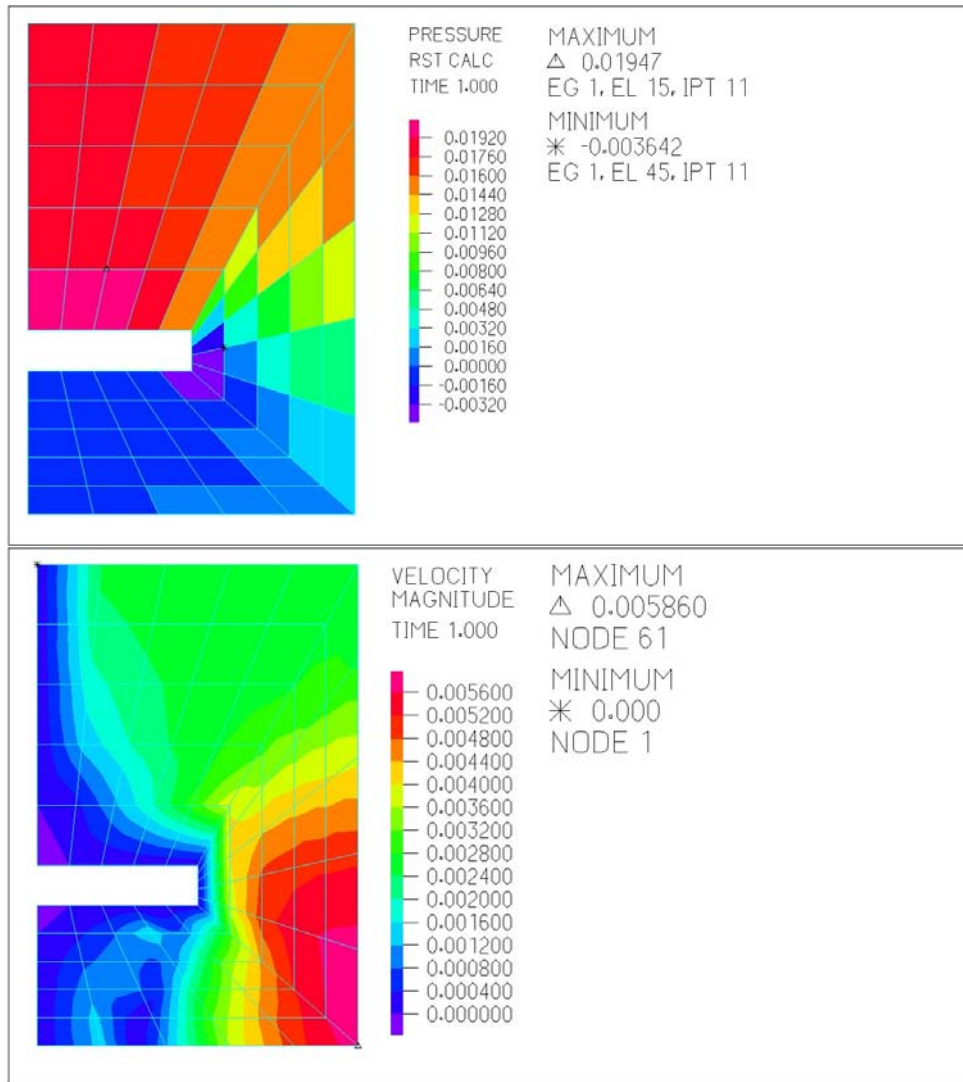


Figure 7: Velocity and pressure distributions with 75 4-node elements

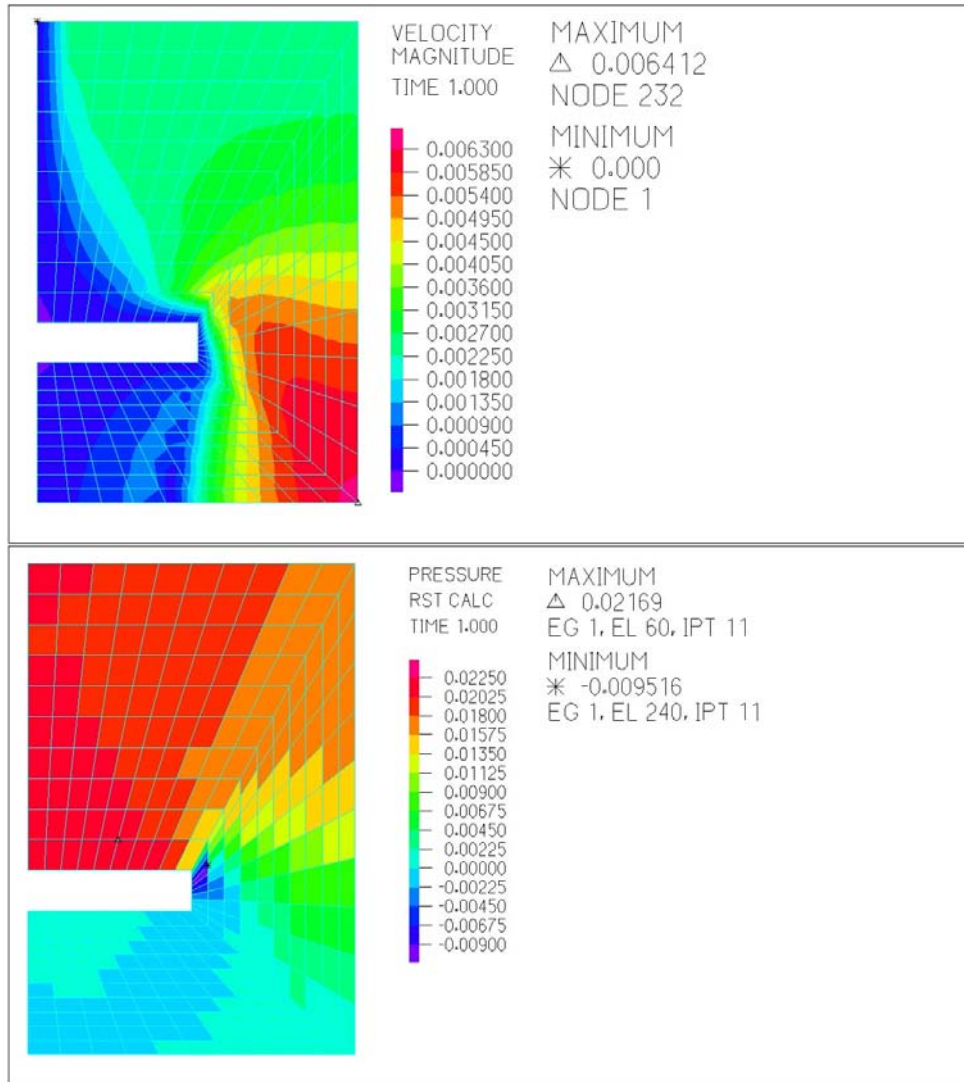


Figure 8: Velocity and pressure distributions with 300 4-node elements

