

## MATERIALS IN NATURE

### MATERIAL PROPERTIES

Hooke's Law:

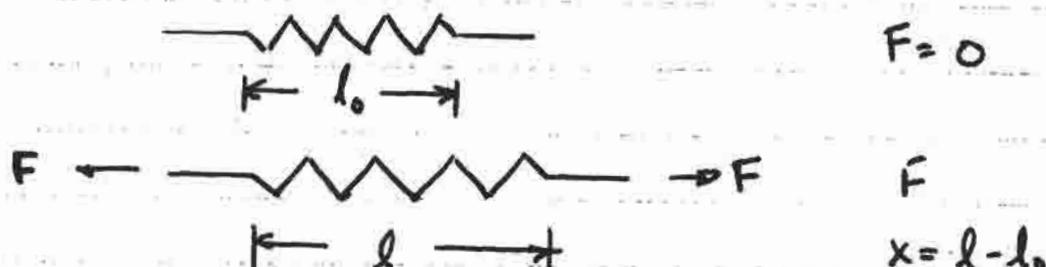


Diagram showing a spring with length  $l_0$ . A force  $F$  is applied to the left, and an equal force  $F$  is applied to the right, resulting in a total extension  $x = l - l_0$ .

$$F = kx \quad k = \text{SPRING CONSTANT}$$

FOR MANY MATERIALS, Hooke's LAW APPLIES

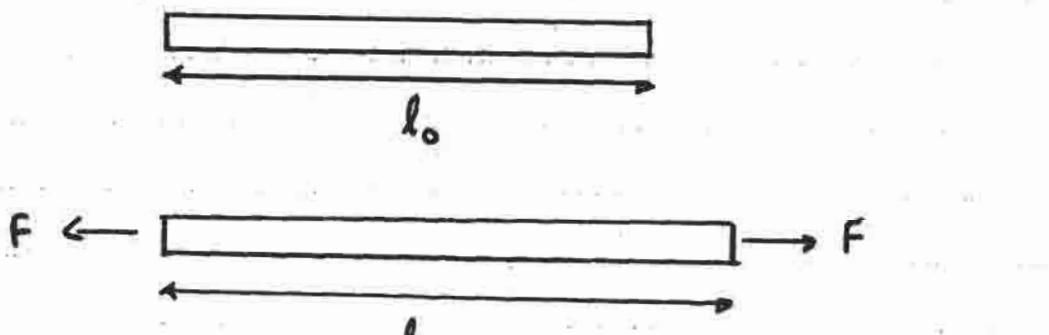


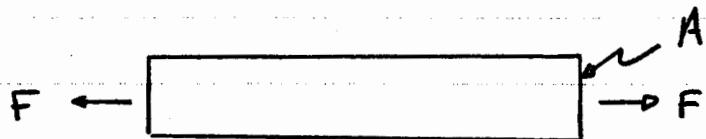
Diagram showing a rectangular bar with original length  $l_0$ . A force  $F$  is applied to the left, and an equal force  $F$  is applied to the right, resulting in a total extension  $\delta = l - l_0$ .

$$F = k'\delta \quad \delta = l - l_0$$

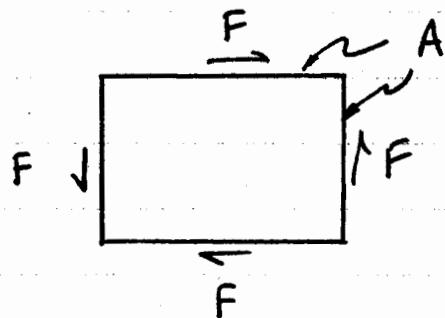
## DEFINE STRESS, STRAIN

STRESS = FORCE / AREA

NORMAL STRESS = FORCE / (AREA NORMAL TO FORCE) =  $\sigma$



SHEAR STRESS = FORCE / (AREA PARALLEL TO FORCE) =  $\tau$



UNITS: pounds / square inch = psi

$N/m^2$  = Pascals = Pa

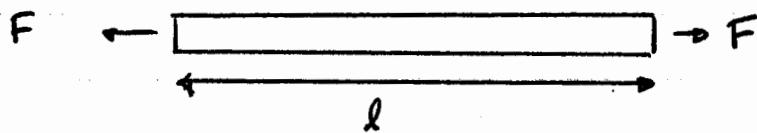
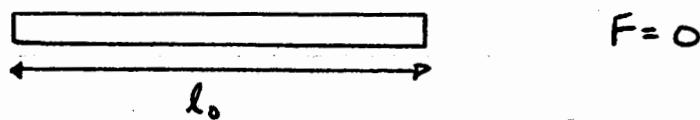
$MN/m^2$  =  $10^6$  Pa

$GN/m^2$  =  $10^9$  Pa.

