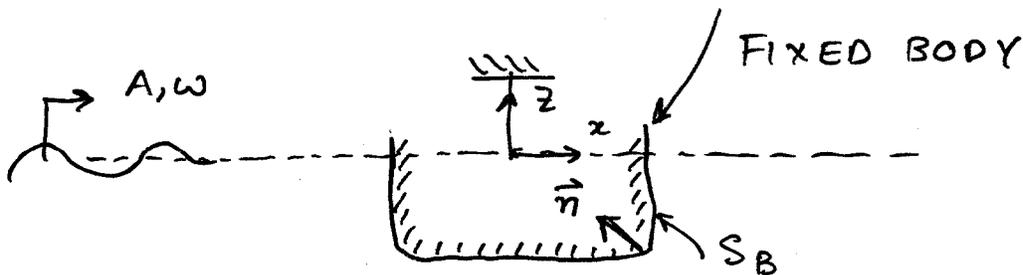


## EXCITING FORCES AND HYDRODYNAMIC COEFFICIENTS

- THERE REMAINS TO DERIVE THE BOUNDARY VALUE PROBLEMS LEADING TO THE DEFINITION OF THE EXCITING FORCES  $X_j(\omega)$  AND THE IMPEDANCE HYDRODYNAMIC COEFFICIENTS  $A_{ij}(\omega), B_{ij}(\omega); i, j = 1, 3, 4$
- THE STEPS FOLLOWED IN TWO DIMENSIONS AND THE RELATIONS THAT FOLLOW EXTEND ALMOST WITH NO CHANGE IN THREE DIMENSIONS.

### EXCITING FORCES



$$\phi_I = \text{Re} \left\{ \frac{igA}{\omega} e^{kz - ikx + i\omega t} \right\}, \quad k = \omega^2/g$$

$$\begin{aligned}\zeta_I &= A \cos(\omega t - kx) \\ &= A \cos \omega t, \quad x=0 \\ &= A \operatorname{Re} \{ e^{i\omega t} \}\end{aligned}$$

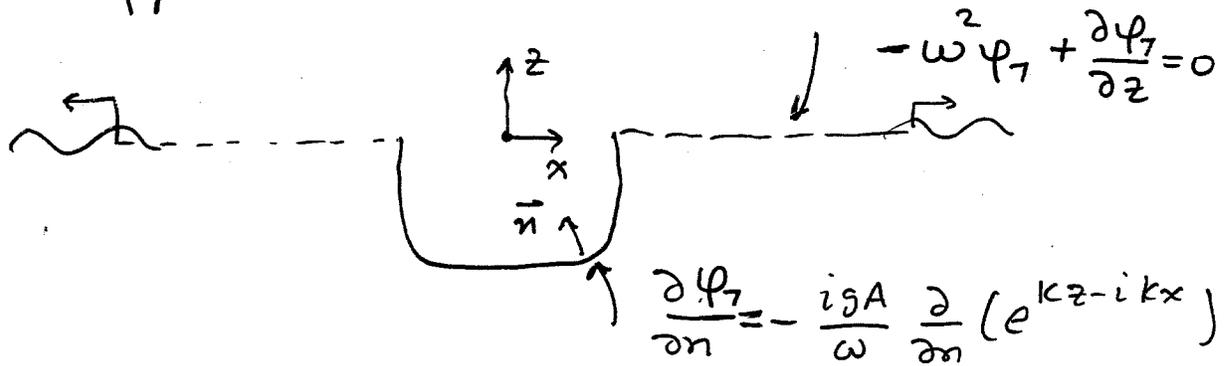
PLANE PROGRESSIVE WAVES WITH POTENTIAL  $\phi_I$  ARE INCIDENT TOWARDS A FIXED 2D BODY. THE CORRESPONDING WAVE ELEVATION IS DEFINED ABOVE AND HAS A ZERO PHASE RELATIVE TO  $x=0$ .

- ALL OTHER SEAKEEPING QUANTITIES HAVE A PHASE MEASURED RELATIVE TO THAT OF  $\zeta$  AT  $x=0$ . —

LET  $\psi$  BE THE "DIFFRACTION" VELOCITY POTENTIAL DEFINED AS FOLLOWS

$$\left\{ \begin{array}{l} \psi = \operatorname{Re} \{ \psi_T e^{i\omega t} \} \\ \frac{\partial}{\partial n} (\phi_I + \psi) = 0, \quad \text{on } S_B \\ (-\omega^2 + g \partial_z) \psi_T = 0, \quad \text{on } z=0 \end{array} \right.$$

THE BOUNDARY-VALUE PROBLEM SATISFIED BY  $\varphi_7$  BECOMES:



AT INFINITY,  $\varphi_7$  REPRESENTS THE WAVES SCATTERED BY THE BODY WHICH MUST BE OUTGOING. SO AS  $|x| \rightarrow \infty$

$$\varphi_7 \propto \begin{cases} \frac{igA^+}{\omega} e^{kz - ikx}, & x \rightarrow +\infty \\ \frac{igA^-}{\omega} e^{kz + ikx}, & x \rightarrow -\infty \end{cases}$$

WHERE  $A^\pm$  ARE UNKNOWN COMPLEX AMPLITUDES

- THE SOLUTION OF THE BVP FOR  $\varphi_7$  DEFINED ABOVE AND SIMILAR PROBLEMS DERIVED BELOW IS CARRIED OUT ROUTINELY WITH PANEL METHODS, DISCUSSED LATER.—

FOLLOWING THE SOLUTION FOR  $\phi_7$  THE HYDRODYNAMIC PRESSURE FOLLOWS FROM THE LINEAR BERNOULLI EQUATION:

$$\begin{aligned} \phi &= -\rho \frac{\partial}{\partial t} (\phi_I + \psi) \\ &= -\rho \operatorname{Re} \{ i\omega (\varphi_I + \varphi_7) e^{i\omega t} \} \\ &= \rho g A \operatorname{Re} \{ (\chi_I + \chi_7) e^{i\omega t} \} \end{aligned}$$

WHERE THE DEFINITIONS:

$$\varphi_I \equiv \frac{igA}{\omega} \chi_I, \quad \varphi_7 = \frac{igA}{\omega} \chi_7$$

WERE USED, WITH  $\chi_I = e^{kz - ikx}$ .