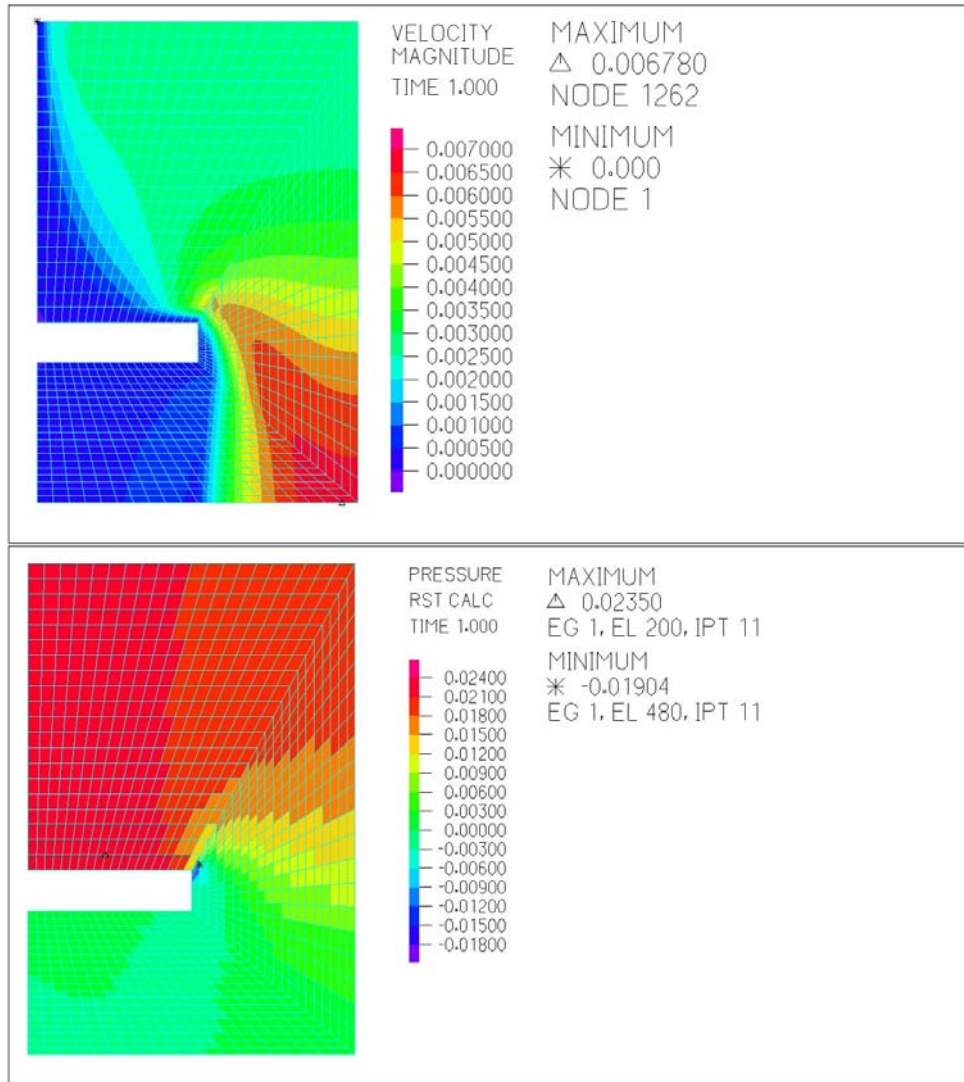


Figure 9: Velocity and pressure distributions with 1200 4-node elements

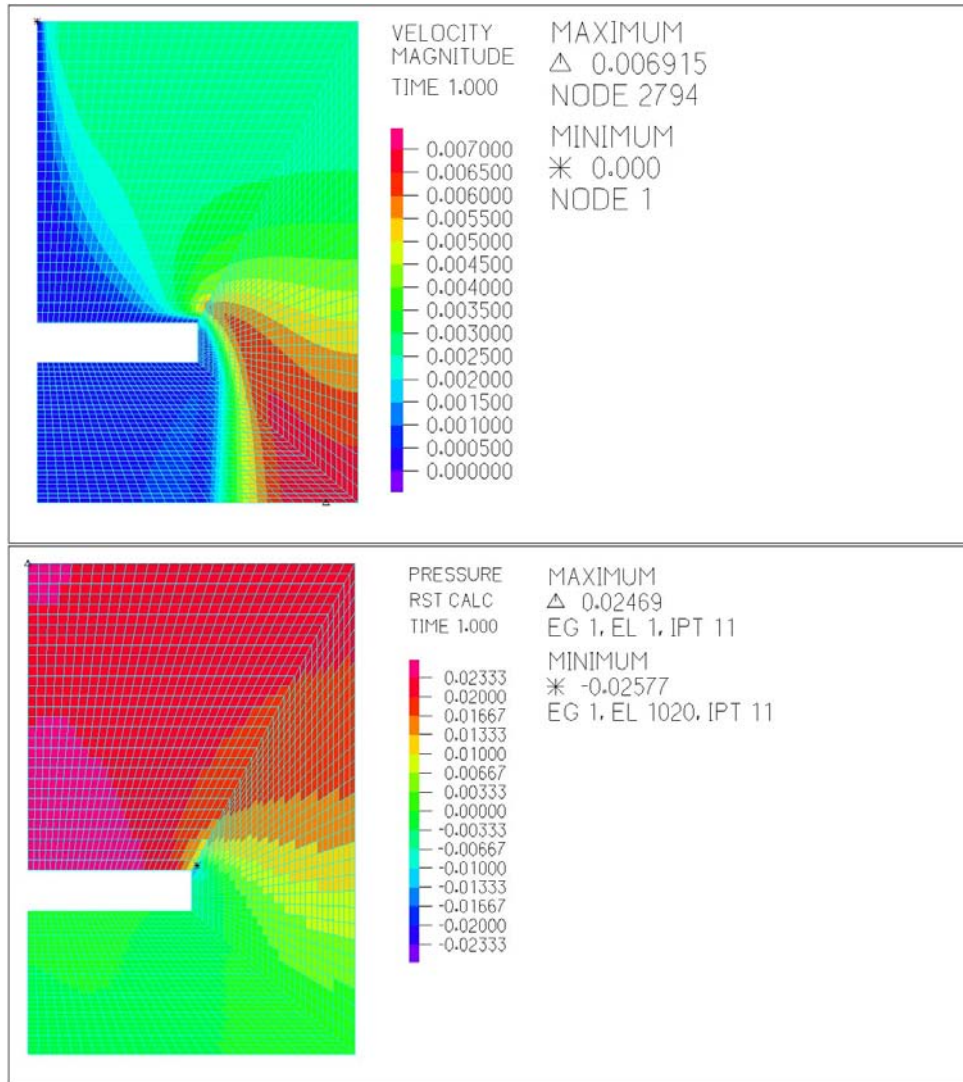


Figure 10: Velocity and pressure distributions with 2700 4-node elements

The velocity and pressure distributions are converging as the number of elements increases. Since the Reynolds number featuring this flow is relatively low, the laminar flow assumption is reasonable.

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