

ANALYTICAL SOLUTIONS OF THE WAVE-BODY PROBLEM FORMULATED ABOVE ARE RARE. THE FEW EXCEPTIONS WHICH FIND FREQUENT USE IN PRACTICE ARE :

- ① WAVEMAKER THEORY (STUDIED)
- ② DIFFRACTION BY A VERTICAL CIRCULAR CYLINDER (STUDIED BELOW)
- ③ LONG-WAVELENGTH APPROXIMATIONS (STUDIED NEXT).

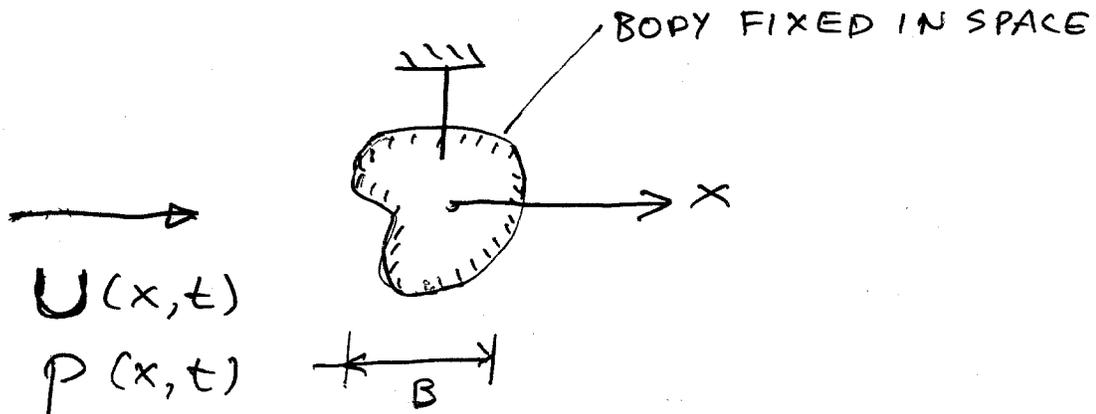
LONG-WAVELENGTH APPROXIMATIONS

VERY FREQUENTLY THE LENGTH OF AMBIENT WAVES λ IS LARGE COMPARED TO THE DIMENSION OF FLOATING BODIES.

FOR EXAMPLE THE LENGTH OF A WAVE WITH PERIOD $T=10$ sec IS $\lambda \approx T^2 + \frac{T^2}{2} \approx 150$ m

THE BEAM OF A SHIP WITH LENGTH $L=100$ m CAN BE 20 m AS IS THE CASE FOR THE DIAMETER OF THE LEG OF AN OFFSHORE PLATFORM.

GI TAYLOR'S FORMULA



$U(x,t)$: VELOCITY OF AMBIENT UNIDIRECTIONAL FLOW

$P(x,t)$: PRESSURE CORRESPONDING TO $U(x,t)$

- $\lambda \sim \frac{|U|}{|\nabla U|} \gg B = \text{BODY CHARACTERISTIC DIMENSION}$

- IN THE ABSENCE OF VISCOUS EFFECTS AND TO LEADING ORDER FOR $\lambda \gg B$:

$$F_x = - \left(\nabla + \frac{A_{11}}{\rho} \right) \frac{\partial P}{\partial x} \Big|_{x=0}$$

- F_x : FORCE IN X-DIRECTION
 - ∇ : BODY DISPLACEMENT
 - A_{11} : SURGE ADDED MASS
- } PROOF IN MH.