● BY DEFINITION:

$$\vec{F} = \iint_{S_B} p \vec{n} \, ds, \quad \vec{n} = (n_1, n_3)$$

WITH

$$F_1 = \iint_{S_B} p n_1 \, ds = \operatorname{Re} \{ X_1(\omega) e^{i\omega t} \}$$

$$F_3 = \iint_{S_B} p n_3 \, ds = \operatorname{Re} \{ X_3(\omega) e^{i\omega t} \}$$

IT FOLLOWS THAT:

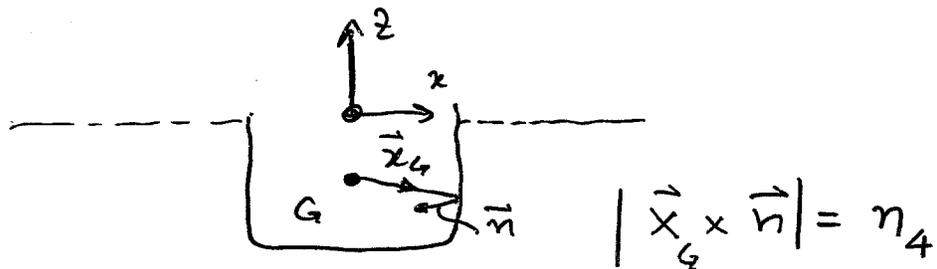
$$\bullet \quad X_1(\omega) = \rho g A \iint_{S_B} (X_I + X_7) n_1 ds$$

$$\bullet \quad X_3(\omega) = \rho g A \iint_{S_B} (X_I + X_7) n_3 ds$$

WITH:

$$X_I = e^{kz - ikx} \dots$$

THE ROLL MOMENT ABOUT THE CENTER OF GRAVITY  $G$  IS DEFINED IN AN ANALOGOUS MANNER



$$\vec{M} = \iint_{S_B} (\vec{x}_G \times \vec{n}) p ds$$

IT FOLLOWS THAT THE COMPLEX ROLL MOMENT BECOMES:

$$\bullet \quad X_4(\omega) = \rho g A \iint_{S_B} (X_I + X_7) \eta_4 ds$$

THE COMPLEX WAVE EXCITING FORCES

$\chi_j(\omega)$  ;  $j=1, 3, 4$  CAN BE DECOMPOSED

IN AN OBVIOUS MANNER AS FOLLOWS:

$$\chi_j(\omega) = \overset{\text{FK}}{\chi_j(\omega)} + \overset{\text{DIF}}{\chi_j(\omega)}$$

WHERE

$$\chi_j^{\text{FK}} = \rho g A \iint_{S_B} \chi_I \eta_j ds, \quad j=1, 3, 4$$

$$\chi_j^{\text{DIF}} = \rho g A \iint_{S_B} \chi_7 \eta_j ds, \quad j=1, 3, 4$$

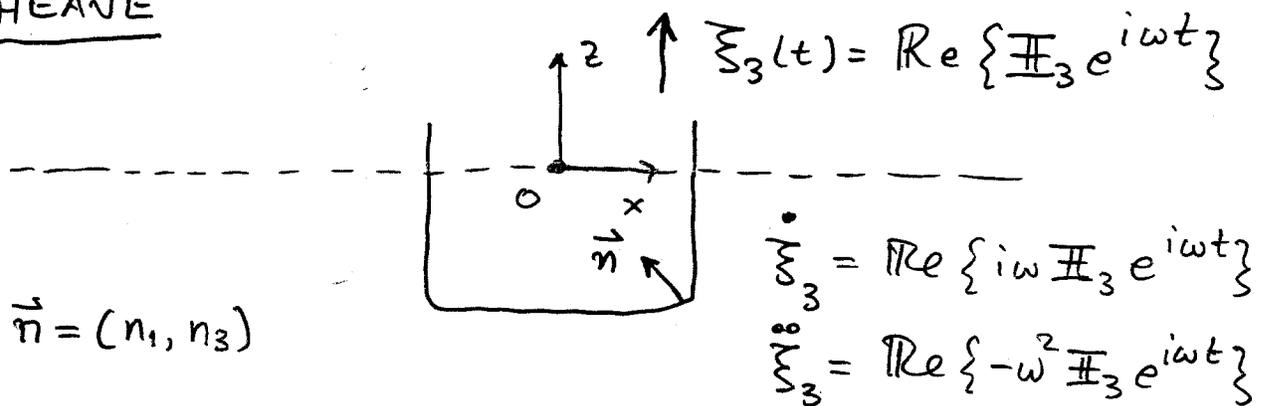
- THE FROUDE-KRYLOV COMPONENT IS ONLY A FUNCTION OF THE AMBIENT WAVE POTENTIAL AND VERY EASY TO EVALUATE.
- THE DIFFRACTION COMPONENT IS A FUNCTION OF THE DIFFRACTION POTENTIAL AND REQUIRES THE SOLUTION OF A BOUNDARY VALUE PROBLEM. —
- FURTHER PROPERTIES OF  $\chi_j^{\text{DIF}}$  ARE DISCUSSED LATER. —

## ADDED MASS AND DAMPING COEFFICIENTS

THE PRINCIPAL STEPS FOR THE EVALUATION OF  $A_{ij}(\omega)$  ARE PRESENTED FOR HEAVE.

THE IDEAS EXTEND TRIVIALY TO SURGE AND ROLL.—

HEAVE



DUE TO THE FORCED OSCILLATION OF THE BODY WITH DISPLACEMENT  $\xi_3(t)$  A FLUID DISTURBANCE WILL RESULT WITH VELOCITY POTENTIAL

$$\phi_3 = \text{Re} \{ \psi_3 e^{i\omega t} \}$$

$$\text{LET } \psi_3 = \Xi_3 \psi_3 ;$$

$$\text{ON } z=0; \quad -\omega^2 \psi_3 + g \psi_{3z} = 0. \text{---}$$

- THE NORMAL COMPONENT OF THE BODY VELOCITY IN THE DIRECTION OF THE UNIT NORMAL VECTOR  $\vec{n}$  DUE TO THE HEAVE VELOCITY  $\dot{\xi}_3 \vec{k}$  IS SIMPLY:

$$V_{3n} = \dot{\xi}_3 \vec{k} \cdot \vec{n} = \dot{\xi}_3 \eta_3$$

WHERE  $\vec{k}$  IS THE UNIT VECTOR IN DIRECTION  $z$ .

- THE ZERO RELATIVE NORMAL FLUX CONDITION ON THE BODY BOUNDARY REQUIRES THAT:

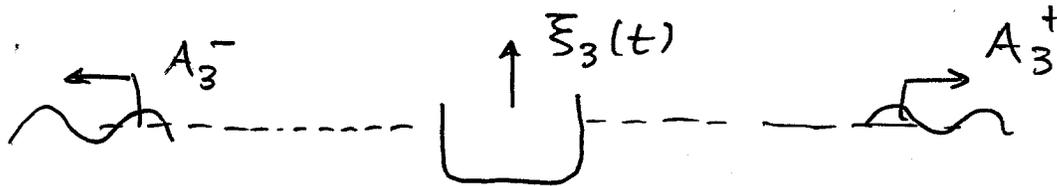
$$\frac{\partial \phi_3}{\partial n} = V_{3n} = \dot{\xi}_3 \eta_3$$

EXPRESSING IN COMPLEX FORM, WE OBTAIN:

$$\frac{\partial \psi_3}{\partial n} = i\omega \eta_3, \text{ on } \overline{S_B}$$

