

## 13. NATURAL FREQUENCY AND DAMPING RATIO

There is a standard, and useful, normalization of the second order homogeneous linear constant coefficient ODE

$$m\ddot{x} + b\dot{x} + kx = 0$$

under the assumption that both the “mass”  $m$  and the “spring constant”  $k$  are positive. It is illustrated in the Mathlet **Damping Ratio**.

In the absence of a damping term, the ratio  $k/m$  would be the square of the circular frequency of a solution, so we will write  $k/m = \omega_n^2$  with  $\omega_n > 0$ , and call  $\omega_n$  the **natural circular frequency** of the system.

Divide the equation through by  $m$ :  $\ddot{x} + (b/m)\dot{x} + \omega_n^2 x = 0$ . Critical damping occurs when the coefficient of  $\dot{x}$  is  $2\omega_n$ . The **damping ratio**  $\zeta$  is the ratio of  $b/m$  to the critical damping constant:  $\zeta = (b/m)/(2\omega_n)$ . The ODE then has the form

$$(1) \quad \boxed{\ddot{x} + 2\zeta\omega_n\dot{x} + \omega_n^2 x = 0}$$

Note that if  $x$  has dimensions of cm and  $t$  of sec, then  $\omega_n$  had dimensions  $\text{sec}^{-1}$ , and the damping ratio  $\zeta$  is “dimensionless,” a number which is the same no matter what units of distance or time are chosen. Critical damping occurs precisely when  $\zeta = 1$ : then the characteristic polynomial has a repeated root:  $p(s) = (s + \omega_n)^2$ .

In general the characteristic polynomial is  $s^2 + 2\zeta\omega_n s + \omega_n^2$ , and it has as roots

$$-\zeta\omega_n \pm \sqrt{\zeta^2\omega_n^2 - \omega_n^2} = \omega_n(-\zeta \pm \sqrt{\zeta^2 - 1}).$$

These are real when  $|\zeta| \geq 1$ , equal when  $\zeta = \pm 1$ , and nonreal when  $|\zeta| < 1$ . When  $|\zeta| \leq 1$ , the roots are

$$-\zeta\omega_n \pm i\omega_d$$

where

$$(2) \quad \omega_d = \sqrt{1 - \zeta^2}\omega_n$$

is the **damped circular frequency** of the system. These are complex numbers of magnitude  $\omega_n$  and argument  $\pm\theta$ , where  $-\zeta = \cos\theta$ . Note that the presence of a damping term decreases the frequency of a solution to the undamped equation—the natural frequency  $\omega_n$ —by the factor  $\sqrt{1 - \zeta^2}$ . The general solution is

$$(3) \quad x = Ae^{-\zeta\omega_n t} \cos(\omega_d t - \phi)$$

Suppose we have such a system, but don't know the values of  $\omega_n$  or  $\zeta$ . At least when the system is underdamped, we can discover them by a couple of simple measurements of the system response. Let's displace the mass and watch it vibrate freely. If the mass oscillates, we are in the underdamped case. We can find  $\omega_d$  by measuring the times at which  $x$  achieves its maxima. These occur when the derivative vanishes, and

$$\dot{x} = Ae^{-\zeta\omega_n t} (-\zeta\omega_n \cos(\omega_d t - \phi) - \omega_d \sin(\omega_d t - \phi)).$$

The factor in parentheses is sinusoidal with circular frequency  $\omega_d$ , so successive zeros are separated from each other by a time lapse of  $\pi/\omega_d$ . If  $t_1$  and  $t_2$  are the times of neighboring maxima of  $x$  (which occur at every other extremum) then  $t_2 - t_1 = 2\pi/\omega_d$ , so we have discovered the damped natural frequency:

$$(4) \quad \omega_d = \frac{2\pi}{t_2 - t_1}.$$

We can also measure the ratio of the value of  $x$  at two successive maxima. Write  $x_1 = x(t_1)$  and  $x_2 = x(t_2)$ . The difference of their natural logarithms is the **logarithmic decrement**:

$$\Delta = \ln x_1 - \ln x_2 = \ln \left( \frac{x_1}{x_2} \right).$$

Then

$$x_2 = e^{-\Delta} x_1.$$

The logarithmic decrement turns out to depend only on the damping ratio, and to determine the damping ratio. To see this, note that the values of  $\cos(\omega_d t - \phi)$  at two points of time differing by  $2\pi/\omega_d$  are equal. Using (3) we find

$$\frac{x_1}{x_2} = \frac{e^{-\zeta\omega_n t_1}}{e^{-\zeta\omega_n t_2}} = e^{\zeta\omega_n(t_2 - t_1)}.$$

Thus, using (4) and (2),

$$\Delta = \ln \left( \frac{x_1}{x_2} \right) = \zeta\omega_n(t_2 - t_1) = \zeta\omega_n \frac{2\pi}{\omega_d} = \frac{2\pi\zeta}{\sqrt{1 - \zeta^2}}.$$

From the quantities  $\omega_d$  and  $\Delta$ , which are directly measurable characteristics of the unforced system response, we can calculate the system parameters  $\omega_n$  and  $\zeta$ :

$$(5) \quad \zeta = \frac{\Delta/2\pi}{\sqrt{1 + (\Delta/2\pi)^2}}, \quad \omega_n = \frac{\omega_d}{\sqrt{1 - \zeta^2}} = \sqrt{1 + \left( \frac{\Delta}{2\pi} \right)^2} \omega_d.$$

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