AN ALTERNATIVE FORM OF GI TAYLOR'S FORMULA FOR A FIXED BODY FOLLOWS FROM EULER'S EQUATIONS:

$$\frac{\partial U}{\partial t} + U \frac{\partial U}{\partial x} \approx -\frac{1}{\rho} \frac{\partial p}{\partial x}$$

THUS:

$$\bullet F_x = (\rho \psi + A_{11}) \left(\frac{\partial U}{\partial t} + U \frac{\partial U}{\partial x} \right)_{x=0}$$

IF THE BODY IS ALSO TRANSLATING IN THE X-DIRECTION WITH DISPLACEMENT $X_1(t)$

THEN THE TOTAL FORCE BECOMES

$$\bullet F_x = (\rho \psi + A_{11}) \left(\frac{\partial U}{\partial t} + U \frac{\partial U}{\partial x} \right) - A_{11} \frac{d^2 X_1(t)}{dt^2}$$

OFTEN, WHEN THE AMBIENT VELOCITY U IS ARISING FROM PLANE PROGRESSIVE WAVES,

$$\left| U \frac{\partial U}{\partial x} \right| = O(A^2) \text{ AND IS OMITTED. NOTE}$$

THAT U DOES NOT INCLUDE DISTURBANCE EFFECTS DUE TO THE BODY. —

- APPLICATIONS OF G I TAYLOR'S FORMULA IN WAVE-BODY INTERACTIONS

A) ARCHIMEDEAN HYDROSTATICS

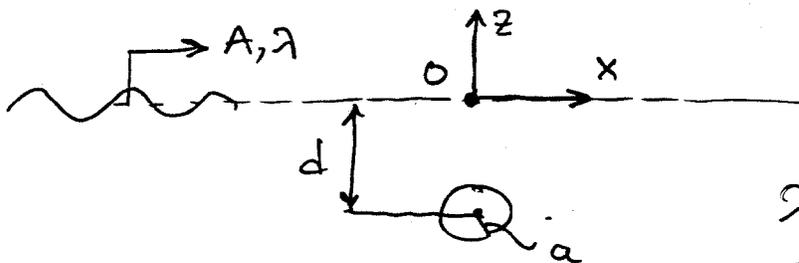
$$p = -\rho g z, \quad \frac{\partial p}{\partial z} = -\rho g$$

$$F_z = -(\mathcal{V} + \phi) \frac{\partial p}{\partial z} = \rho g \mathcal{V}$$

↑
NO ADDED MASS
SINCE THERE IS
NO FLOW

- SO ARCHIMEDES' FORMULA IS A SPECIAL CASE OF G I TAYLOR WHEN THERE IS NO FLOW. THIS OFFERS AN INTUITIVE MEANING TO THE TERM THAT INCLUDES THE BODY DISPLACEMENT.

B) REGULAR WAVES OVER A CIRCLE FIXED UNDER THE FREE SURFACE



$$\lambda \gg a$$

$$\lambda \sim d$$

$$\phi_I = \text{Re} \left\{ \frac{igA}{\omega} e^{kz - ikx + i\omega t} \right\}, \quad k = \omega^2/g$$

$$u = \frac{\partial \phi_I}{\partial x} = \text{Re} \left\{ \frac{igA}{\omega} (-ik) e^{kz - ikx + i\omega t} \right\}$$

$$= \text{Re} \left\{ \omega A e^{-kd + i\omega t} \right\}_{\substack{x=0 \\ z=-d}}$$

$$v = \frac{\partial \phi_I}{\partial z} = \text{Re} \left\{ \frac{igA}{\omega} k e^{kz - ikx + i\omega t} \right\}$$

$$= \text{Re} \left\{ i\omega A e^{-kd + i\omega t} \right\}_{\substack{x=0 \\ z=-d}}$$

SO THE HORIZONTAL FORCE ON THE CIRCLE IS:

$$F_x = \left(\forall + \frac{a_{11}}{\rho} \right) \frac{\partial u}{\partial t} + O(A^2)$$