ENFORCED ON THE MEAN POSITION OF THE BODY BOUNDARY DUE TO LINEARITY. —

DUE TO THE FORCED OSCILLATION OF THE BODY IN HEAVE, THE RESULTING WAVE DISTURBANCE WILL REPRESENT OUTGOING WAVES AT INFINITY



SO:

$$\psi_3 \sim \begin{cases} \frac{igA_3^+}{\omega} e^{kz - ikx + i\omega t}, & x \rightarrow +\infty \\ \frac{igA_3^-}{\omega} e^{kz + ikx + i\omega t}, & x \rightarrow -\infty \end{cases}$$

WHERE  $A_3^\pm$  ARE THE COMPLEX AMPLITUDES OF THE WAVES RADIATED AWAY. FOR A SYMMETRIC BODY AND FOR THE HEAVE MOTION

$$|A_3^+| = |A_3^-|. -$$

THIS COMPLETES THE STATEMENT OF THE BOUNDARY VALUE PROBLEM SATISFIED BY  $\psi_3$  WHICH IS ALSO SOLVED BY PANEL METHODS. -

ASSUMING THAT THE SOLUTION FOR  $\varphi_3$  HAS BEEN CARRIED OUT, THE HYDRODYNAMIC PRESSURE FOLLOWS AGAIN FROM BERNOULLI:

$$p_3 = -\rho \frac{\partial \phi_3}{\partial t} = -\rho \operatorname{Re} \{ i\omega \varphi_3 e^{i\omega t} \}$$

THE FORCE ACTING ON THE BODY IN THE HEAVE DIRECTION FOLLOWS:

$$F_3 = \iint_{S_B} p_3 n_3 ds = \operatorname{Re} \{ F_3 e^{i\omega t} \}$$

WITH:

$$F_3 = -\rho \iint_{S_B} i\omega \varphi_3 n_3 ds$$

$$= -i\omega\rho \Xi_3 \iint_{S_B} \varphi_3 n_3 ds$$

- WHEN STATING NEWTON'S LAW,  $F_3(t)$  WAS ALSO DEFINED IN TERMS OF THE ADDED MASS AND DAMPING COEFFICIENTS:

$$F_3(t) = -A_{33}(\omega) \frac{d^2 \xi_3}{dt^2} - B_{33}(\omega) \frac{d\xi_3}{dt}$$

A DEFINITION VALID ONLY FOR HARMONIC MOTION.

EXPRESSING IN COMPLEX FORM:

$$F_3(t) = -\operatorname{Re} \left\{ \left[ -\omega^2 A_{33}(\omega) \mathbb{H}_3 + i\omega B_{33}(\omega) \mathbb{H}_3 \right] \times e^{i\omega t} \right\}$$

EQUATING THE TWO DEFINITIONS OF  $F_3(t)$   
AND KEEPING THE COMPLEX TERMS:

$$\omega^2 A_{33}(\omega) - i\omega B_{33}(\omega) = -i\omega\rho \iint_{S_B} \psi_3 \eta_3 ds$$

THIS EQUATION DEFINES THE HEAVE ADDED  
MASS AND DAMPING COEFFICIENTS IN TERMS  
OF  $\psi_3$ , OBTAINED FROM THE SOLUTION OF  
THE FREE-SURFACE BOUNDARY-VALUE PROBLEM

NOTING THAT:  $\frac{\partial \psi_3}{\partial n} = i\omega \eta_3$ , on  $S_B$

WE MAY ALSO USE:

$$\omega^2 A_{33}(\omega) - i\omega B_{33}(\omega) = -\rho \iint_{S_B} \psi_3 \frac{\partial \psi_3}{\partial n} ds$$

THIS DEFINITION EXTENDS TO ALL OTHER MODES OF MOTION TRIVIAALLY AND FROM TWO TO THREE DIMENSIONS:

- $$\omega^2 A_{ij}(\omega) - i\omega B_{ij}(\omega) = -\rho \iint_{S_B} \psi_i \frac{\partial \psi_j}{\partial n} ds$$

WHERE FOR SURGE AND HEAVE :

$$\frac{\partial \psi_j}{\partial n} = i\omega \eta_j \quad ; \quad j = 1, 3$$

AND FOR THE ROLL MOTION ABOUT THE ORIGIN:

$$\frac{\partial \psi_4}{\partial n} = i\omega \eta_4, \quad \eta_4 = |\vec{x} \times \vec{n}| = z\eta_1 - x\eta_3 \dots$$

(VERIFY)

ASSUMING THAT  $\psi_i$  IS AVAILABLE ON  $S_B$ :

- $$A_{ij}(\omega) = \text{Re} \left\{ -\frac{\rho}{\omega^2} \iint_{S_B} \psi_i \frac{\partial \psi_j}{\partial n} ds \right\}$$

- $$B_{ij}(\omega) = \text{Im} \left\{ -\frac{\rho}{\omega} \iint_{S_B} \psi_i \frac{\partial \psi_j}{\partial n} ds \right\}$$

## SYMMETRY - RECIPROCITY RELATIONS

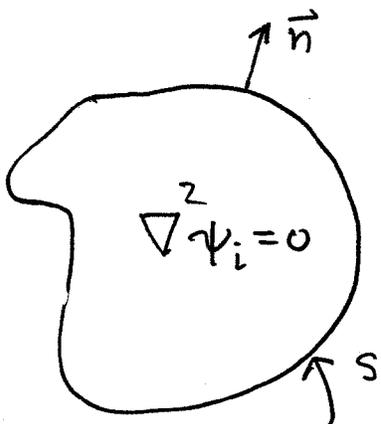
IT WILL BE SHOWN THAT:

$$A_{ij}(\omega) = A_{ji}(\omega)$$

$$B_{ij}(\omega) = B_{ji}(\omega)$$

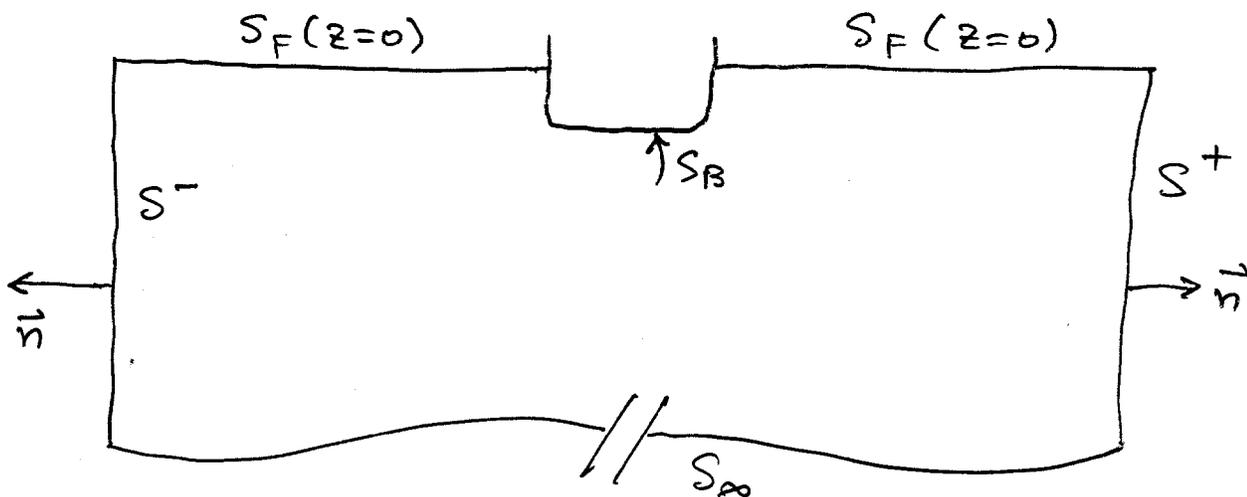
ALONG THE SAME LINES IT WILL BE SHOWN THAT THE EXCITING FORCE  $X_j$  CAN BE EXPRESSED IN TERMS OF  $\psi_j$  CIRCUMVENTING THE SOLUTION FOR THE DIFFRACTION POTENTIAL