## DEFINE STRESS, STRAIN

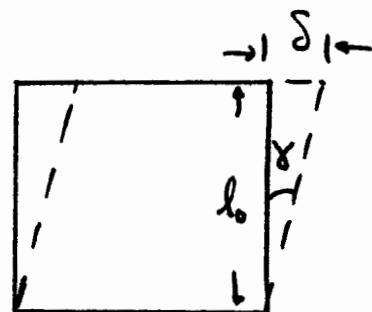
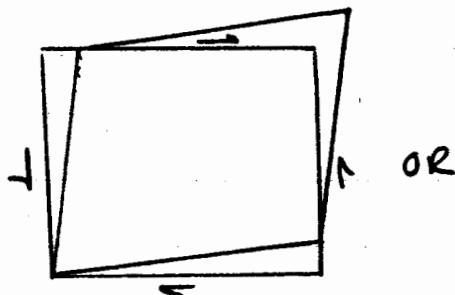
STRAIN = DEFORMATION PER UNIT ORIGINAL LENGTH.  
=  $\delta / l_0$ .

NORMAL STRAIN,  $\epsilon$



$$\epsilon = \frac{l - l_0}{l_0} = \frac{\delta}{l_0}$$

SHEAR STRAIN,  $\gamma$



$$\tan \gamma = \frac{\delta}{l_0}$$

$$\gamma = \frac{\delta}{l_0} \quad (\text{SMALL } \gamma, \tan \gamma = \gamma)$$

UNITS (-)

## YOUNG'S MODULUS

HOOKE'S LAW FOR LINEAR ELASTIC SOLIDS:

$$F = k \times$$

$$\frac{F}{A} = k' \frac{\delta}{l_0}$$

$$\sigma = E \epsilon \quad E = \text{YOUNG'S MODULUS}$$

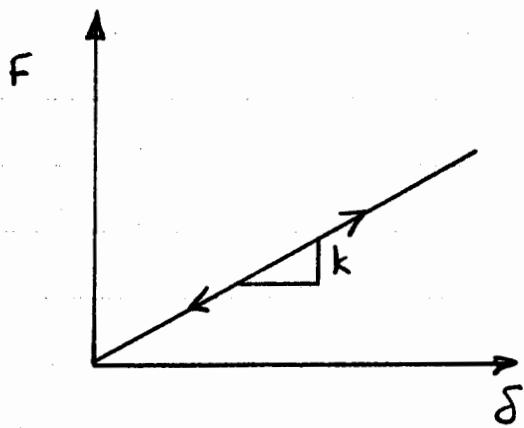
$$\text{OR} \quad \tau = G \gamma \quad G = \text{SHEAR MODULUS}$$

E, G UNITS:  $10^6$  psi, GPa

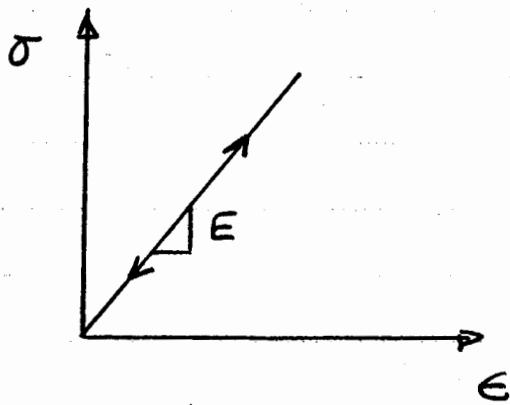
eq.	Steel	$30 \times 10^6$ psi	210 GPa
	Aluminum	$10 \times 10^6$ psi	70 GPa
	Glass	$10 \times 10^6$ psi	70 GPa
	Wood	$1 \times 10^6$ psi	7 GPa.

YOUNG'S MODULUS IS A MEASURE OF A MATERIAL'S  
STIFFNESS.

## YOUNG'S MODULUS



$$F = k \delta$$



$$\sigma = E \epsilon$$

LINEAR : FORCE + DISPLACEMENT LINEARLY RELATED.