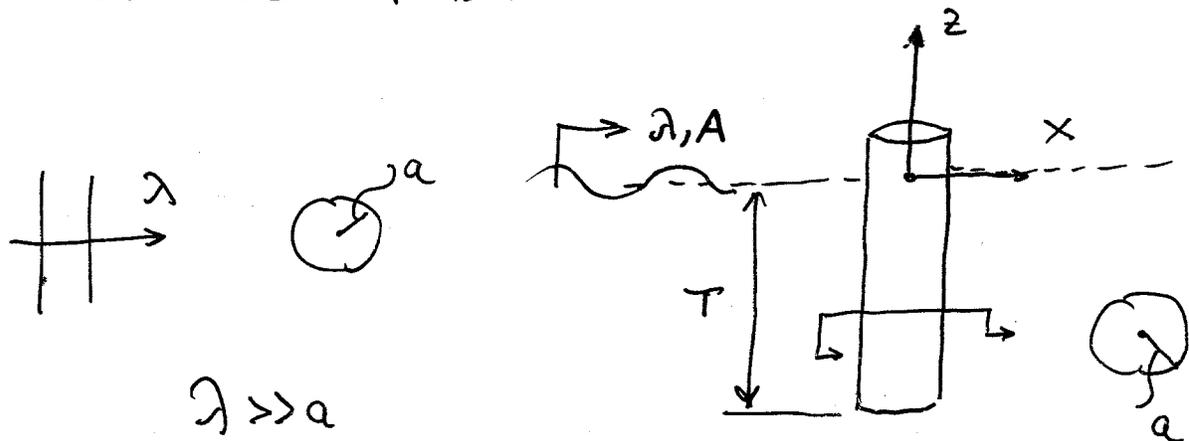
$$\forall = \pi a^2, \quad a_{11} = \pi \rho a^2$$

$$\frac{\partial u}{\partial t} = \text{Re} \left\{ i\omega^2 e^{-kd + i\omega t} \right\}$$

THUS:
$$F_x = -2\pi a^2 \omega^2 A e^{-kd} \sin \omega t$$

- DERIVE THE VERTICAL FORCE ALONG VERY SIMILAR LINES. IT IS SIMPLY 90° OUT OF PHASE RELATIVE TO F_x WITH THE SAME MODULUS.

C) HORIZONTAL FORCE ON A FIXED CIRCULAR CYLINDER OF DRAFT T:



THIS CASE ARISES FREQUENTLY IN WAVE INTERACTIONS WITH FLOATING OFFSHORE PLATFORMS.

HERE WE WILL EVALUATE $\frac{\partial u}{\partial t}$ ON THE AXIS OF THE PLATFORM AND USE A STRIPWISE INTEGRATION TO EVALUATE THE TOTAL HYDRODYNAMIC FORCE.

$$u = \frac{\partial \phi_I}{\partial x} = \text{Re} \left\{ \frac{iga}{\omega} (-ik) e^{kz - ikx + i\omega t} \right\}$$

$$= \text{Re} \left\{ \omega A e^{kz + i\omega t} \right\}_{x=0}$$

$$\frac{\partial u}{\partial t}(z) = \text{Re} \left\{ \omega A (i\omega) e^{kz + i\omega t} \right\}$$

$$= -\omega^2 A e^{kz} \sin \omega t.$$

THE DIFFERENTIAL HORIZONTAL FORCE OVER A STRIP dz AT A DEPTH z BECOMES:

$$\begin{aligned} dF_z &= \rho (\nabla + a_{11}) \frac{\partial u}{\partial t} dz \\ &= \rho (\pi a^2 + \pi a^2) \frac{\partial u}{\partial t} dz \\ &= 2\pi \rho a^2 (-\omega^2 A e^{kz}) \sin \omega t dz \end{aligned}$$

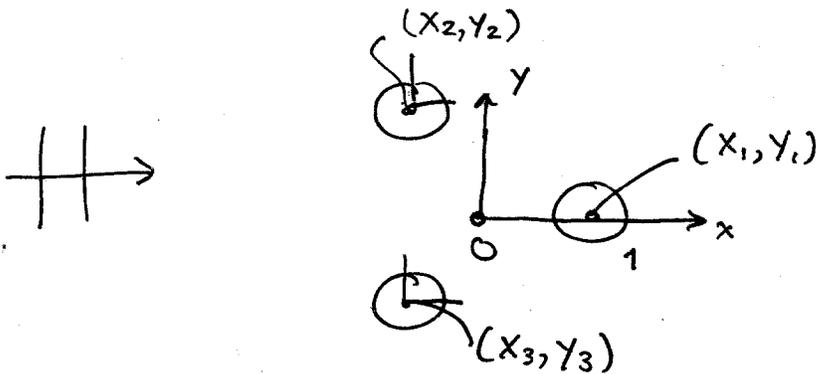
THE TOTAL HORIZONTAL FORCE OVER A TRUNCATED CYLINDER OF DRAFT T BECOMES:

$$F_x = \int_{-T}^0 dz dF = -2\pi \rho a^2 \omega^2 A \sin \omega t \times \int_{-T}^0 e^{kz} dz$$

$$X_1 \equiv F_x = -2\pi \rho a^2 \omega^2 A \sin \omega t \cdot \frac{1 - e^{-kT}}{k}$$

- THIS IS A VERY USEFUL AND PRACTICAL RESULT. IT PROVIDES AN ESTIMATE OF THE SURGE EXCITING FORCE ON ONE LEG OF A POSSIBLY MULTI-LEG PLATFORM
- AS $T \rightarrow \infty$; $\frac{1 - e^{-kT}}{k} \rightarrow \frac{1}{k}$

D) HORIZONTAL FORCE ON MULTIPLE VERTICAL CYLINDERS IN ANY ARRANGEMENT :



THE PROOF IS ESSENTIALLY BASED ON A PHASING ARGUMENT, RELATIVE TO THE REFERENCE FRAME :

$$\phi_I = \text{Re} \left\{ \frac{i g A}{\omega} e^{kz - ikx + i\omega t} \right\}$$

- EXPRESS THE INCIDENT WAVE RELATIVE TO THE LOCAL FRAMES BY INTRODUCING THE PHASE FACTORS :

$$P_i = e^{-ikx_i}$$

LET : $x = x_i + \xi_i$

THEN RELATIVE TO THE i -TH LEG :

$$\phi_I^{(i)} = \text{Re} \left\{ \frac{i g A}{\omega} e^{kz - ik\xi_i + i\omega t} P_i \right\}$$

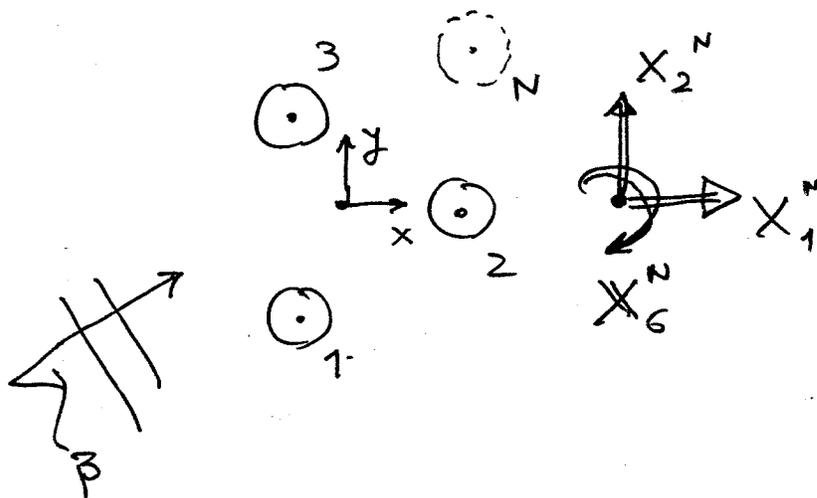
$$i = 1, \dots, N$$

IGNORING INTERACTIONS BETWEEN LEGS,
WHICH IS A GOOD APPROXIMATION IN LONG
WAVES, THE TOTAL EXCITING FORCE ON
AN N-CYLINDER PLATFORM IS:

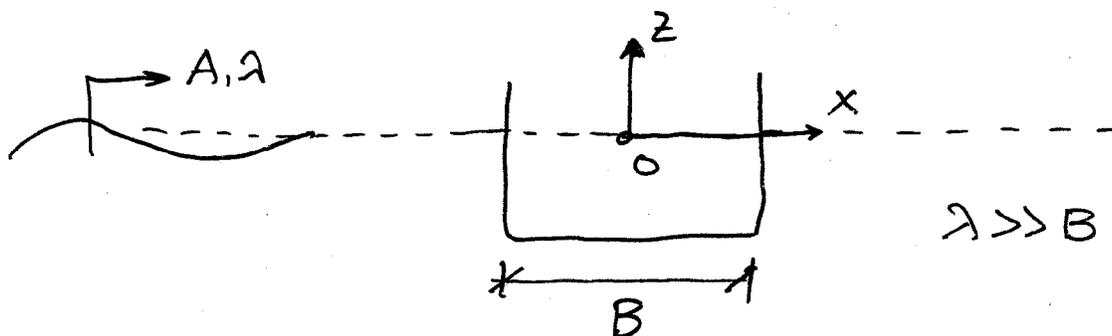
$$X_1^N = \sum_{i=1}^N P_i X_1$$

THE ABOVE EXPRESSION GIVES THE COMPLEX
AMPLITUDE OF THE FORCE WITH X_1 , GIVEN
IN THE SINGLE CYLINDER CASE. —

- THE ABOVE TECHNIQUE MAY BE EASILY
EXTENDED TO ESTIMATE THE SWAY FORCE
AND YAW MOMENT ON N-CYLINDERS WITH
LITTLE EXTRA EFFORT (LEFT AS AN EXERCISE)



E) SURGE EXCITING FORCE ON A 2D SECTION



$$\phi_I = \text{Re} \left\{ \frac{igA}{\omega} e^{kz - ikx + i\omega t} \right\}$$

$$u = \text{Re} \left\{ \frac{igA}{\omega} (-ik) e^{kz - ikx + i\omega t} \right\}$$

$$\frac{\partial u}{\partial t} = \text{Re} \left\{ \frac{igA}{\omega} \left(-i\frac{\omega^2}{g}\right) (i\omega) e^{i\omega t} \right\}_{\substack{x=0 \\ z=0}}$$

$$= \text{Re} \left\{ i\omega^2 A e^{i\omega t} \right\} = -\omega^2 A \sin \omega t$$

$$X_1 = \left(\underset{\substack{\uparrow \\ \text{FROUDE} \\ \text{KRYLOV}}}{\rho \psi + A_{II}} \right) \frac{\partial u}{\partial t} = -\omega^2 A \sin \omega t \left(\underset{\substack{\uparrow \\ \text{DIFFRACTION}}}{\rho \psi + A_{II}} \right)$$

- IF THE BODY SECTION IS A CIRCLE WITH RADIUS a :

$$\rho \psi = A_{II} = \pi \rho a^2 / 2$$

SO IN LONG WAVES THE SURGE EXCITING FORCE IS EQUALLY DIVIDED BETWEEN THE FROUDE-KRYLOV AND THE DIFFRACTION COMPONENTS. THIS IS NOT THE CASE FOR HEAVE!

F) HEAVE EXCITING FORCE ON A SURFACE PIERCING SECTION

IN LONG WAVES, THE LEADING ORDER
EFFECT IN THE EXCITING FORCE IS THE
HYDROSTATIC CONTRIBUTION:

$$X_3 \sim \rho g A_w A$$

WHERE A_w IS THE BODY WATER PLANE
AREA IN 2D OR 3D. A IS THE WAVE
AMPLITUDE. THIS CAN BE SHOWN TO
BE THE LEADING ORDER CONTRIBUTION
FROM THE FROUDE-KRYLOV FORCE

$$X_3^{FK} = \rho g A \iint_{S_B} e^{kz - ikx} \eta_3 ds$$

USING THE TAYLOR SERIES EXPANSION:

$$e^{kz - ikx} = 1 + (kz - ikx) + O(kB)^2$$

IT IS EASY TO VERIFY THAT: $X_3 \rightarrow \rho g A A_w$.
THE SCATTERING CONTRIBUTION IS OF ORDER
KB. FOR SUBMERGED BODIES: $X_3^{FK} = O(kB)$.