THE CORE RESULT NEEDED FOR THE PROOF OF THE ABOVE PROPERTIES IS GREEN'S THM:



$$\oint_S (\psi_1 \frac{\partial \psi_2}{\partial n} - \psi_2 \frac{\partial \psi_1}{\partial n}) ds = 0$$

IN THE SURFACE WAVE-BODY PROBLEM DEFINE THE CLOSED SURFACE  $S$  AS FOLLOWS:



LET  $\varphi_j$  BE RADIATION OR DIFFRACTION POTENTIALS. OVER THE BOUNDARIES  $S^\pm$  WE HAVE:

$$S^+ : \varphi_j \sim \frac{igA_j^+}{\omega} e^{kz-ikx}$$

$$\frac{\partial \varphi_j}{\partial n} = \frac{\partial \varphi_j}{\partial x} \sim -ik \varphi_j$$

$$S^- : \varphi_j \sim \frac{igA_j^-}{\omega} e^{kz+ikx}$$

$$\frac{\partial \varphi_j}{\partial n} = -\frac{\partial \varphi_j}{\partial x} \sim -ik \varphi_j$$

$$\text{ON } S_F : \frac{\partial \varphi_j}{\partial z} = k \varphi_j, \quad \frac{\partial \varphi_j}{\partial n} = \frac{\partial \varphi_j}{\partial z}$$

$$\text{ON } S_\infty : |\varphi_j|, |\nabla \varphi_j| \rightarrow 0. \text{---}$$

APPLYING GREEN'S IDENTITY TO ANY PAIR OF THE RADIATION POTENTIALS  $\psi_i, \psi_j$ :

$$\begin{aligned}
 \iint_{S_B} \left[ \psi_i \frac{\partial \psi_j}{\partial n} - \psi_j \frac{\partial \psi_i}{\partial n} \right] ds &= \underbrace{\hspace{10em}}_{=0} \\
 - \iint_{S_F} \left[ \psi_i \frac{\partial \psi_j}{\partial z} - \psi_j \frac{\partial \psi_i}{\partial z} \right] ds & \\
 \quad \quad \quad \parallel \quad \quad \quad \parallel & \\
 \quad \quad \quad k\psi_j \quad \quad \quad k\psi_i & \\
 - \iint_{S^+} \left[ \psi_i \frac{\partial \psi_j}{\partial x} - \psi_j \frac{\partial \psi_i}{\partial x} \right] ds & \\
 \quad \quad \quad \parallel \quad \quad \quad \parallel & \\
 \quad \quad \quad \sim -ik\psi_j \quad \quad \quad \sim -ik\psi_i & \\
 \quad \quad \quad \underbrace{\hspace{10em}}_{=0} & \\
 + \iint_{S^-} \left[ \psi_i \frac{\partial \psi_j}{\partial x} - \psi_j \frac{\partial \psi_i}{\partial x} \right] ds & \\
 \quad \quad \quad \parallel \quad \quad \quad \parallel & \\
 \quad \quad \quad \sim ik\psi_j \quad \quad \quad \sim ik\psi_i & \\
 \quad \quad \quad \underbrace{\hspace{10em}}_{=0} & \\
 = 0 &
 \end{aligned}$$

IT FOLLOWS THAT:

$$\iint_{S_B} \psi_i \frac{\partial \psi_j}{\partial n} ds = \iint_{S_B} \psi_j \frac{\partial \psi_i}{\partial n} ds \quad \text{OR}$$

$$A_{ij}(\omega) = A_{ji}(\omega), \quad B_{ij}(\omega) = B_{ji}(\omega). \quad \text{—}$$

## HASKIND RELATIONS OF EXCITING FORCES

$$\begin{aligned} X_i(\omega) &= -i\omega\rho \iint_{S_B} (\varphi_I + \varphi_7) n_i ds \\ &= -\rho \iint_{S_B} (\varphi_I + \varphi_7) \frac{\partial \varphi_i}{\partial n} ds \end{aligned}$$

WHERE THE RADIATION VELOCITY POTENTIAL  $\varphi_i$  IS KNOWN TO SATISFY:

$$\frac{\partial \varphi_i}{\partial n} = i\omega n_i, \text{ on } S_B$$

AND

$$\frac{\partial \varphi_7}{\partial n} = -\frac{\partial \varphi_I}{\partial n}, \text{ on } S_B$$

BOTH  $\varphi_i$  AND  $\varphi_7$  SATISFY THE CONDITION OF OUTGOING WAVES AT INFINITY. BY VIRTUE OF GREEN'S THM:

$$\iint_{S_B} \varphi_7 \frac{\partial \varphi_i}{\partial n} ds = \iint_{S_B} \varphi_i \frac{\partial \varphi_7}{\partial n} ds = -\iint_{S_B} \varphi_i \frac{\partial \varphi_I}{\partial n} ds$$

THE HASKIND EXPRESSION FOR THE EXCITING FORCE FOLLOWS:

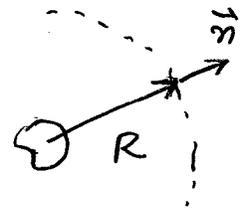
$$X_i(\omega) = -\rho \iint_{S_B} \left[ \varphi_I \frac{\partial \varphi_i}{\partial n} - \varphi_i \frac{\partial \varphi_I}{\partial n} \right] ds$$

## COMMENTS:

- THE SYMMETRY OF THE  $A_{ij}(\omega)$ ,  $B_{ij}(\omega)$  MATRICES APPLIES IN 2D AND 3D. THE APPLICATION OF GREEN'S THM IN 3D IS VERY SIMILAR USING THE FAR-FIELD REPRESENTATION FOR THE POTENTIAL  $\varphi_j$

$$\varphi_j \sim \frac{A_j(\theta)}{\sqrt{R}} e^{kz - ikR} + O(1/R^{3/2})$$

$$\frac{\partial \varphi_j}{\partial n} = \frac{\partial \varphi_j}{\partial R} \sim -ik \varphi_j + O(1/R^{3/2})$$



WHERE  $R$  IS A RADIUS FROM THE BODY OUT TO INFINITY AND THE  $R^{-1/2}$  DECAY ARISES FROM THE ENERGY CONSERVATION PRINCIPLE. DETAILS OF THE 3D PROOF MAY BE FOUND IN MEI AND W&L

- THE USE OF THE HASKIND RELATIONS FOR THE EXCITING FORCES DOES NOT REQUIRE THE SOLUTION OF THE DIFFRACTION PROBLEM. THIS IS CONVENIENT AND OFTEN MORE ACCURATE.