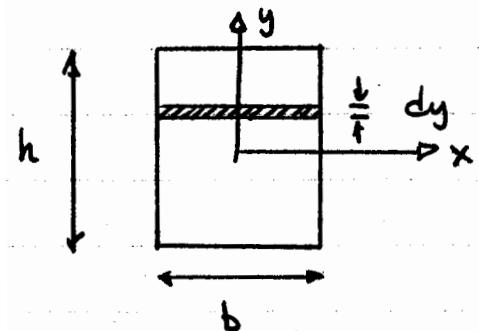
ELASTIC : DISPLACEMENT IS COMPLETELY RECOVERABLE,  
INSTANTANEOUSLY

## SECTION GEOMETRY.

(i) AREA, A

(ii) MOMENT OF INERTIA (SECOND MOMENT OF AREA), I

$$I_{xx} = \int_A y^2 dA$$



$$dA = b dy$$

$$I_{xx} = \int_{-h/2}^{h/2} y^2 b dy$$

$$= \frac{by^3}{3} \Big|_{-h/2}^{h/2}$$

$$= \frac{b}{3} \left[ \left(\frac{h}{2}\right)^3 - \left(-\frac{h}{2}\right)^3 \right]$$

$$= \frac{b}{3} \left( \frac{h^3}{8} + \frac{h^3}{8} \right)$$

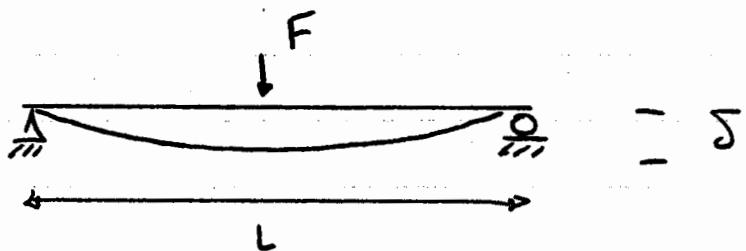
$$I_{xx} = \frac{bh^3}{12}$$

$\Rightarrow$  BEAM BENDING

$\Rightarrow$  COLUMN BUCKLING.

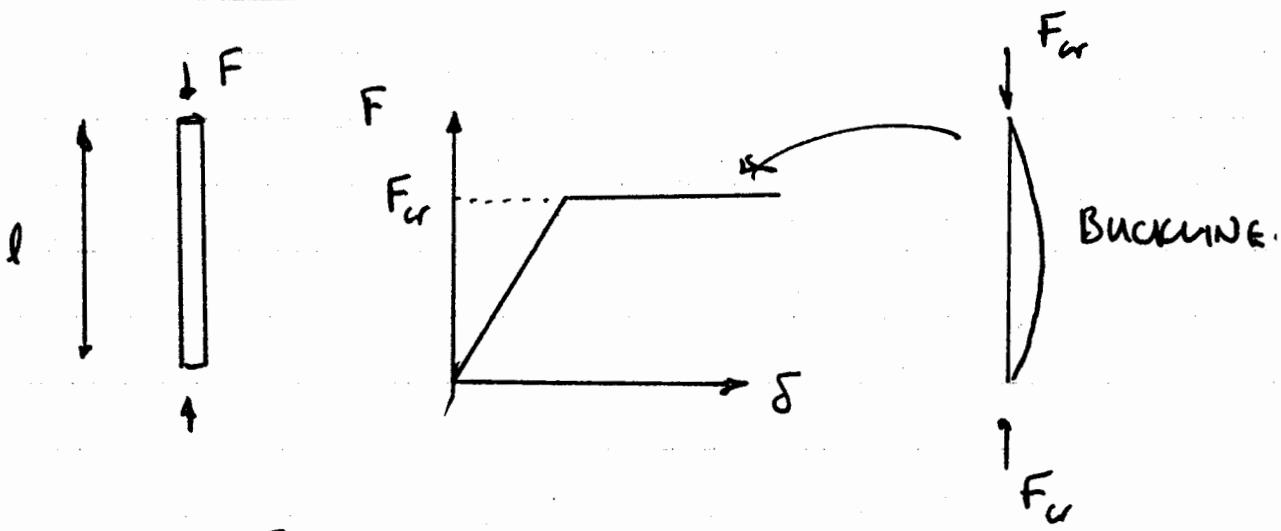
## MOMENT OF INERTIA IMPRINT IN

### BEAM BENDING :



$$\delta = \frac{FL^3}{48EI}$$

### COLUMN BUCKLING :



$$F_{cr} = \frac{\pi^2 EI}{l^2}$$