● THE HASKIND RELATIONS TAKE OTHER FORMS WHICH WILL NOT BE PRESENTED HERE BUT ARE DETAILED IN W&L.

THE ONES THAT ARE USED IN PRACTICE RELATE THE EXCITING FORCES TO THE DAMPING COEFFICIENTS.

THEY TAKE THE FORM :

$$\underline{2D}: \quad B_{ii} = \frac{|X_i|^2}{2\rho g V_g}, \quad V_g = \frac{g}{2\omega}, \quad \text{DEEP WATER}$$

$$\begin{aligned} \underline{3D}: & \quad B_{33} = \frac{k}{4\rho g V_g} |X_3|^2 - \text{HEAVE} \\ & \quad B_{22} = \frac{k}{8\rho g V_g} |X_2|^2 - \text{SWAY} \end{aligned}$$

(AXISYMMETRIC BODIES)

SO KNOWLEDGE OF  $X_i(\omega)$  ALLOWS THE DIRECT EVALUATION OF THE DIAGONAL DAMPING COEFFICIENTS. THESE EXPRESSIONS ARE USEFUL IN DERIVING THEORETICAL RESULTS IN WAVE-BODY INTERACTIONS TO BE DISCUSSED LATER.

THE TWO-DIMENSIONAL THEORY OF WAVE-BODY INTERACTIONS IN THE FREQUENCY DOMAIN EXTENDS TO THREE DIMENSIONS VERY DIRECTLY WITH LITTLE DIFFICULTY

THE STATEMENT OF THE 6 DOF SEA KEEPING PROBLEM IS:

$$\sum_{j=1}^6 \left[ -\omega^2 (M_{ij} + A_{ij}) + i\omega B_{ij} + C_{ij} \right] \Xi_j = X_i, \quad i=1, \dots, 6$$

WHERE:

$M_{ij}$ : BODY INERTIA MATRIX INCLUDING MOMENTS OF INERTIA FOR ROTATIONAL MODES. FOR DETAILS REFER TO MH

$A_{ij}(\omega)$ : ADDED MASS MATRIX

$B_{ij}(\omega)$ : DAMPING MATRIX

$C_{ij}$ : HYDROSTATIC AND STATIC INERTIA RESTORING MATRIX. FOR DETAILS REFER TO MH.

$X_i(\omega)$ : WAVE EXCITING FORCES AND MOMENTS.

AT ZERO SPEED THE DEFINITIONS OF THE ADDED-MASS, DAMPING MATRICES AND EXCITING FORCES ARE IDENTICAL TO THOSE IN TWO DIMENSIONS.

THE BOUNDARY VALUE PROBLEMS SATISFIED BY THE RADIATION POTENTIALS  $\varphi_j, j=1, \dots, 6$  AND THE DIFFRACTION POTENTIAL  $\varphi_7$  ARE AS FOLLOWS:

FREE-SURFACE CONDITION

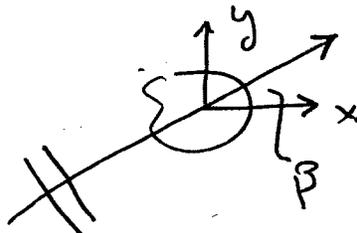
$$-\omega^2 \varphi_j + g \frac{\partial \varphi_j}{\partial z} = 0, \quad z=0$$

$$j=1, \dots, 7$$

BODY-BOUNDARY CONDITIONS

$$\frac{\partial \varphi_7}{\partial n} = -\frac{\partial \varphi_I}{\partial n}, \quad \text{on } S_B$$

$$\varphi_I = \frac{igA}{\omega} e^{kz - ikx \cos \beta -iky \sin \beta + i\omega t}$$



$$i=1: \text{SURGE} \quad \frac{\partial \varphi_j}{\partial n} = i\omega \eta_j, \quad j=1, \dots, 6$$

$i=2: \text{SWAY}$

$i=3: \text{HEAVE}$

$i=4: \text{ROLL}$

$i=5: \text{PITCH}$

$i=6: \text{YAW}$

$$\eta_j = \left\{ \begin{array}{l} \eta_j, \quad j=1, 2, 3 \\ (\vec{x} \times \vec{n})_{j+3}, \quad j=4, 5, 6 \end{array} \right\}$$

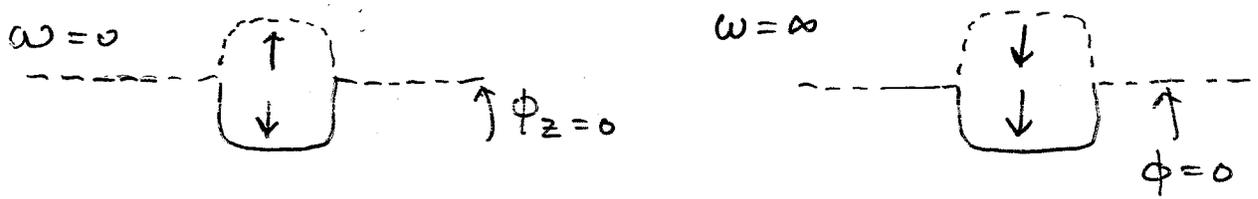
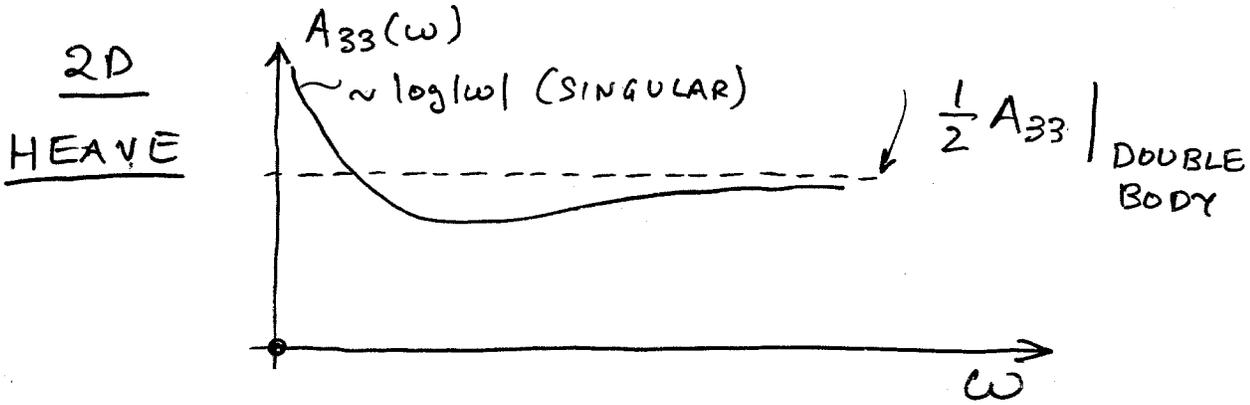
AT LARGE DISTANCES FROM THE BODY THE VELOCITY POTENTIALS SATISFY THE RADIATION CONDITION:

$$\varphi_j(R, \theta) \sim \frac{A_j(\theta)}{\sqrt{R}} e^{kz - ikR} + O(1/R^{3/2})$$

WITH  $k = \omega^2/g$ .

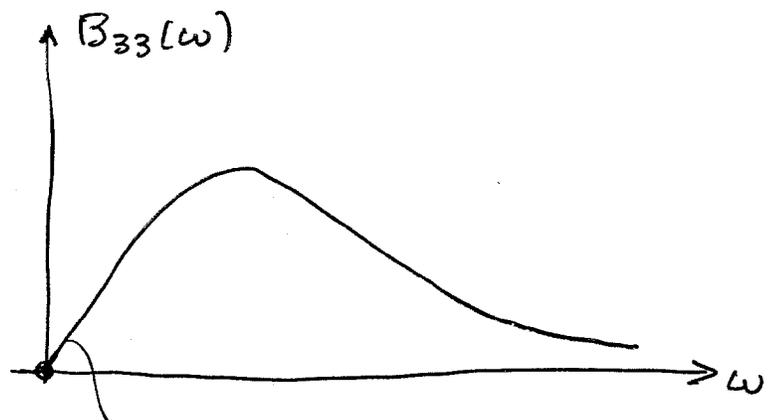
THIS RADIATION CONDITION IS ESSENTIAL FOR THE FORMULATION AND SOLUTION OF THE BOUNDARY VALUE PROBLEMS FOR  $\varphi_j$  USING PANEL METHODS WHICH ARE THE STANDARD SOLUTION TECHNIQUE AT ZERO AND FORWARD SPEED. —

● QUALITATIVE BEHAVIOUR OF THE FORCES, COEFFICIENTS AND MOTIONS OF FLOATING BODIES



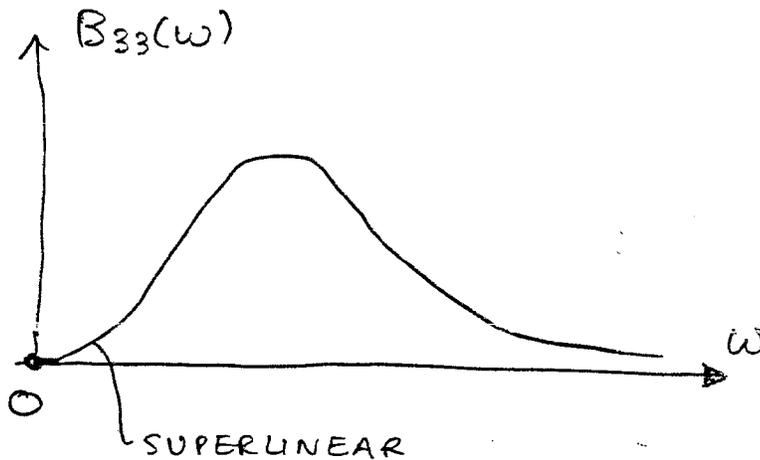
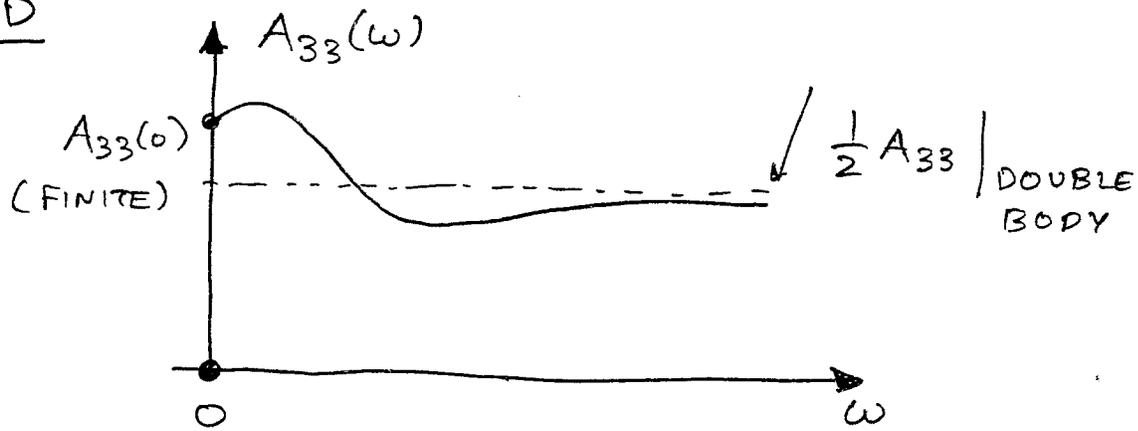
$$-\omega^2 \phi + g \phi_z = 0$$

$\nearrow \phi_z = 0, \omega = 0$   
 $\searrow \phi = 0, \omega \rightarrow \infty$



$B_{33}(\omega) \sim \omega$ , AT LOW  $\omega$

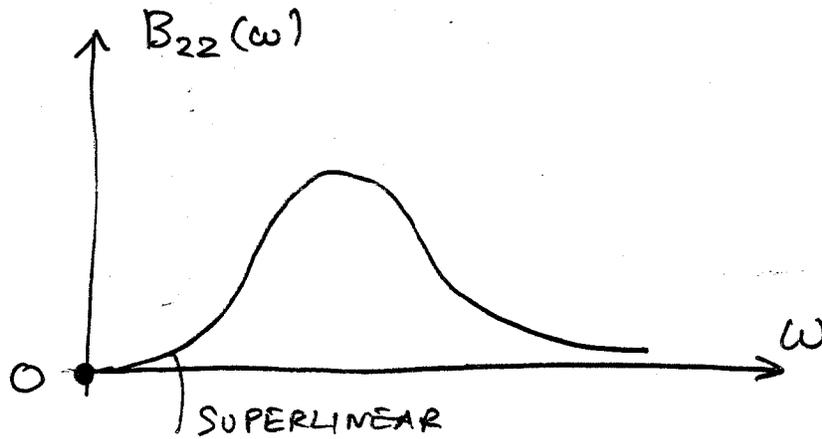
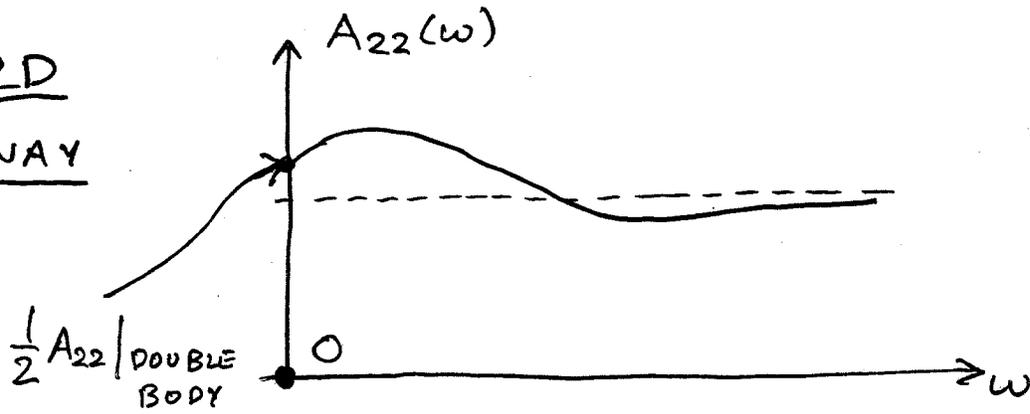
3D



COMMENTS:

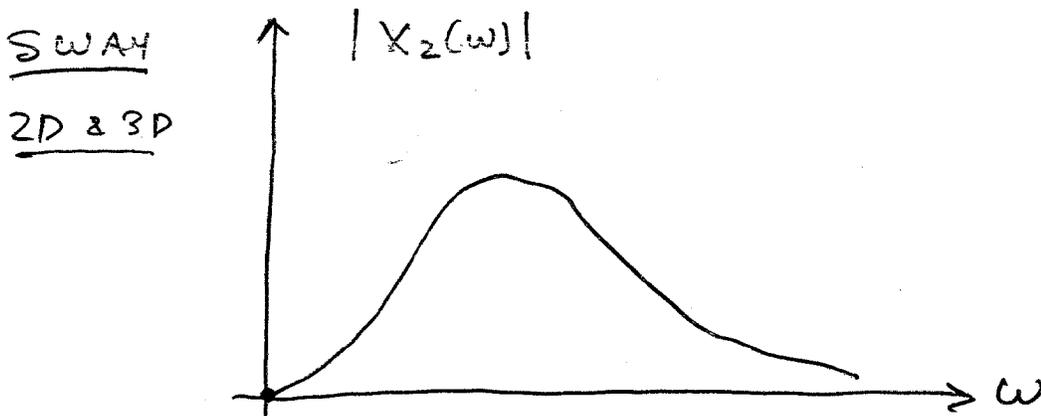
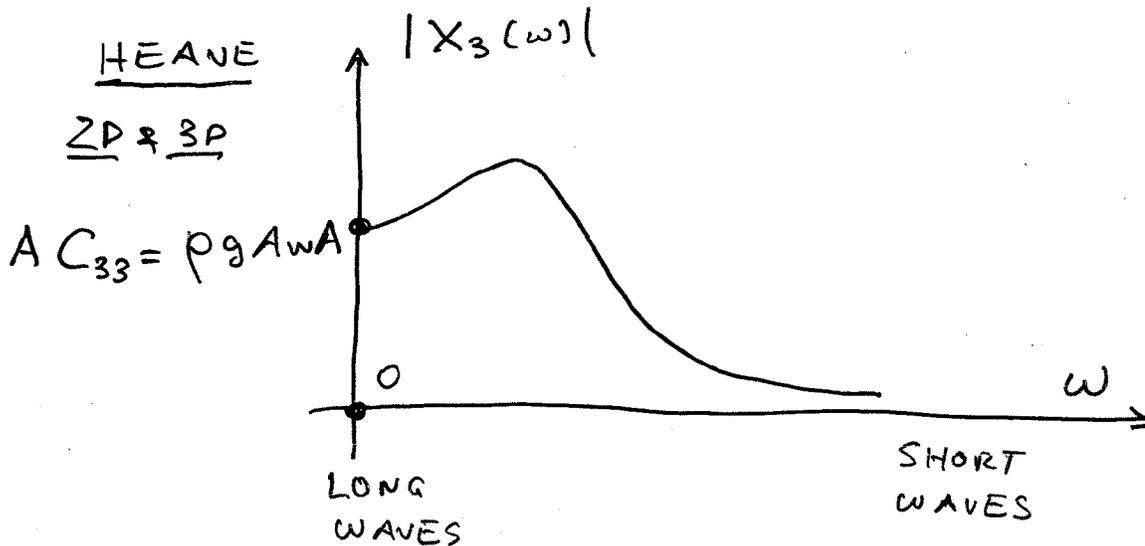
- THE 2D HEAVE ADDED MASS IS SINGULAR AT LOW FREQUENCIES. IT IS FINITE IN 3D
- THE 2D HEAVE DAMPING COEFFICIENT IS DECAYING TO ZERO LINEARLY IN 2D AND SUPERLINEARLY IN 3D. A TWO-DIMENSIONAL SECTION IS A BETTER WAVE MAKER THAN A THREE-DIMENSIONAL ONE

2D  
SWAY



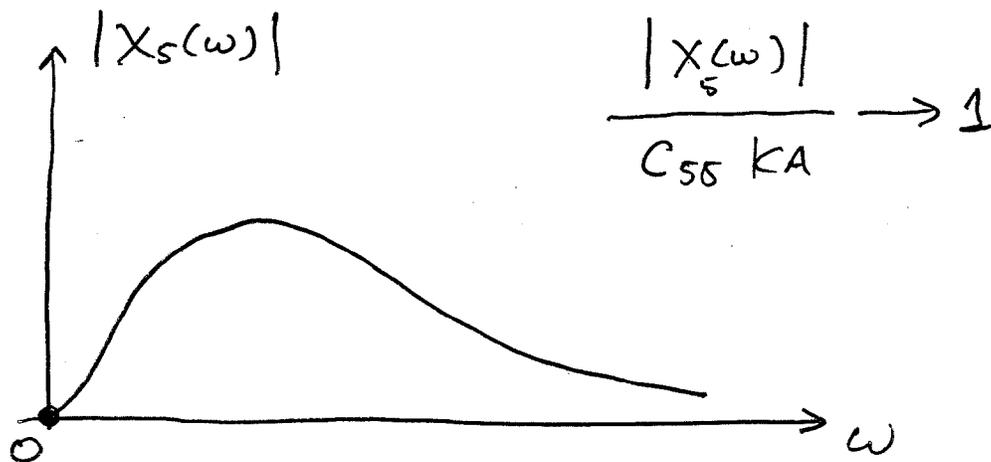
- A 2D SECTION OSCILLATING IN SWAY IS LESS EFFECTIVE A WAVE MAKER AT LOW FREQUENCIES THAN THE SAME SECTION OSCILLATING IN HEAVE
- THE ZERO-FREQUENCY LIMIT OF THE SWAY ADDED MASS IS FINITE AND SIMILAR TO THE INFINITE FREQUENCY LIMIT OF THE HEAVE ADDED MASS.

# EXCITING FORCES



- IN LONG WAVES THE HEAVE EXCITING FORCE TENDS TO THE HEAVE RESTORING COEFFICIENT TIMES THE AMBIENT WAVE AMPLITUDE  
THE FREE SURFACE BEHAVES LIKE A FLAT SURFACE MOVING UP & DOWN
- IN LONG WAVES THE SWAY EXCITING FORCE TENDS TO ZERO. PROOF WILL FOLLOW
- IN SHORT WAVES ALL FORCES TEND TO ZERO.

## PITCH OR ROLL EXCITING MOMENT



- ③ PITCH EXCITING MOMENT (SAME APPLIES TO ROLL) TENDS TO ZERO. LONG WAVES HAVE A SMALL SLOPE WHICH IS PROPORTIONAL TO  $KA$ , WHERE  $K$  IS THE WAVENUMBER AND  $A$  IS THE WAVE AMPLITUDE
- ④ PROVE THAT TO LEADING ORDER FOR  $KA \rightarrow 0$  :

$$|X_s(\omega)| \sim KA C_{55}$$

WHERE  $C_{55}$  IS THE PITCH ( $C_{44}$  FOR ROLL) HYDROSTATIC RESTORING COEFFICIENT.

[NB: VERY LONG WAVES LOOK LIKE A FLAT SURFACE INCLINED @  $KA$ ].

# BODY MOTIONS IN REGULAR WAVES

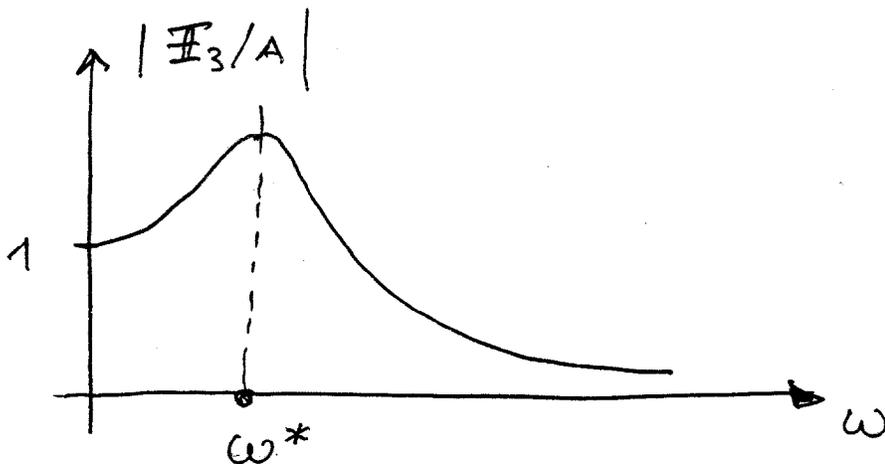
## HEAVE

$$\underline{\mathbb{F}}_3 = \frac{X_3(\omega)}{-\omega^2(A_{33} + M) + i\omega B_{33} + C_{33}}$$

RESONANCE:

$$\omega^2 = \frac{C_{33}}{M + A_{33}} = \frac{\rho g A_w}{M + A_{33}(\omega)}$$

- IN PRINCIPLE THE ABOVE EQUATION IS NONLINEAR FOR  $\omega$ . WILL BE APPROXIMATED BELOW

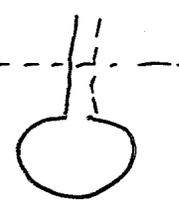


AT RESONANCE:  $\underline{\mathbb{F}}_3 = \frac{X_3(\omega^*)}{i\omega^* B_{33}(\omega^*)}$

INVOKING THE RELATION BETWEEN THE DAMPING COEFFICIENT AND THE EXCITING FORCE IN 3D:

$$\left| \frac{\Xi_3}{A} \right| = \frac{|\mathbb{X}_3(\omega)|}{\omega \frac{k}{4\rho g V_g} |\mathbb{X}_3|^2}, \quad V_g = \frac{g}{2\omega}$$

$$= \frac{2\rho g}{\omega^3 |\mathbb{X}_3(\omega)|}, \quad \text{AT RESONANCE}$$

- THIS COUNTER-INTUITIVE RESULT SHOWS THAT FOR A BODY UNDERGOING A PURE HEAVE OSCILLATION, THE MODULUS OF THE HEAVE RESPONSE AT RESONANCE IS INVERSELY PROPORTIONAL TO THE MODULUS OF THE HEAVE EXCITING FORCE
- SWATH VESSELS OF THE FORM  MAY BENEFIT FROM THIS PROPERTY
- VISCOUS EFFECTS NOT DISCUSSED HERE MAY AFFECT HEAVE RESPONSE AT RESONANCE
- STUDY AS AN EXERCISE THE BEHAVIOUR OF THE SWAY RESPONSE.

OFTEN IT IS USEFUL TO ESTIMATE THE HEAVE AND PITCH RESONANT FREQUENCIES IN TERMS OF THE PRINCIPAL DIMENSIONS OF A BODY.

2D : THE HEAVE ADDED MASS  $A_{33}$  IS OFTEN NOT TOO FAR FROM ITS INFINITE FREQUENCY LIMIT, OR:

$$A_{33} \approx A_{33}(\infty) = \frac{1}{2} \rho V$$

FOR A BOX-LIKE SECTION WITH BEAM  $B$  AND DRAFT  $T$ :

$$A_{33} \approx \frac{1}{2} \rho BT = \frac{1}{2} M$$

$$\omega^2 = \frac{\rho g B}{\rho BT + \frac{1}{2} \rho BT} = \frac{3}{2} \frac{g}{T}$$

$$\text{OR } \omega^* \approx \left( \frac{3}{2} \frac{g}{T} \right)^{1/2}$$

③ DERIVE THE CORRESPONDING RESULT FOR ROLL

3D : THE EXTENSION TO A 3D BODY OF GENERAL SHAPE IS EASY IN PRINCIPLE

- FOR A SHIP LIKE BODY WITH LENGTH LARGE RELATIVE TO ITS BEAM,  $B$ , WE MAY APPROXIMATE ITS SHAPE AS A BARGE WITH LENGTH  $L$ , BEAM  $B$  AND DRAFT  $T$ . IN THIS CASE THE 2D ANALYSIS IS VERY ACCURATE
- PROVE THAT THE SAME RESONANT FREQUENCY  $\omega^*$  IS APPLICABLE TO THE PITCH MOTION OF THE BARGE AS WELL. SO HEAVE & PITCH RESONATE AT THE SAME FREQUENCY. THIS RESULT IS VERY WELL VERIFIED BY 3D COMPUTATIONS
- SIMILAR ARGUMENTS APPLY TO THE ROLL MOTION OF A SHIP APPROXIMATED AS A BOX-LIKE BODY. THE RESONANT FREQUENCY FROM A 2D ANALYSIS IS VERY CLOSE TO THE ROLL RESONANT FREQUENCY FOR A SLENDER SHIP. —